## Study of Interaction of Entrained Coal Dust Particles in Lean Methane – Air Premixed Flames

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### **Chapter 1**

### Introduction

In industries that manufacture, transport, process and/or use combustible dusts, accidental dust deflagrations represent a real hazard to both personnel and equipment. Among all the dust deflagrations, coal mine explosion is a worldwide phenomenon that causes a high number of casualties each year. An example is the recent coal mine explosion in West Virginia (April 5, 2010) killing 29 miners and considered as one of the most disastrous mining accidents in US history. Interestingly, most coal mine explosions often involve both a methane deflagration combined with fugitive coal dust (coal particles that are small enough to be easily suspended in air  $\sim 100~\mu m$ ) that is collected by the combustion wave as it progresses through the mine. The evolution of coal mine explosion usually has three different stages: 1. a small methane-air explosion (fire damp) is generated due to various ignition sources, such as sparks caused by mining equipment; 2. the pressure blast disperses the dust deposited on various surfaces of mine tunnels; 3 the methane-air flame interacts with the coal dust suspended in the air.

The objective of this work is to use a lab-scale experiment to study the interaction of coal dust particles and premixed methane-air flame at lean conditions. Specifically, the influence of coal dust particle size, number density on the laminar burning velocity of the flame is analyzed. It is observed that (in the laminar regime) a particle size less than 25  $\mu m$  is required to cause an increase in the burning velocity for the particular type of coal dust examined in this study (Pittsburgh seam coal). The reason for the promotion of the burning velocity is mainly due to the volatiles released by the coal dust particles due to heat transferred from the flame which causes a local increase in the equivalence ratio. The small particle size (<25  $\mu m$ ) ensures that the additional heat evolved due to the release of volatiles is greater than that absorbed by the particles. A mathematical model is developed to analyze this behavior and good agreement is observed with theory and experiment. Ultimately, given reliable property data for a particular type of dust, the model can be used as a predictive tool to analyze the hazard associated with any kind of dust.

A novel dust injector is developed as the first part of the thesis. The design of the dust injector is inspired by Bunsen burner where the venturi effect is used to entrain the dust and create a dust-gas mixture. The performance of the novel dust injector is investigated and presented in Chapter 2. The results show that there is an optimum area for the side openings for dust entrainment, at which the injector performance will be the best The entrainment rate is found to be a non-linear function of the flow-rate for low air flow-rates (till around 9.4 gram per minute) and for higher air flow-rates it becomes an linear function

The second part of the thesis involves analyzing the change in laminar burning velocity by using a dust burner based on the injector. This study is presented in Chapter 3 of this thesis. In this study, the changes of laminar burning velocity with particles are investigated experimentally and theoretically. The laminar burning velocity of the coal dust-methane-air mixture is determined by taking a shadowgraph of the resulting flame and using the cone-angle method. The results show that the addition of coal dust in methane-air premixed flame reduces the laminar burning velocity at particle sizes of 53 to 63  $\mu$ m and 75 to 90  $\mu$ m. However, burning velocity promotion is observed for 0 to 25  $\mu$ m particles at a methane-air equivalence ratio  $\phi = 0.80$ . A mathematical model is developed to analyze the influence of particle – gas - flame interaction. The model successfully predicts the change of laminar burning velocity at various dust concentrations.

Based on the observations made in the study, a Conclusions and Future Work section is included as Chapter 4. The Appendices contain additional information about the study and experiment data. The detailed information about experiment equipment and test procedures are available in Appendix A. The algorithm used for cone angle measurement in this study is provided in Appendix B. Appendix C contains additional experimental data, including entrainment data for Chapter 2, cone angle measurements, and images that are used to obtain laminar burning velocity using the cone-angle method in Chapter 3.

The following publications and conference papers have resulted from this work:

### **Peer Reviewed Publications**

- 1. Y. Xie, V. Raghavan, A. S. Rangwala, "Naturally Entraining Solid Particle Injector," *Powder Technology, Vol. 213, pp. 199-201, 2011 (Short Communication)*
- 2. Y. Xie, V. Raghavan and A. S. Rangwala, "Interaction of Coal Dust Particles on Lean Premixed Methane-air Flames," Combustion and Flame (Under Review)

### **Conference Papers**

- 1. Y. Xie, V. Raghavan, and A.S. Rangwala, "Study of Interaction of Entrained Coal Dust Particles in Lean Methane Air Premixed Flames," 7th U.S. National Combustion Meeting, Atlanta, Georgia, Mar 20-23, 2011.
- **2.** Y. Xie, V. Raghavan, and A.S. Rangwala, "Study of Interaction of Entrained Coal Dust Particles in Lean Methane Air Premixed Flames," Eastern States Combustion Meeting, University of Connecticut, Storrs, 9-12, Oct 2011.

### Chapter 2

### **Naturally Entraining Solid Particle Injector**

Y. Xie, V. Raghavan, A. S. Rangwala, "Naturally Entraining Solid Particle Injector," *Powder Technology, Vol. 213, pp. 199-201, 2011 (Short Communication)* 

### Abstract

The objective of the present work is to develop and calibrate a novel, vertical solid particle injector that uses the pressure drop in the air flow across an orifice plate in a circular pipe, to naturally entrain micron-sized solid particles such as coal dust. The particles continuously drop from a feeder located outside the pipe into the orifice plate through peripheral (side) openings in the pipe, where they are carried upwards by the air flow accelerated near the orifice exit. Three types of designs for the peripheral openings, in terms of the shape, size and number are evaluated by testing which one of them results in maximum particle entrainment. The device is calibrated by recording the mass loss rate of the powder as a function of volumetric flow-rate of air. The results show that there is an optimum area for the side openings, at which the injector performance will be the best for the given pipe and orifice-hole sizes. The entrainment rate is found to be a non-linear function of the flow-rate for low air flow-rates (till around 9.4 grams per minute) and for higher air flow-rates it becomes an almost linear function.

*Keywords:* Particle or powder injector; Orifice plate; Pressure drop; Natural entrainment; Mass loss rate

### 1. Introduction

Injection of solid particles of different sizes into a fluidized flow device or in a pneumatic conveying process has applications in several chemical, food processing and energy related industries. The pulverized coal combustor, coal gasification systems, and dryforming processes for paper are few examples. In these, there is a need to reliably meter the flow-rate of particle laden fluid mixtures. In pneumatic conveying, the solid particles are mixed with air either under the influence of gravitational force or due to the pressure force resulting from the pressure drop in an accelerating air flow. Typical materials used

for conveying are pulverized coal, ash, food grains, sand particles and dry chemicals [1]. Wagenknecht and Bohnet [2] were the first and foremost to study an injector type particle feeder. In their study, a high-velocity air stream exists from a primary nozzle and entrains the particles into a secondary nozzle, where the necessary pressure is built up for conveying the particles. Following them, Chellappan and Ramaiyan [3] and Westaway et al. [4] carried out investigations on the effects of important design parameters of a gassolid injector feeder, which can be used in pneumatic conveying systems. All these designs utilize a nozzle to create high velocity air flow that carries the solid particles. Apart from the pneumatic conveying application, powder dispensers are designed for particle size measurements [5], including that of aerosols. These devices are usually oriented horizontally. Recently, dry particle dispersion methods, process and the dispersion efficiency have been discussed in a review article by Masuda [6].

The techniques currently available have limited capability for fine control of feeding rate. In certain applications involving fuel characterization and energy studies, the quantity of particle injected has to be precisely known and the particles are to be fed in wide ranges of quantities. In these cases, there is a need to have a particle injecting device, which allows for simple and precise control of both fluid flow and particle feed rates. A typical solid-particle injector is designed, developed and calibrated in this study. This injector, which is oriented upwards, utilizes the venturi effect observed when the flow passes through a reduced area passage such as in orifice plates, converging and diverging passages and nozzles. An orifice plate is used to create the venturi effect in this study because of ease of manufacture and cost effectiveness. The entire setup consists of a pipe having an orifice plate, connected to another pipe having openings of on its periphery from which the powder is easily fed into the orifice plate, and an extension pipe connected to this, to make the particle laden air flow develop further. Furthermore, since the powder does not pass through the orifice hole, erosion of the hole is prevented. The control of the powder feed rate is basically achieved by varying the air flow-rate and/or by changing the area of the side openings in the pipe. Three design configurations are studied and a thorough calibration is done for each case for varying air flow-rates.

### 2. Solid-Particle Injector

The injector device comprises of three parts:

- 1. A lower pipe with an orifice plate. A straight hole, sharp edged orifice has been employed.
- 2. A middle section, which is a pipe with openings in its periphery and with a dust feeder.
- 3. A pipe on the top of the middle section to further develop the flow.

The assembly is shown in Fig. 1. The upper (124.5 mm long and 3.3 mm thick) and lower (135.5 mm and 3.3 mm thick) pipes can be inserted to the middle section and socket head screws are used to secure these sections intact (Fig. 1). The top end of the lower pipe has an orifice plate of 1 mm thickness with a hole at its center having a diameter of 1 mm. The length of the middle section is 50.8 mm. It has provisions to insert the top and lower pipes at its top and lower ends. The middle section has peripheral openings as shown in Fig. 1. Both the number and size of these openings have been varied in this study.

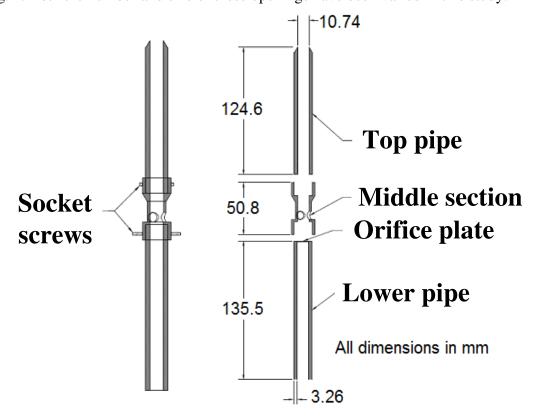


Figure 1: Solid particles injector

To study the effects of size and number of openings in the middle section, three different middle sections are used:

- 1. First middle section (case 1) has six circular openings each with a diameter of 5 mm, amounting to a total particle-feed area of 118 mm<sup>2</sup>.
- 2. The second one (case 2) has three circular openings with 8 mm diameter each and a total opening area of 151 mm<sup>2</sup>.
- 3. The third middle section (case 3) consists of three rectangular openings each with an area of 90 mm<sup>2</sup> and a total opening area of 270 mm<sup>2</sup>.

A particle feeder which comprises of an inverted hollow 60° inverted cone made of acrylic is connected around the middle section as shown in Fig. 2. The feeder is open at the top and has a smooth surface to minimize frictional losses. The feeder rests on the outer portion of socket screws.

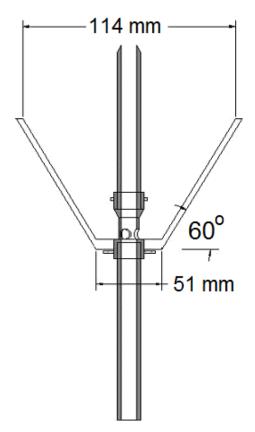


Figure 2: Injector with particle feeder

### 3. Experimental Setup and Procedure

A schematic of the experimental setup is shown in Fig. 3. Air is supplied with an upstream pressure of 5 psig (34474 Pa) to the lower pipe through a pressure regulator, valve and air flow meter. The air flow meter can supply air at flow-rates ranging from 0 liters per minute (lpm) to 10 lpm in increments of 0.5 lpm. The accuracy of the flow meter at its highest flow-rate is within ±4%. Pittsburgh seam coal dust is used in the present study, and has a particle size in the range of 90 - 106 micro-meters (μm), a reported constant sphericity value of 0.73 [6], and a bulk density of 0.553 g/cm<sup>3</sup>. A particle collection pan of 300 mm diameter, as shown in Fig. 3, is used to collect the particles injected out of the top pipe. The injector and the particle feeder are secured in a support frame as shown in Fig. 3 and the entire assembly is weighed using a Cole Palmer load cell (capacity of 4.2 kg with a sensitivity of 0.01 g). The uncertainty in the mass measurement is ±0.03 g. The ring stand is used to support the collection pan so that the particles collected in the pan will not be weighed by the load cell.

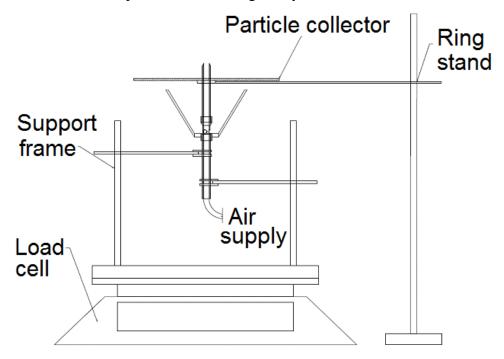


Figure 3: Schematic of experimental setup

The experiments are carried out in atmospheric temperature equal to 300 K. The experimental procedure comprises of fitting the desired middle section, filling the conical

feeder with coal dust and then turning the air supply on. The flow meter is set to deliver the desired air flow-rate and the load cell output is recorded as a function of time. The recording of the mass loss rate is continued for 5 minutes and then stopped. Each experiment is repeated at least three times to ensure consistent results within  $\pm 2\%$ . An instantaneous photograph showing particle injection and collection in the pan is presented in Fig. 4 (case 2: three circular openings with 8 mm diameter each at an air flow-rate of 5 lpm).



Figure 4: An instantaneous photograph showing particle injection and collection in the pan

### 4. Results and Discussions

The particle injection rate is mainly dependent on the flow-rate and the size of the side openings in the three middle sections studied. For each middle section, air flow-rate is varied from 1.57 grams per minute (gpm) to 15.7 gpm (1 lpm to 10 lpm). However, no entrainment is recorded for 1.57 gpm, irrespective of the middle section used. As flow-rate increases, the entrainment of coal dust into the pipe increases due to increased venturi effect. This is shown in Fig. 5, where the mass of dust injected (grams per minute) is plotted as a function of the air-flow rate in gpm. The variation of the coal dust injection rate is non-linear for air flow-rates up to 9.4 gpm. After this, as the air flow-rate increases, the dust injection rate follows an almost linearly increasing trend. For low air flow rates (< 9.4 gpm), the dust injection is found to be the highest for case 3 (three rectangular openings with a total area of 270 mm<sup>2</sup>). For instance, at an air flow-rate of around 7.9

gpm, the entrainment for case 3 is around 15% more than that of case 2 and 50% more than that of case 1 (six circular openings each with a diameter of 5 mm). In fact at an air flow-rate of around 3.15 gpm (2 lpm), maximum deviation is recorded between case 3 and case 2 (35% more for case 3 than case 2) and between case 3 and case 1 (89% more for case 3 than case 1). When the air flow-rate is increased beyond 9.4 gpm (6 lpm), the variation between all the three middle section cases are almost the same with a maximum deviation within ±3%. This shows that when the inertia of air is increased, venturi effect is increased and as a result, the air is capable of carrying the coal dust fed to the orifice plate irrespective of the side opening area. At low flow-rates, the pressure drop created due to venturi effect is partly overcome by the friction, which the inner surfaces of the holes offer to the coal dust flow. Therefore, the injection rate is higher for large openings where the friction effects are minimal. This means that the middle section opening area is critical only for lower air flow-rates and as the air flow-rate increases an optimum opening area can be employed, which in this case can be either case 2 or case 3. The usage of variable opening area, if designed carefully, can be employed to control the particle feeding rate even more precisely.

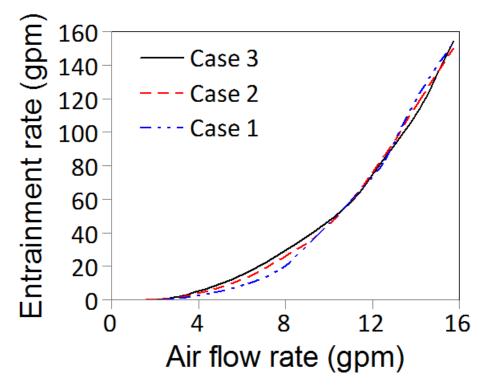


Figure 5: Particle entrainment rate in grams per minute (gpm) as a function of air flowrate in gpm for all the three middle section cases

### **5. Conclusions**

Design, development, and calibration of a novel solid particle injector have been reported. The injector uses the pressure drop in the air flow across an orifice plate fitted in a circular pipe (venturi effect), to naturally entrain micron-sized solid particles such as coal dust. Coal dust is continuously fed from a feeder located outside the pipe into the orifice plate through peripheral openings in the pipe. Three types of designs for the peripheral openings, in terms of the shape, size and number are evaluated by testing which one of them results in maximum particle entrainment, especially at lower air flow-rates. Calibration of the device is carried out by precisely recording the mass loss rate of the coal dust as a function of volumetric flow-rate of air. The results show that there is an optimum area for the side openings, at which the injector performance will be the best for the given pipe and orifice-hole sizes. The entrainment rate is found to be a non-linear function of the flow-rate for low air flow-rates (till around 9 grams per minute) and dependent on the net area of the openings. For higher air flow-rates, the entrainment becomes almost independent of the opening area and also it becomes an almost linear function.

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### Chapter 3

# Study of Interaction of Entrained Coal Dust Particles in Lean Methane – Air Premixed Flames

Y. Xie, V. Raghavan and A. S. Rangwala, "Interaction of Coal Dust Particles on Lean Premixed Methane-air Flames," Combustion and Flame (Under Review)

#### Abstract

This study investigates the interaction of micron-sized coal particles entrained into lean methane – air premixed flames. In a typical axisymmetric burner, coal particles are made to naturally entrain into a stream of the premixed reactants using an orifice plate setup. Pittsburgh seam coal dust, with three particle sizes in the range of 0 to 25 µm, 53 to 63  $\mu$ m, and 75 to 90  $\mu$ m is used. The effects of different coal dust concentrations (10 – 300 g/m<sup>3</sup>) at three lean equivalence ratios,  $\phi$  (methane-air) of 0.75, 0.80 and 0.85 on the laminar burning velocity are determined experimentally. The laminar burning velocity of the coal dust-methane-air mixture is determined by taking a shadowgraph of the resulting flame and using the cone-angle method. The results show that the addition of coal dust in methane-air premixed flame reduces the laminar burning velocity at particle size of 53 to 63 μm and 75 to 90 μm. However, burning velocity promotion is observed for 0 to 25 μm particles at  $\phi = 0.80$ . Two competing effects are assumed involved in the process. The first is burning velocity promotion effect that the released volatile increases the gaseous mixture equivalence ratio and thus the burning velocity. The second is the heat sink effect of the coal particles to reduce the flame temperature and accordingly the burning velocity. A mathematical model is developed based on such assumption and it can successfully predict the change of laminar burning velocity at various dust concentration. Furthermore, the implication of this study to coal mine safety is discussed.

### NOMENCLATURE

A	Parameter characterizing rate of	$n_s$	Number of particles
В	vaporization of particles, Eq. 2 Frequency factor characterizing	$\dot{n}_{air}$	Number of mole of air per unit time
Б	rate of gas phase oxidation of	$\dot{Q}$	Heat release rate
	gaseous fuel	$\dot{q}''$	Heat flux to particles in Fig. 5
$\boldsymbol{B}$	Burner base width	R	Gas constant
$C_p$	Heat capacity of air	r	Mean radius of particles
$C_s$	Heat capacity of solid particle	$S_u$	Burning velocity
$C_{total}$	Heat capacity of particle-gas	$T_b$	Flame temperature based on
	mixture		original premix mixture
c	Density	$T_f$	Flame temperature with particles
E	Activation energy characterizing	$T_f'$	Promoted flame temperature due
1.	the gas phase reaction		to locally increased equivalence
h K	Heat of the flame cone Heat conductivity of air	T."	ratio  Reduced flame temperature due to
	•	$T_f''$	Reduced flame temperature due to
$L_v$	Latent heat of vaporization		heat sink effect of particles
<i>M</i>	Molecular mass	$t_r$	Residence time
$m_{CH_4}^{\prime\prime\prime}$	Mass of air presents in 1 m <sup>3</sup>	U	Average flow velocity at burner
$m_{air}'''$	Mass of methane presents in 1 m <sup>3</sup>		nozzle
$m_{\scriptscriptstyle fuel}^{\prime\prime\prime}$	Mass of fuel presents in 1 m <sup>3</sup>	$\dot{V}_{air}$	Volumetric flow rate of air
$m_v'''$	Mass of vapor presents in 1 m <sup>3</sup>	$\dot{V_{_{CH_{_{4}}}}}$	Volumetric flow rate of methane
n	Temperature exponent	V	Volume
	characterizing rate of vaporization	$\dot{w}_{v}^{\prime\prime\prime}$	Rate of vaporization of fuel
	of coal particles in Eq. 2	·	particles
	1	X	Spatial coordinate
		$Z_e$	Zeldovich number
<b>C</b> 1		G 1	• .4
	Symbols  Flome cone angle	Subsc	Adiabatic condition based on
lpha	Flame cone angle $=1/Z_e$ , expansion parameter	b	original gas phase condition
$\rho$	Density of the solid-gas mixture	d	Devolatilization
$ ho_s$	Density of the particle	f	Flame
$\delta$	Thickness of devolatilization zone	S	Solid particle
$\phi$	Original gaseous mixture	и	Conditions in the ambient
,	equivalence ratio		condition
$\phi_s$	Equivalence ratio of coal particles and air	v	Vapor

### 1. INTRODUCTION

Many materials that are virtually non-flammable in bulk form become explosive if dispersed as a cloud of fine particles in air. From a combustion viewpoint this can be treated as both a benefit and a hazard. In industries that manufacture, transport, process and/or use combustible dusts, accidental dust deflagrations represent a real hazard to both personnel and equipment. An example is the recent coal mine explosion in West Virginia (April 5, 2010) killing 29 miners and considered as one of the most disastrous mining accidents in US history. Interestingly, most coal mine explosions often involve both a methane deflagration combined with fugitive coal dust that is collected by the combustion wave as it progresses through the mine. The physical and chemical processes involved during the travel of a combustion wave through a flammable gas-dust-air mixture are shown in Fig. 1. Three distinct steps are identified as shown in Fig. 1. First, the dust deposited on the floor, walls, and ceiling can be lifted up by the pressure blast of the initial methane explosion causing a cloud of dust to be suspended in the air. When the methane-air flame front meets the dust cloud, the coal particles pyrolyze and contribute volatile vapor to the methane-air mixture. The coal particles can also cause instabilities, which could potentially alter the structure of the premixed flame. Greenberg et al. [1] have shown that adding combustible liquid droplets to a gas flame can increase the burning velocity under certain conditions. Based on droplet concentration and size, Suard et al. [2] identified different spreading regimes for such droplet – gas – air flames. Goral et al. [3] studied upward propagation of flames in a lean methane-air mixture to which had been added inert particles (sand). It was found that the upward flame velocity increased with increasing sand grain size. And such increase was attributed mainly to the enhanced combustion due to the micro-turbulence generated in the wake of the falling particles. Wendt and Graves [4] studied the flammability of coal dust in a laminar opposed jet diffusion flame. However, the interaction between solid combustible dust particles and a gaseous premixed flame have been rarely investigated in combustion literature and are the focus of the current study.

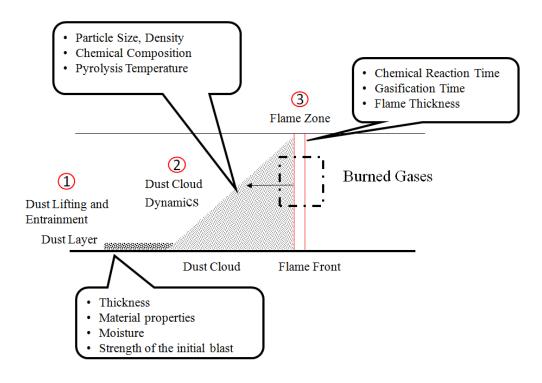


Fig.1: Dust explosion process

### 2. EXPERIMENTAL APPARATUS

Fig. 2 (a) is a schematic of the experiment used for measurement of laminar burning velocity and dust entrainment rate. The design of such a burner is based on the Bunsen burner, with the difference that the side openings entrain coal dust particles flow instead of air. Specific details of the dust injection using such burner are discussed by Xie et al. [5]. The burner is made of a steel tube with inner diameter of 10.2 mm and wall thickness of 1.2 mm. A 1 mm thick acrylic plate with a 1 mm diameter orifice is installed inside the steel tube, 150mm away from the burner top. Entrainment of dust is achieved through three 7.5 mm wide and 9 mm long slot openings located above the orifice plate as shown in Fig. 2 (b). A brass jacket tube (highlighted in blue) whose inner diameter is 0.1mm larger than the outer diameter of the steel tube and secured by two socket head screws is used to adjust the opening size and thus the dust entrainment. The coal dust is filled in an inverted cone-shaped acrylic container which is also attached to the steel tube. The cone angle of the container is equal to 60° which represent the critical angle of repose of dust

particle size ranges used in this study. The repose angle was determined using an experimental method discussed by Botz et al. [6]. The adjustable burner and its attachments are secured in a support frame and the entire assembly is kept over the *Cole-Parmer Symmetry PR 4200* load cell. The load cell has a total weighing capacity of 4.2 kg with a sensitivity of 0.01 g and the factory uncertainty in the mass measurement is  $\pm$  0.03 g. A ring stand kept outside the load cell is used to support a collection pan. Pittsburgh seam bituminous coal dust with particle size in the range of 0 to 25  $\mu$ m, 53 to 63  $\mu$ m, and 75 to 90  $\mu$ m are used in the experiment. The size ranges are obtained by *Retsch AS 300* sieve shaker. Compressed air and methane (99.99% purity) cylinders are used to supply the burner with an upstream pressure of 0.5 bar. Each gas flow is controlled by a *SIERRA* Model 100 mass flow controller with an accuracy of  $\pm$ 1% of its flow capacity. The desired gases are mixed in a "tee" connection before it enters the burner.

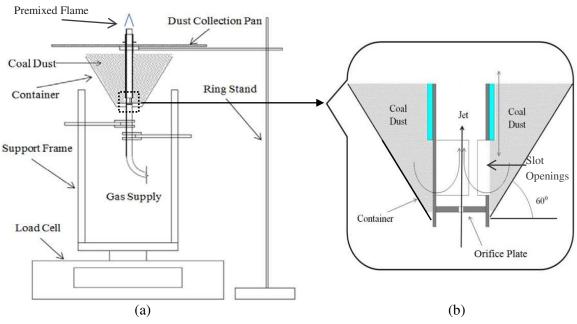


Fig. 2: Experiment apparatus: (a) dust burner and the weighing assembly; (b) the details of the dust entrain mechanism

Direct shadowgraph technique is used to capture the flame cone with and without coal dust injection. The set-up is shown in Fig. 3. A 420 W projector lamp is converted into a

point light source and placed one focal length away from a double convex lens. A Canon EOS 5D single-lens reflex (SLR) camera with a macro-lens (Canon EF100/2.8 Macro USM) that has a minimum focal length of 31 mm is placed behind the flame along the center axis of the parallel light beam. The camera is manually adjusted where shutter speed of 1/4000 sec, ISO of 1600, and aperture of 2.8 are used for the optimization. A sample image of the actual flame and the corresponding shadowgraph image obtained by the macro-lens are shown in Fig. 4 (a) and 4 (b).

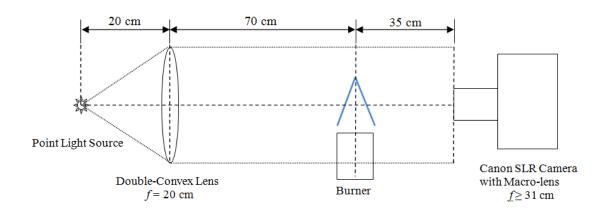


Fig. 3: Shadowgraph set-up

The main advantage of the shadowgraph is that it can capture clearly the flame cone even when the flame is loaded with high concentration of burning particles. As shown in Fig.4 (a), it is hard to locate the edge of the flame cone using conventional photograph techniques while the flame can be easily identified in the shadowgraph as shown in Fig. 4 (b). For each dust concentration and gaseous fuel equivalence ratio, a minimum of 15 images are processed by an image process algorithm programmed in Matlab and the corresponding average cone angle is used to estimate the burning velocity. A sample of the processed image is shown in Fig. 4 (c). The algorithm converts the shadowgraph into a grey-scale image and detects the cone edge where a significant change in the normalized intensity (a value from 0 to 1) occurs on each row within the preselected boundary that encloses the flame cone. The detected cone edge is shown as two clusters of red or green dots in Fig 4. (c). Slopes that connect each dot on one side of the cone is

calculated. Then a best-fit line for all the detected dots, shown as black solid line in Fig. 4 (c), is used to obtain the cone half angle  $\alpha$  where slope of the black solid line is the mean of all slopes. The standard deviation of the angle experimentally measurement is within  $\pm$  1.5°.

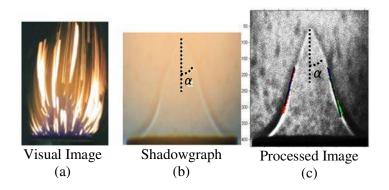


Fig. 4: Example of flame images

The laminar burning velocity is obtained by using angle method where

$$S_u = U \cdot \sin(\alpha) , \qquad (1)$$

where  $S_u$  represents the laminar burning velocity, U is the average flow velocity, and  $\alpha$  is the cone half-angle as shown in Fig. 4.

### 3. MATHEMATICAL MODEL

A mathematical model is developed to explain the observation made in the experiment and predict  $S_u$  with particles whose sizes are not included in the experiment. Fig. 5 illustrates the interaction of the particles with the premixed flame and is used as a basis to develop a model to estimate the change in  $S_u$ .

The path of a coal particle, assumed to be along a flow streamline, is shown in Fig. 5. The particle absorbs the heat from the flame while it travels though the devolatilization zone as illustrated by  $\dot{q}''$  in the inset to Fig. 5. The temperature of the particle is assumed to be equal to the gas temperature and once it reaches  $T_{\nu}$  the devolatilization is initiated thereby

releasing gaseous volatiles as shown by  $\dot{m}_v^m$ . This additional fuel released from coal particle can increase the equivalence ratio locally in the flame. For a lean mixture flame, increased equivalence ratio can promote  $S_u$ . In Fig. 5,  $T_f$  is the temperature of the flame and  $T_v$  represents the approximate temperature at which devolatilization initiates. The thickness of the zone between  $T_f$  and  $T_v$  is denoted by  $\delta$ . It is assumed that the particle will pyrolize and release volatiles when it traverses the distance  $\delta$ . The height and the width of the flame cone are represented by h and b respectively. The cone half angle is designated as  $\alpha$ . In addition, the particle also acts as a heat sink thereby reducing the flame temperature and thus  $S_u$ . These two effects are competing in nature and affect the flame simultaneously.

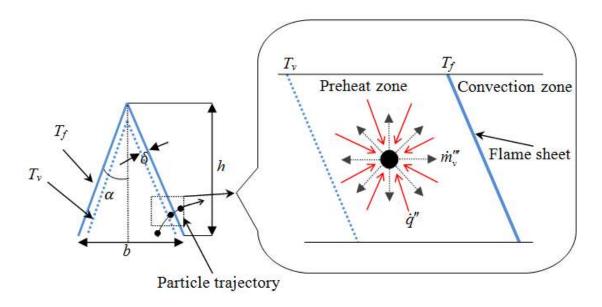


Fig. 5: Illustration of parameters involved ( $\dot{q}''$  is the heat flux from flame to the particle;  $\dot{m}_{v}'''$  is volatile release rate per unit volume of the mixture;  $T_{f}$  is fame temperature;  $T_{v}$  is coal devolatilization initiation temperature;  $\delta$  is the thickness of devolatilization zone;  $\alpha$  is the cone half angle; b is the burner diameter; h is the cone height.)

To simplify the analysis, the two processes are decoupled into separate and parallel process in the model. The change in the flame temperature due to each effect is calculated separately and then superimposed to obtain the estimated true flame temperature.

### 3.1 Effect of equivalence ratio promotion

Coal particles can release volatiles into the gaseous mixture as the result of the elevated particle temperature. The volatiles are the additional gaseous fuel which increases the equivalence ratio locally. In order to estimate the amount of volatiles present in the gaseous mixture, it is necessary to explore the rate of the devolatilization process and quantify such rate.

There are four possible methods to estimate the devolatilization. The first method is based on the assumption that all volatiles are released when particle reach a given temperature. However, such method excludes the fact that devolatilization as it is also a time depended process. The second method is to treat the particles as liquid droplets where the vaporization can be estimated as discussed by Suard et al. [2]. This method, however, does not capture the non-linear behavior of devolatilization of coal while the droplet evaporation is fairly linear. The devolatilization can be solved by a set of equations which is based on the Arrhenius law as described by Solomon and Colket [7]. Or devolatilization rate can be estimated based on a temperature depended power-law relation as proposed by Seshadri et al [8]. Since the method based on Arrhenius law requires high computational cost, the model uses a temperature depended power law is used in this study.

Seshadri et al. [8] used the expression below to predict devolatilization rate of coal particles,

$$\dot{m}_{v}^{\prime\prime\prime} = A n_{s} 4 \pi r^{2} T_{s}^{n}, \qquad (2)$$

where the unit of  $\dot{m}_{v}^{m}$  is mass of gaseous fuel vaporized per unit volume per second. The quantities A and n are constants which are presumed to be known and will be discussed in a later section. The particle temperature is denoted as  $T_{s}$  and it is assumed that

 $T_s = \frac{T_b + T_v}{2}$ , where  $T_b$  is the adiabatic flame temperature based on the methane-air equivalence ratio and  $T_v$  devolatilization initiation time. For coal particles, devolatilization initiates at about  $T_v = 600$ K [9]. The value of  $T_b$  is calculated using an

equilibrium solver based on the minimization of free energy [10]. The total volume of particles per unit volume of mixture is estimated by dividing the dust concentration  $c_s$  (g/m³) by the particle density  $\rho_s$  (g/m³). The number of particles per unit volume ( $n_s$ ) is then equal to  $n_s = (c_s / \rho_s)/V_s$ , where  $V_s$  is the volume of a single particle.

The duration of devolatilization needs to be determined to use Eq. 2 to estimate the total amount of vaporized fuel. Devolatilization is limited to a narrow band with thickness of  $\delta$  which is the region close to the reaction zone on the unburned side where temperature increases from  $T_v$  to  $T_b$  as shown in Fig. 5. The thickness of this zone is estimated by applying an energy balance.

$$k\frac{d^2T}{dx^2} - \rho UC_{total}\frac{dT}{dx} = 0, (3)$$

with boundary condition that x=0,  $T=T_v$ , and  $x=\delta$ ,  $T=T_b$ . The specific heat of the mixture  $C_{total}=C_P+\frac{4\pi r^3C_s\rho_s n_s}{3\rho}$ , U is the averaged flow velocity,  $\rho$  is the density of the mixture, and thermal conductivity is assumed to be as same as air of  $k=0.052W/K\cdot m$  at  $400^{\circ}\mathrm{C}$  which is approximately equal to the average temperature in the devolatilization zone.

Integrating Eq. 3, the thickness of the devolatilization zone can be obtained as

$$\delta = \frac{k(T_b - T_v)}{\rho U C_{total} (T_b - T_u)},\tag{4}$$

where the density of the mixture  $\rho = \frac{\rho_{air}\dot{V}_{air} + \rho_{CH_4}\dot{V}_{CH_4}}{\dot{V}_{air} + \dot{V}_{CH_4}} + c_s$ , and the averaged flow

velocity 
$$U = (\dot{V}_{air} + \dot{V}_{CH_4}) / \left(\frac{\pi b^2}{4}\right)$$
.

The time of devolatilization or the residence time of particles is then given by:

$$t_r = \left[\frac{\delta}{\sin(\alpha)}\right]/U. \tag{5}$$

The devolatilization time is used to estimate the total mass of released volatiles per unit volume during the passage of dust particles through  $\delta$  and is given by:

$$m_{v}^{\prime\prime\prime} = \dot{m}_{v}^{\prime\prime\prime} t_{r}. \tag{6}$$

This additional fuel is assumed to be CH<sub>4</sub> for simplicity as suggested by Seshadri et al. [8] and added to the original gaseous mixture to obtain a new equivalence ratio. The new amount of gaseous fuel per unit volume in the mixture per unit volume is denoted as  $m_{fuel}^{\prime\prime\prime}$  and can be estimated as  $m_{fuel}^{\prime\prime\prime} = m_{CH_4}^{\prime\prime\prime\prime} + m_{v}^{\prime\prime\prime\prime}$  where  $m_{CH_4}^{\prime\prime\prime\prime}$  is the mass of the original methane per unit volume of the mixture. Accordingly the new equivalence ratio can be calculated as  $\phi = 9.52 \cdot \left(\frac{m_{fuel}^{\prime\prime\prime\prime}}{M_{CH_4}} / \frac{m_{air}^{\prime\prime\prime\prime}}{M_{air}}\right)$  where the coefficient 9.52 is the ratio of numbers of moles of methane to air when  $\phi$  equals to 1.

For the original mixture, the mass of methane and air present per unit volume are given by:

$$m_{CH_4}''' = \frac{\left(\frac{\dot{V}_{CH_4}P}{RT_u}\right)}{\left(\dot{V}_{air} + \dot{V}_{CH_4}\right)} M_{CHL_4},\tag{7}$$

$$m_{air}''' = \frac{\left(\frac{\dot{V}_{air}P}{RT_u}\right)}{\left(\dot{V}_{air} + \dot{V}_{CH_4}\right)}M_{air},$$
(8)

where  $P = 101,354 \, \mathrm{Pa}$ ,  $T_u = 293 \, \mathrm{K}$ , and  $R = 8.314 \, \mathrm{J/K} \cdot \mathrm{mole}$ . The molecular weights of air and methane are  $M_{air} = 28.97 \, \mathrm{g/mole}$  and  $M_{CH_4} = 16.04 \, \mathrm{g/mole}$ .  $\dot{V}_{air}$  and  $\dot{V}_{CH_4}$  are the volumetric flow rates of air and methane obtained experimentally using the mass flow controllers.

With this new equivalence ratio, a new flame temperature  $T_f'$  is estimated by an equilibrium solver [10]. The calculated results are present in Fig. 6 for coal particles at size of 75-90  $\mu$ m with original methane-air equivalence ratio of 0.75, 0.80, and 0.85. In

Fig. 6,  $\phi_s$  represents the equivalence ratio in terms of coal dust and air. As shown in Fig. 6, the flame temperature increases with increased dust concentration. This is because the increased dust concentration causes more volatiles to be released and a higher equivalence ratio can be obtained. Consequently, the flame temperature and the laminar burning velocity are promoted.

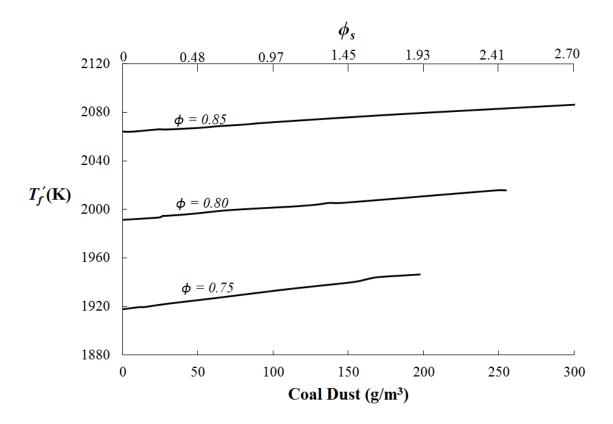


Fig. 6: Adiabatic flame temperature based on  $\phi$  with additional gaseous fuel released from coal particles in the size range of 75-90  $\mu$ m.

### 3.2 Heat sink effect of coal particles

In addition to a local increase in equivalence ratio, a coal particle will also act as a heat sink as it absorbs the heat from flame to raise its temperature for devolatilization. Two aspects are considered for the heat sink effect: 1. the heat used to raise the temperature of coal particles from ambient condition to the flame temperature; 2. the heat used as devolatilization heat for the coal particles. In order to model these two aspects, three assumptions have been made: 1. the heat release rate of the flame is assumed to remain

constant; 2. coal particles simply act as an inert particle which is also able to devolatilize to absorb additional energy from the flame; and 3. coal particles reach the flame temperature when they reach the flame sheet. Based on these three assumptions, the heat released to raise the gas temperature per unit time for flame without dust equals the sum of heat release to raise the temperature of gas and particles and heat of gasification per unit time. Therefore, each term should be determined to estimate the flame temperature that accounts for heat sink effect. It should be noted that the radiation is ignored from the analysis.

First, the heat released for flame without dust needs to be calculated. For a lean or stoichiometric methane-air mixture, the chemical reaction for the combustion with equivalence equal to  $\phi$  is given by

$$\frac{\phi}{2}CH_4 + (O_2 + 3.76N_2) \Rightarrow \frac{\phi}{2}CO_2 + \phi H_2O + 3.76N_2 + 2(1 - \phi)O_2 . \tag{9}$$

The total heat released for the combustion shown in Eq. 9 is solved by energy balance:

 $[(T_b - T_u)\sum C_p \cdot n_{product}]$  for  $\frac{\phi}{2}$  mole of methane or 4.76 mole of air where  $T_b$  is the adiabatic flame temperature which is estimated using an equilibrium solver [10]. Thus, the heat release rate of the premixed flame without presence of coal dust for a given flow of air and  $\phi$  can be calculated as

$$\dot{Q} = \left[ \left( T_b - T_u \right) \sum_p C_p \cdot n_{product} \right] \frac{\dot{n}_{air}}{4.76}, \tag{10}$$

where  $\dot{n}_{air}$  is the number of moles of air supplied per unit time.

Based on the first assumption, the flame with particles releases the same amount of heat while it is also affected by the volatile gasification and temperature rise of particles. Therefore the flame temperature  $T_f''$  can be estimated using energy conservation as shown in Eq. 11

$$\dot{Q} = \left[ \left( T_f'' - T_u \right) \sum_{p} C_p \cdot n_{product} \right] \frac{\dot{n}_{air}}{4.76} + \dot{n}_s C_s \left( T_f'' - T_u \right) + L_v / t , \qquad (11)$$

where  $\dot{n}_s$  represents the local number of particles per unit volume per unit time that are passing through the flame

$$\dot{n}_s = (\dot{V}_{air} + \dot{V}_{CH_A}) n_s \rho_s V_{particle} / M_c , \qquad (12)$$

where  $M_c$  is the molecular weight of carbon (12g/mole).

Similarly  $L_v$  in Eq.11 represents the heat of gasification which is assumed to be a fraction of the heat produced and given by

$$L_{v} = \chi(m_{v}^{"'} \cdot V_{d} \cdot \Delta h_{CH_{A}}), \tag{13}$$

where  $\chi$  is assumed to be 0.01 as suggested by Seshadri et al. [8],  $\Delta h_{CH_4}$  is the heat of combustion of methane, and the volume of the devolatilization zone is the conical space with a thickness of  $\delta$  as shown in Fig. 5 given by

$$V_d = \frac{1}{3}\pi h \frac{b^2}{4} - \frac{1}{3}\pi \left(h - \frac{\delta}{\sin(\alpha)}\right) \left(\frac{b}{2} - \frac{\delta}{\cos(\alpha)}\right)^2. \tag{14}$$

The height of the flame cone is denoted as h, the diameter of the burner is represented by b, and  $\alpha$  is assigned for the cone half-angle.  $L_{\nu}/t$  represents the heat consumed by gasification process where t denotes the time the particle spends in the devolatilization zone and is estimated by Eq. 5. Re-arranging Eq. 11, the new flame temperature  $T_f''$  can be obtained as

$$T_f'' = \frac{\dot{Q} - L_v / t}{\frac{\dot{n}_{air}}{4.76} \sum_{p} C_p \cdot n_{product} + \dot{n}_s C_s} + T_u.$$

$$\tag{15}$$

Using Eq. 15, the calculated results of  $T_f''$  is plotted in Fig. 7 for coal particle with size range between 75 to 90  $\mu$ m and equivalence ratio of the gaseous mixture of 0.75, 0.80 and 0.85. As observed in Fig. 7, the flame temperature reduces as the concentration of dust increases for all three equivalence ratios.

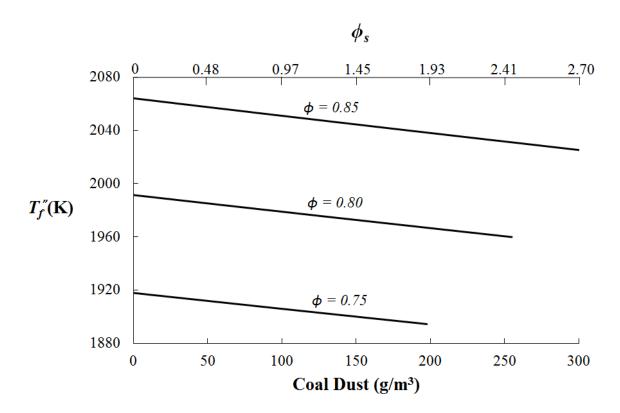


Fig. 7: The flame temperature based on the heat sink effect of coal particles at 75-90 µm

### 3.3 Combined effects

The combined effect of particle devolatilization and corresponding equivalence ratio promotion and heat sink effect as they interact with a premixed flame are accounted for by calculating an average flame temperature  $T_f = (T'_f + T''_f)/2$ . The corresponding flame temperature is then used to estimate  $S_u$  using the model developed by Seshadri et al. [8]:

$$S_{u} = \sqrt{\frac{2Bk\varepsilon^{2}}{\rho C_{total}}} \exp\left(-\frac{E}{RT_{f}}\right), \tag{16}$$

where 
$$\varepsilon = 1/Z_e$$
,  $Z_e = \frac{E(T_f - T_u)}{RT_f^2}$ .  $B = 3.5 \times 10^6$  /mole·s and  $E = 88800$  kJ/mole are

chosen to match the calculated burning velocity with burning velocity obtained by experiment for flames without dust injection.  $\rho$  is the density of the mixture. The

constants  $A = 0.034 \, \text{g/m}^2 \, K \cdot \text{s}$  and n = 1.1 are introduced in Eq. 2 to obtain the calculated burning velocities of dust-gas mixtures to match with the experiment measurements. The calculated burning velocities and experiment data are presented in Fig. 8. Due to the strong cohesive force, the dust entrainment rate is very unsteady for smaller particles. Therefore only one data point is obtained for 0 to 25  $\mu$ m coal particles, and no data is obtained for 53 to 63  $\mu$ m at  $\phi$ =0.75.

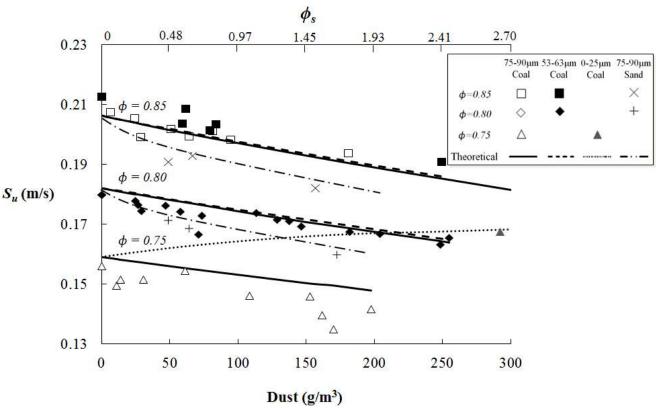


Fig. 8: Burning velocity methane-air premixed flame with injected coal particles and sand particles at lean conditions (Experimental data are represented in different shape of points and theoretical prediction is shown in lines.)

As shown in Fig. 8, the interaction of the coal particles with a laminar premixed methaneair flame reduces  $S_u$  when the particle size is larger than 25 µm. However, for 0 to 25 µm coal particles,  $S_u$  is promoted as shown by the solid grey triangular symbol in Fig. 8. It should be noted that at the same size range of 75 to 95 µm, the reduction in  $S_u$  for coal particles is less than that of sand. This observation demonstrates that the vaporized fuel is acting against the heat sink effect to maintain  $S_u$  as compared to the sand particles only the heat sink effect plays a role. It is shown in the figure that the slope of  $S_u$  decreasing trend of  $S_u$  of 53 to 63 µm coal particles is less than the ones with size of 75 to 95 µm. It indicates that as the size of coal particle reduces, the increased equivalence ratio increases its significance compared with the heat sink effect. When the size of coal particles are as small as 0 to 25 µm, the influence of heat sink plays a minor role while the increased equivalence ratio affects  $S_u$  such that it increases.

The model is also used to calculate  $S_u$  with different dust concentrations at various equivalence ratios of methane-air mixture and coal particle sizes. The results are plotted in Fig. 9 where each color represents one original gaseous fuel equivalence ratio and different line type stands for a different particle size.

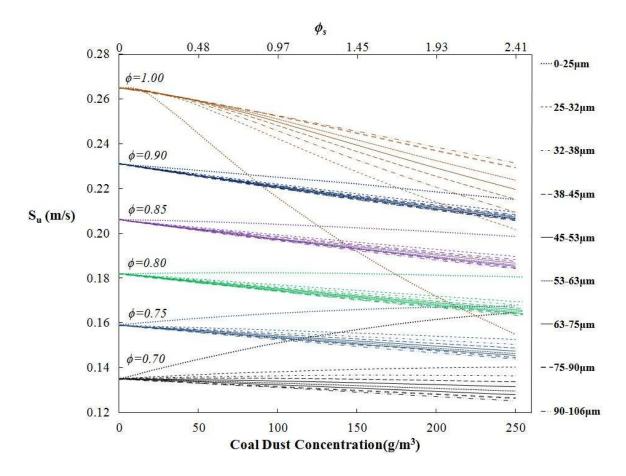


Fig. 9: Burning velocity for coal particle with different sizes at lean conditions

As shown in Fig. 9, the slope of the curve of  $S_u$  vs. dust concentration is negative but it gets flatter as the particle size decreases at a lean condition. However opposite trend is

shown in Fig. 9 when the  $\phi$  equals 1 where the curve has a steeper slope with smaller particles. This can be explained by the increased specific surface area for smaller particles; and thereby the amount of released fuel is increased to promote the equivalence ratio locally. Therefore  $S_u$  increases as  $\phi$  increases locally for a lean mixture while it decreases as  $\phi$  increases locally as the stoichiometric mixture becomes fuel rich. It is also observed that the curve flatters as the mixture gets leaner while the particle size is kept the same; and the slopes become positive for 0 to 25  $\mu$ m particles where  $\phi$  ranges from 0.70 to 0.80 and particles with size range of 25 to 32  $\mu$ m at  $\phi = 0.70$ . This can be explained by the fact that a lower flow rate is supplied for a learner mixture in order anchor the flame in the experiment. The flow velocity U is slower as flow rate gets smaller. Then the devolatilization zone thickness  $\delta$  and residence time  $t_r$  as defined by Eq. 4 and Eq. 5 both get larger since U is the denominator in both equations. Therefore more vapors can be released due to a longer residence time  $t_r$ . Therefore the slope is flatter as the mixture gets leaner. Due to the larger specific surface area and longer residence time  $t_r$ , the slopes for the particles with size ranges from 0 to 25 µm and 25 to 32 µm becomes positive at range of 0.70 to 0.80 and 0.70 respectively.

### 4. DISCUSSION

The explosion in coal mine tunnels evolves through three stages: laminar flame, turbulent flame and deflagration. The Bunsen burner type experiment set-up in this study facilitates a better understanding of interaction of coal particles and flame at the first stage of the explosion. As shown in this study, small coal particles (0 to 32  $\mu$ m) have the ability to enhance the laminar burning velocity  $S_u$  of methane-air mixture under certain conditions. It is also found that larger size coal particles reduce  $S_u$ .

In a coal mine explosion, the initial pressure blast of methane-air explosion (fire-damp) can shake off the dust deposited on various surfaces. Small particles can easily suspend in air as compare to the larger ones. Therefore, the laminar methane-air flame will first interact with fine particles. This study shows that if the laminar flame is at lean condition, the flame velocity will be enhanced by small particles. As the flame develops to a

turbulent condition, more dust at various sizes are drawn in the flame by the turbulence eddies formed in front of the flame front. In a turbulent flame, eddies will greatly increase the heat and mass transfer within the devolatilization zone which enhances the fuel vaporization process. This also increases the residence time of particles leading to more vaporized fuel. Therefore, it is expected that in a lean turbulent flame the increased amount of vaporized fuel will counteract the heat sink effect for the particle sizes larger than 32  $\mu$ m. As a result, the flame velocity will be promoted by the presence of significant lager particles (as compared to the laminar cases). As the flame front is accelerated by the cumulated combustion products, a deflagration to detonation transition is likely.

### 5. CONCLUSION

In this study, a lab scale experiment has been conducted to understand the effects of the coal dust injected through a lean methane-air premixed flame. The laminar burning velocity  $S_u$  of the coal dust-methane-air mixture is determined by taking a shadowgraph of the resulting flame and using the cone-angle method. Experiment results show that the presence of coal dust at 53 to 63  $\mu$ m and 75 to 90  $\mu$ m reduces  $S_u$  at equivalence ratio of 0.75, 0.80, and 0.85 with dust concentration varies from 10 g/m<sup>3</sup> to 300 g/m<sup>3</sup>. However, particles with size at 0 to 25  $\mu$ m promotes  $S_u$  at  $\phi$ =0.80. It is assumed that the coal particle acts as gaseous fuel contributor to increasing the equivalence ratio while it is also acting as heat sink to quench the flame. Both effects have been investigated mathematically based on such assumption. A model has been developed to predict the burning velocity of premixed gas-methane flame with coal dust at various dust concentration. It is observed that under certain conditions the laminar burning velocity increases mainly due to the released volatiles which locally increase the mixture equivalence ratio and thus the burning velocity. The amount of released volatiles is governed by the surface area per unit mass of dust and the residence time. However, under certain conditions, the heat sink effect of the coal particles reduces the flame temperature and thus the burning velocity as well.

### 6. ACKNOWLEDGEMENT

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## Chapter 4 Conclusions and Future Work

This thesis consists of two parts of studies. The first study on a novel dust injector facilitates the second study where the interaction of premixed laminar methane-air flame with coal dust particles is investigated.

In the first study, design, development, and calibration of a novel solid particle injector have been reported. The injector uses the pressure drop in the air flow across an orifice plate fitted in a circular pipe (venturi effect), to naturally entrain micron-sized solid particles such as coal dust. Coal dust is continuously fed from a feeder located outside the pipe into the orifice plate through peripheral openings in the pipe. Three types of designs for the peripheral openings, in terms of the shape, size and number are evaluated by testing which one of them results in maximum particle entrainment, especially at lower air flow-rates. Calibration of the device is carried out by precisely recording the mass loss rate of the coal dust as a function of volumetric flow-rate of air. The results show that there is an optimum area for the side openings, at which the injector performance will be the best for the given pipe and orifice-hole sizes. The entrainment rate is found to be a non-linear function of the flow-rate for low air flow-rates (till around 9 grams per minute) and dependent on the net area of the openings. For higher air flow-rates, the entrainment becomes almost independent of the opening area and also it becomes an almost linear function.

In the second study, a lab scale experiment has been conducted to understand the effects of the coal dust injected through a lean methane-air premixed flame. The laminar burning velocity  $S_u$  of the coal dust-methane-air mixture is determined by taking a shadowgraph of the resulting flame and using the cone-angle method. Experiment results show that the presence of coal dust at 53 to 63  $\mu$ m and 75 to 90  $\mu$ m reduces  $S_u$  at equivalence ratio of 0.75, 0.80, and 0.85 with dust concentration varies from 10 g/m<sup>3</sup> to 300 g/m<sup>3</sup>. However, particles with size at 0 to 25  $\mu$ m promotes  $S_u$  at  $\phi$ =0.80. It is assumed that the coal particle acts as gaseous fuel contributor to increasing the equivalence ratio while it is also acting as heat sink to quench the flame. Both effects have been investigated mathematically based on such assumption. A model has been developed to predict the burning velocity of premixed gas-methane flame with coal dust at various dust

concentration. It is observed that under certain conditions the laminar burning velocity increases mainly due to the released volatiles which locally increase the mixture equivalence ratio and thus the burning velocity. The amount of released volatiles is governed by the surface area per unit mass of dust and the residence time. However, under certain conditions, the heat sink effect of the coal particles reduces the flame temperature and thus the burning velocity as well.

During the tests, it is found that it is difficult to obtain good data for particles that are extremely small, such as 0-25  $\mu$ m particles. Similarly there is no good experiment data for the tests with the 53-63  $\mu$ m particles at  $\phi$ =0.75 where the flow rate is smallest. This is mainly due to the high cohesive forces that exist between the particles which cause difficulty in entrainment with the current design. A higher flow rate is required to entrain the smaller particles where a stronger cohesive force exists. However, the lean flame blows off under such high flow rate. Furthermore, it is noted that the agglomeration is more obvious for the particles that are smaller than 45  $\mu$ m. Although the agglomeration is prevented by the orifice jet in the burner tube, it is difficult to prevent agglomeration during the entrainment from the side openings which ultimately stop the dust entrainment. At end of the study, a motorized cylindrical metal mesh was installed vertically, annularly around the burner tube inside the cone feeder. Four fan-like blades were installed horizontally on the mesh. This provided sufficient agitation to the dust particles and improved the dust entrainment. A better dust feeding system can be investigated in the future if smaller particles (<25  $\mu$ m) need to be studied further.

It is found that the measured burning velocity of pure gas flame where the equivalence ratio is close to 1 is less than the published data by about 10%. It is believed that the cause of such discrepancy is the velocity profile at burner opening which is not uniform. A Mach Hebra nozzle can be used as an improvement for the existing burner.

As discussed in Chapter 3, the current study is valid for laminar flames. Therefore the study on the interaction of coal particles with turbulent flame is the logical next step. Moreover, such study is directly related to dust explosions in industry since the deflagration is turbulent in nature.

# **Appendices**

## **Appendix A: Experimental Apparatus Details**

This section of appendix is supplemental information for the Experimental Apparatus section in the paper titled as *Naturally Entraining Solid Particle Injector*, and paper titled as *Study of Interaction of Entrained Coal Dust Particles in Lean Methane – Air Premixed Flames*.

#### A.1 Experimental Equipment

In this section, each experiment is presented and a short description each equipment is also provided

## I. Sieving Dust

The coal dusts are the ones used for power generation and are sieved to different sizes for experimental purposes. *Retsch* AS300 Sieve Shaker was used. This machine is able to sieve the dust particles with multiple different sizes (particle diameter): 0-25µm, 25-32µm, 32-38µm, 38-45µm, 45-53µm, 53-63µm, 63-75µm, 75-90µm, 90-106µm, and 106-125µm. Besides coal dust, sand was also sieved at size of 75-90µm for a control experiment set as mentioned in the paper. It is important to maintain the purity while sieving different materials at each time. Therefore, whenever a different material will be sieved, all sieving steel pans are cleaned by the ultrasonic cleaner prior to sieving. it is worth being mentioned that the sieving should be ran for multiple times (at least 3 times) for one batch of dust. It is because that some small particles tend to stay in the pans with larger sieving size since some sieving openings are blocked by the larger particles and the sieving time is not sufficient. In order to fix this problem after each sieving period (maximum setting for the machine is 99 min), paper towel can be used to gently wipe the bottom side of the sieving pan and then start another sieving period. All sieved particles are kept in different containers according to their sizes for future uses.



Fig. A.1-1: Sieve Shaker and Ultrasonic Cleaner

# II. Novel Dust Injector



Fig.A.1-2: Middle Sections of the Novel Dust Injector



Fig.A.1-3: Dust Injector

## III. Load Cell



Fig. A-1.3: Cole-Parmer Symmetry PR 4200 with Precision of 0.01g

The load cell has a capacity of 4200g with precision of 0.01g. A RS-232 interface is built-in this device. The RS-232 connection is connect to computer by a RS-232 to USB cable. The load cell transfers the instantaneous weight every 0.2 sec to the computer. The data is recorded by a free computer software called *Terminal 1.9* which can be obtained online.

#### IV. Mass Flow Controller



Fig. A-1-4: Sierra Smark Trak 2 Model 100 Mass Flow Controller with Manual Control Panel

Two mass flower controllers were used in the tests: 1. a Methane Mass Flow Controller with capacity up to 500 SCCM with precision of 5SCCM; 2. an Air Mass Flow Controller with capacity up to 5 SLPM with precision of 0.05 SLPM. The flow rate is controlled by the manual control panel. During the experiment, it is found that the two Mass Flow Controllers need to be re-zeroed if it has not been used for a while. It should be noted that this equipment is very sensitive the in solid impurities in the gas. Therefore, a filter was installed for each controller. This action is recommended by the manufacture. Furthermore, the device should be set to "Close" position and unplugged when it was not in use otherwise it will heat up and damage may be made to the controllers.

# V. Dust Burner Assembly

The dust burner is the dust injector with few minor modifications: 1. slot side openings that can be partially or totally covered by the brass jacket tube; 2. the thickness of the burner tube is reduced to 1mm for a better flame stability, and minimize the friction between tube wall and dust at the openings. Fig. A-1.5 and A-1.6 illustrate the details of the burner.



Fig. A-1.5: Burner with 60° Cone Feeder

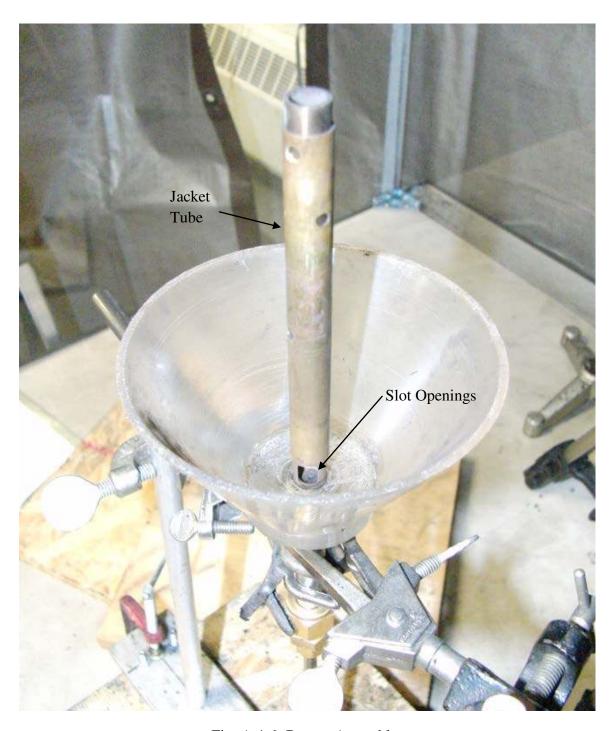


Fig. A-1.6: Burner Assembly

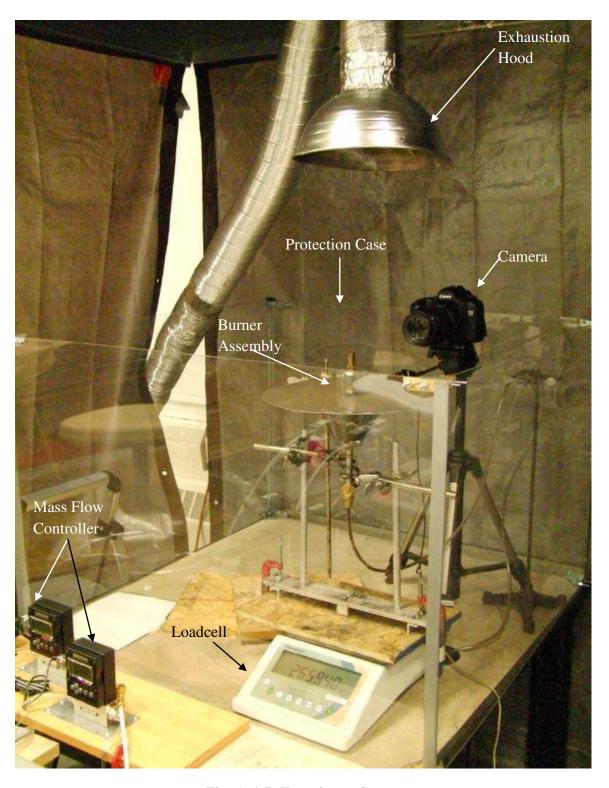


Fig. A-1.7: Experiment Setup

Fig. A-1.7 is the image shows the experiment setup (exclude the light source and double convex lens for shadowgraph but they are presented in nest section). As shown, a transparent protection case is built and incased the apparatus. This is used to minimize

the draft that can impact the flame. Furthermore, an addition exhaustion hood is installed on the top of the burner which runs at a much lower flow rate compare to the lab hood. This hood is powered by the fan and duct work is made that all smoke is exhausted to a enclosed hood in the lab. The small hood was proved to be sufficient to extract the smoke produced during each tests.

#### VI. Shadowgraph Set-up

The Shadowgraph (also called as Shadowgrams) is a fairly simple and less expensive visualization technique as compared to a similar technique called Schlieren photography. The details of Shadowgraph and Schlieren photography can be found in Schlieren and Shadowgraph Techniques: Visualizing Phenomena in Transparent Media by G.S. Settles. In short words, the Schlieren images are based on the first derivative of density while it is the second derivative of density for Shadowgraph. It should be noted that Shadowgraph used in in this study is Direct Shadowgraph in parallel light. This method essentially eliminate the distortion/magnification of shadowgraph with diverge light. The locations of visualized flame cone using these techniques are different from the visible flame cone as illustrated in the figure below. However, these techniques do not make a difference in flame cone angle.

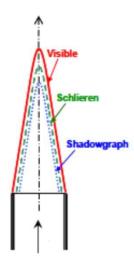


Fig. A-1.8: Location of Visible Flame, Schlieren Image, and Shadowgraph in a Flame Cone

In this study, a simple shadowgraph set-up was constructed using the project lamp assembly from an old-fashion projector, a double convex lens with focus length of 20cm, and Canon SLR camera with a macro-lens as shown in the figure below. The light from the project lamp (420W) is converted into a point light source by covering the lamp with a steel plate with a pin-hole. Extra caution needs to be taken since the lamp produces extensive heat that will turn the steel plate into red-hot. It is recommended that the lamp

should not be kept on for more than 15min and cool-down time of around 5min is needed for each run.

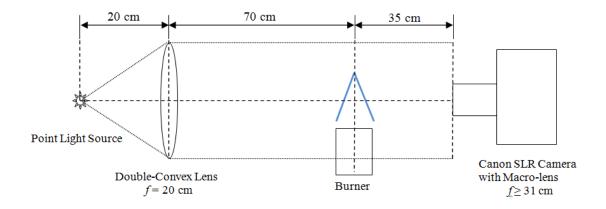


Fig. A-1.9: Shadowgraph Setup

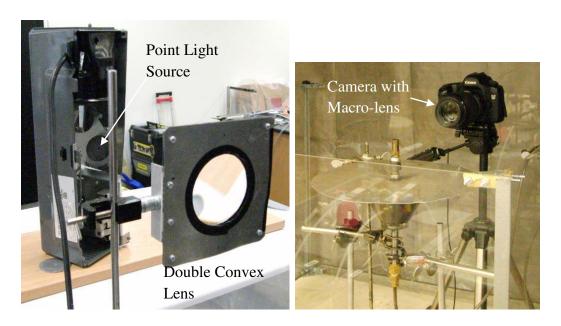


Fig. A-1.9: Shadowgraph Apparatus

## A.2 Procedures of Conducting Experiment for Study

The experiment is conducted as follows:

- 1. Check the apparatus and fix any abnormal conditions.
- 2. Place the burner assembly on the load cell.
- 3. Lower the jacket brass tube completely and thus close the side openings

- 4. Fill the cone feeder with coal dust with desired size.
- 5. Install the dust collection pan and ensure that it is free of contact from the burner.
- 6. Place camera at the right position and take sample picture with lamp on and without flame to ensure the position of burner, light source and camera are on a same axis.
  - Note: Camera was configured as all manual where the focus is at the maximum (to the nearest point), shutter speed at 4000, and aperture is maximized at 2.8.
- 7. Set up the load cell connection, and camera remote shooting connection with a PC.
- 8. Turn on the exhaust hood.
- 9. Open the valve and adjust the pressure of methane and air pressurized gas cylinder.
- 10. Plug in the power chord for the Mass Flow Controller (MFC).
- 11. Ensure that the MFC is at *Closed* mode to prevent unwanted gas leak or accumulation
- 12. Turn on the valves at gas distribution piping systems for both air and methane.
- 13. Adjust the flow rate to the desired value at the manual control panel of MFC.
- 14. Set the MFC mod at *Automatic* for both air and methane at the same time.
- 15. Ignite the flame with a torch.
- 16. Adjust the jacket tube to enable dust injection at a desired level.
- 17. Start the load cell recording.
- 18. Take Shadowgraph pictures using remote control.
- 19. Stop the flow by change the MFC mod to *Closed* after 5min or more if more pictures need to be taken. For high unstable flame with very high dust concentration, stop the flow once the flame is extinguished.
- 20. Close the side opening completely.
- 21. Close the main valves at the gas cylinders.
- 22. Set MFC mod as *Purge* at the same time and ignite the burner.
- 23. Shut off all valves once all gases have been bled out.
- 24. Set MFC mod as *Close* and unplug the MFC.
- 25. Turn off the exhaust hood.

## **Appendix B: Algorithm for Angle Measurement**

### **B.1 Description of the Algorithm using Matlab**

The Matlab subscript has been developed for the flame cone measurements. There are two parts of this code: 1. The main code handles the image cropping, edge detection, angle measuring, and finally image saving; 2. The automatic supplement code enable Matlab automatically process multiple pictures.

The concepts detecting the cone edge and effectively make the angle measurement are the most important part of this code.

- 1. Edge Detection: First each image has been converted into binary image with enhanced contrast. And each image has been cropped with around 420 pixels on the x-axis for the sake of processing number of images. Then each row within the scan boundary which is manual defined is scanned to find out most dramatic change of the pixel value change (0 for black and 1 for white) for both sides of the cone, but processed separately. This method can successfully detect the edges of the cone. However, it will also pick up areas other than on the edge since the maximum change of pixel value may also occurs at the burning particles. Nevertheless, the majority of identified pixels are on the cone edge.
- 2. Effective Angle Measurement: Many other methods have been tried to make the most effective or accurate angle measurement. However, the one present here is the most effective one. As mentioned in last paragraph, the edge detection is essentially picking up the pixel or point on the image where the maximum value changes occurs. Therefore, a line fit is a best method to get the angle based on these detected points. However, it is problematic to get a line fit with many outliers as the detected pixels. Therefore, the slope of a line between each detected point has been measured and the media of all the values of the slopes is chosen to be the slope of the line for line fit. It should be not that at the right side of the cone, slopes should be positive and thus all slopes with a negative sign are excluded. The same process was taken for the left side of the cone. Once the slope of the line for line fit is determined, a line-fit is drawn on the image to enable to make the final check. If the line-fit does not match the cone edge, this image or data is excluded. However, such line-fit very rarely occurs.

The actual code can be reviewed in section B.2.

#### **B.2** Developed Algorithm (Matlab Code)

## I. Main Algorithm - Angle\_Function\_02

```
function [Angle] = angle_function_02(file_drill) %% start operating on folder
file=strcat(file_drill); % define image file
im1 = imread(file); % read image file
% bc1 = imread(file); % read image file
bc1=im1(:,:,3); % change to blue channel only
% bc1=im1(:,:,:); % change to blue channel only
imgcrop1=imcrop(bc1,[1370.5 1138.5 1213 750]); % crop image file MANUALLY
A=histeq(imgcrop1); % increase contrast of image
I=imresize(A,[259.7,420.01]); % reduce resolution of image MANUALLY, 1/4 of the
original; NOTE: [number of rows(verticle pix number), number of columns(horizontal pix
number)
[row,column] = size(I); % find size of image % setting edge scan area in y direction; And
moving step in y direction
y_max = 200
y_min = 110
del_y = 1; % setting edge scan areas for left and right cone edges
x_min_left = 53;
x_max_left = 200;
x_min_right = 220;
x_max_right = 367; % other edge scan constant
test_{loop} = 1;
aa = 1;
y=0;
x1_store=0;
k_store = 0;% edge detection
y(1) = y_min;
while y(end) \le y_max
test_{loop} = test_{loop} + 1;
% Left side bottom line
i = y(aa);
                     % v1 value
count 01 = 1;
for k = x_min_left:x_max_left;
                                  % limiting x to the center of the cone (214)
k_store(count_01) = k;
P(k) = I(i,k);
count_01 = count_01+1;
end;
```

```
[v1,x1] = max(P);
x1_store(aa) = x1;
P store(y(aa),:) = P;
PP = double(P); %smooth PP
for hh=x_min_left+6:(x_max_left-6);
PP\_smooth(hh) = mean(PP(hh-5:hh+5));
end
ee = 1;
for dd = x_min_left:x_max_left-10;
dd store(dd) = dd;
del_P(dd) = PP_smooth(dd+1) - PP_smooth(dd);
ee=ee+1;
end
[test(aa) max\_del\_P(aa)] = max(del\_P(x\_min\_left+10:x\_max\_left-10));
max_del_P(aa) = max_del_P(aa) + x_min_left + 10; % offset
%% Right Side Bottom Line
count_02 = 1;
for j = x_min_right:x_max_right;
i_store(count_02) = i;
Q(j) = I(i,j);
count 02 = count \ 02 + 1;
end;
[v3,x3] = max(Q);
x3_store(aa) = x3;
Q_{store}(y(aa),:) = Q;
QQ = double(Q);
%smooth QQ
for ll=x_min_right+6:(x_max_right-6);
QQ_{smooth(ll)} = mean(QQ(ll-5:ll+5));
end
mm = 1;
for nn = x_min_right:x_max_right-10;
nn_store(mm) = nn;
del_Q(nn) = QQ_smooth(nn+1) - QQ_smooth(nn);
mm=mm+1;
end
[test(aa) min_del_Q(aa)] = min(del_Q(x_min_right+10:x_max_right-10));
min_del_Q(aa) = min_del_Q(aa) + x_min_right + 10; %offset
y(aa+1) = y(aa) + del_y;
aa = aa + 1;
```

```
end
ff = 1;
for gg = min(x3\_store):max(x3\_store)-1;
gg\_store(ff) = gg;
del_Q(gg) = Q(gg)-Q(gg+1);
ff=ff+1;
end
figure
clf
imshow(I) % show image
hold on
%Slopes calculation and selection
% % compute all slopes on left side
totpoints_l=y_max-y_min;
numslopes_l=(totpoints_l^2-totpoints_l)/2;
slopes_l=zeros(numslopes_l,1);
T=0;
for i=1:totpoints_1
for j=i+1:totpoints_l;
T=T+1;
slopes_l(T)=(y(i)-y(j))/(max_del_P(i)-max_del_P(j));
end
end
%taking only negative slope for left side
slopes_l(slopes_l>0) = [];
% find median of slopes
slope left = median(slopes 1)
% % compute all slopes on right side
totpoints_r=y_max-y_min;
numslopes_r=(totpoints_r^2-totpoints_r)/2;
slopes_r=zeros(numslopes_r,1);
U=0;
for i=1:totpoints_r
for j=i+1:totpoints_r;
U=U+1;
slopes_r(U)=(y(i)-y(j))/(min_del_Q(i)-min_del_Q(j));
end
end
%taking only positive slope for right side
```

```
slopes_r(slopes_r<0) = [];
% find median of slopes
slope right=median(slopes r)
%% plot the slope and detected points
% plot original data with best guess at slope
plot(max_del_P,y(1:length(max_del_P)),'*',sort(max_del_P),slope_left*sort(max_del_P)
+(((y_max+y_min)/2-median(slope_left*sort(max_del_P)))),'g-')
% plot original data with best guess at slope
plot(min_del_Q,y(1:length(min_del_Q)),'+',sort(min_del_Q),slope_right*sort(min_del_Q)
)-(median(slope_right*sort(min_del_Q))-((y_max+y_min)/2)),'r-')
%Calulating Angle based on slope
Angle_a = 90 - (-180/pi*atan(slope_left));
Angle_b = 90 - (180/pi*atan(slope_right));
Angle = (Angle_a + Angle_b)/2;
Angle
hold off
axis on
pause(0.5)
II.
       Automatic Process Algorithm
```

clear all close all clc

dname = ('G:\Combustion Lab\experiment\Burning velocity\images\gas flames with low flow rate\phi=0.7--1.1\_0.081\selected');%Default Directory To be Opened

```
%% Set up basic file name path to read
```

top\_file = [dname '\'] ; %Set up main database to

open and look inside

ls\_top\_file = ls(top\_file) ; %List Files inside main

folder

c = cellstr(ls\_top\_file) ; %Turn cells from ls

function into strings

```
= c(3:length(c))
cc_drill
                                                                 ; %Set up a matrix
without the . and .. produces by the ls function
S
                          = size(cc_drill)
                                                               ; %Find the size of matrix
containing names of files inside of main database
                                                       ; %This counter is set to 3 to
account for the . and .. at the beggining of each matrix created by ls
while a \le S(1)
close all
file drill
                         = char(cellstr([top_file char(cc_drill(a))])); %File to be
operated on
                         = char(cc_drill(a))
data_n
                           = char(cc_drill(a))
file_name
% Operations on files in folder
[Angle(a)] = angle_function_02(file_drill);
set(gcf, 'PaperPositionMode', 'auto');
h = gcf;
saveas(h, [dname '\' 'z_' file_name ], 'jpg');
                       = a+1
a
```

end

# **Appendix C: Experimental Data**

# C.1 Pressure Drop Calculation for Dust Injector

Table C.1-1 (calculation is based on *Introduction to Fluid Mechanics* (2003), by Robert W. Fox et al.)

D=	12	12	12	12	12	12	mm
	0.012	0.012	0.012	0.012	0.012	0.012	m
dt(orifice dia.)=	1	1	1	1	1	1	mm
	0.001	0.001	0.001	0.001	0.001	0.001	m
m(mass flow rate)=	1000	1500	2000	2500	3000	3500	cc/min
	1.13E-05	0.000017	2.27E-05	2.83E-05	0.000034	3.97E-05	kg/s
miu(viscosity)=	1.03E-05	1.03E-05	1.03E-05	1.03E-05	1.03E-05	1.03E-05	kg/m.s
roh(density)=	0.68	0.68	0.68	0.68	0.68	0.68	kg/m^3
A=	0.000113	0.000113	0.000113	0.000113	0.000113	0.000113	m^2
At(orifice area)=	7.85E-07	7.85E-07	7.85E-07	7.85E-07	7.85E-07	7.85E-07	m^2
Re=	117.089	175.6335	234.178	292.7225	351.267	409.8115	
beta(=At/A)=	0.006944	0.006944	0.006944	0.006944	0.006944	0.006944	
C(discharge coeff.)=	0.595911	0.595909	0.595907	0.595906	0.595905	0.595905	corner tap config.
K(flow ceoff.)=	0.595911	0.595909	0.595907	0.595906	0.595905	0.595905	
V=	0.147366	0.221049	0.294731	0.368414	0.442097	0.51578	m/s
Vt=	21.22066	31.83099	42.44132	53.05165	63.66198	74.27231	m/s
p1-p2=	431.155	970.1077	1724.644	2694.765	3880.471	5281.761	Pa
	0.004312	0.009701	0.017246	0.026948	0.038805	0.052818	bar
D=	0.012	0.012	0.012	0.012	0.012	0.012	m
	1	1	1	1	1	1	mm
dt(orifice dia.)=	0.001	0.001	0.001	0.001	0.001	0.001	m
	4000	4500	5000	5500	6000	6500	cc/min
m(mass flow rate)=	4.53E-05	0.000051	5.67E-05	6.23E-05	0.000068	7.37E-05	kg/s
	1.03E-05	1.03E-05	1.03E-05	1.03E-05	1.03E-05	1.03E-05	kg/m.s
miu(viscosity)=	0.68	0.68	0.68	0.68	0.68	0.68	kg/m^3
roh(density)=	0.000113	0.000113	0.000113	0.000113	0.000113	0.000113	m^2
A=	7.85E-07	7.85E-07	7.85E-07	7.85E-07	7.85E-07	7.85E-07	m^2
At(orifice area)=	468.356	526.9005	585.445	643.9895	702.534	761.0785	
Re=	0.006944	0.006944	0.006944	0.006944	0.006944	0.006944	
beta(=At/A)=	0.595905	0.595904	0.595904	0.595904	0.595904	0.595903	corner tap config.
C(discharge coeff.)=	0.595905	0.595904	0.595904	0.595904	0.595904	0.595903	
K(flow ceoff.)=	0.589463	0.663146	0.736828	0.810511	0.884194	0.957877	m/s
V=	84.88264	95.49297	106.1033	116.7136	127.324	137.9343	m/s
Vt=	6898.635	8731.095	10779.14	13042.77	15521.98	18216.78	Pa
p1-p2=	0.068986	0.087311	0.107791	0.130428	0.15522	0.182168	bar

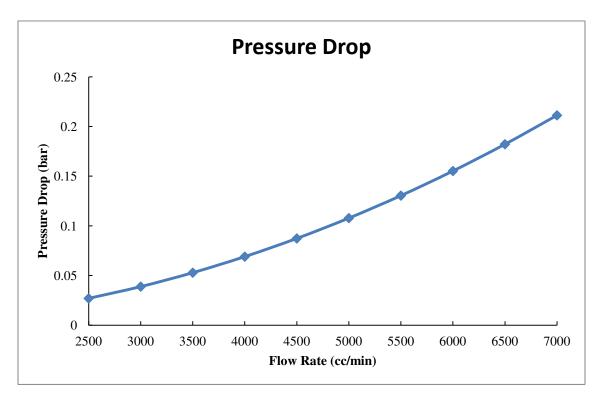


Fig. C.1-1

# C.2 Dust Entrainment Data for Dust Injector

Table C.2-1

Three 2.70 cm <sup>2</sup> Openings						
Flow Rate (L/min)	Averaged Entrainment Rate (g/min)					
1	0					
1.5	0.634					
2	2.118					
3	8.08					
4	16.912					
5	28.34					
6	41.874					
7	58.558					
8	84.964					
9	113.744					
10	154.126					

Table C.2-2

Three 1.51 cm <sup>2</sup> Openings						
Flow Rate (L/min)	Averaged Entrainment Rate (g/min)					
1	0					
1.5	0.566					
2	1.564					
3	6.124					
4	13.42					
5	24.576					
6	37.8					
7	58.73					
8	85.746					
9	118.222					
10	150.14					

Table C.2-3

Six 1.18 cm <sup>2</sup> Openings					
Flow Rate (L/min)	Averaged Entrainment Rate (g/min)				
1.5	0				
2	1.116				
3	4.47				
4	9.744				
5	18.822				
6	37.534				
7	58.686				
8	82.67				
9	122.62				
10	152.846				

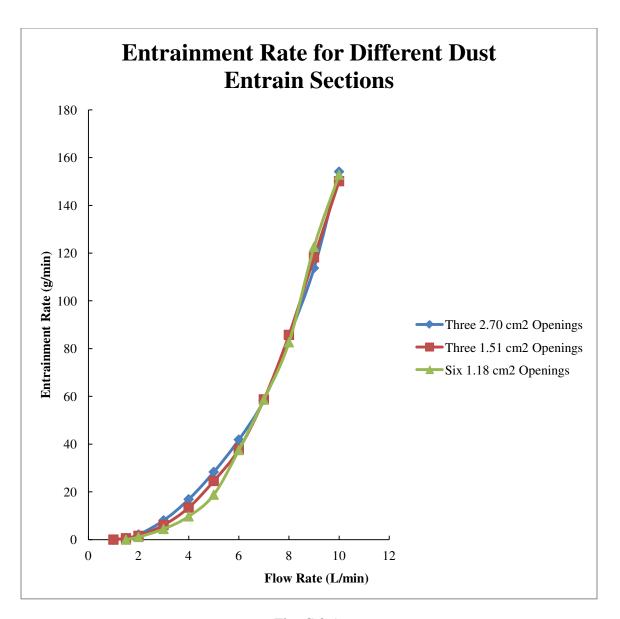


Fig. C.2-1

### **C.3** Summary Sheet of Flame Cone Angle for Experiment.

#### 1. Dust Free Flame

1.1  $\phi = 0.75$  (Flow Rate: Air 2.0 SLPM; Methane: 0.158 SLPM)

Average Cone Angle: 46.386

1.2  $\phi$ =0.80(Flow Rate: Air 2.5 SLPM; Methane: 0.21 SLPM)

Average Cone Angle: 42.344

1.3  $\phi$ =0.85(Flow Rate: Air 2.5 SLPM; Methane: 0.223 SLPM)

Average Cone Angle: 50.351

### 2. Sand at 75-90 μm

**2.1**  $\phi$ =**0.80**(Flow Rate: Air 2.5 SLPM; Methane: 0.21 SLPM)

Experiment NO. 1: Dust Concentration: 48.74; Average Cone Angle: 40.272

Experiment NO. 2: Dust Concentration: 64.24; Average Cone Angle: 39.586

Experiment NO. 3: Dust Concentration: 178.28; Average Cone Angle: 37.488

2.2  $\phi$ =0.85(Flow Rate: Air 2.5 SLPM; Methane: 0.223 SLPM)

Experiment NO. 1: Dust Concentration: 66.14; Average Cone Angle: 45.379

Experiment NO. 3: Dust Concentration: 48.60; Average Cone Angle: 44.583

Experiment NO. 4: Dust Concentration: 48.50; Average Cone Angle: 45.179

Experiment NO. 5: Dust Concentration: 156.54; Average Cone Angle: 40.701

## 3. Coal Particles at 75-90 µm

3.1  $\phi$ =0.75(Flow Rate: Air 2.0 SLPM; Methane: 0.158 SLPM)

Experiment NO. 1: Dust Concentration: 108.50; Average Cone Angle: 43.287 Experiment NO. 2: Dust Concentration: 11.07; Average Cone Angle: 44.333 Experiment NO. 3: Dust Concentration: 161.36; Average Cone Angle: 41.212 Experiment NO. 4: Dust Concentration: 169.71; Average Cone Angle: 39.764 Experiment NO. 5: Dust Concentration: 153.01; Average Cone Angle: 43.222 Experiment NO. 6: Dust Concentration: 13.91; Average Cone Angle: 44.960 Experiment NO. 7: Dust Concentration: 197.53; Average Cone Angle: 41.862 Experiment NO. 8: Dust Concentration: 61.20; Average Cone Angle: 45.907 Experiment NO. 9: Dust Concentration: 30.60; Average Cone Angle: 44.945

## **2.2** $\phi$ **=0.80**(Flow Rate: Air 2.5 SLPM; Methane: 0.21 SLPM)

Experiment NO. 1: Dust Concentration: 26.58; Average Cone Angle: 41.532

Experiment NO. 2: Dust Concentration: 70.89; Average Cone Angle: 39.156

Experiment NO. 4: Dust Concentration: 248.13; Average Cone Angle: 38.250

Experiment NO. 5: Dust Concentration: 24.371; Average Cone Angle: 41.900

Experiment NO. 6: Dust Concentration: 146.22; Average Cone Angle: 39.781

Experiment NO. 8: Dust Concentration: 254.77; Average Cone Angle: 38.848

Experiment NO. 11: Dust Concentration: 128.49; Average Cone Angle: 40.314

Experiment NO. 12: Dust Concentration: 112.98; Average Cone Angle: 40.873

Experiment NO. 13: Dust Concentration: 28.801; Average Cone Angle: 41.021

Experiment NO. 14: Dust Concentration: 137.35; Average Cone Angle: 40.165

Experiment NO. 15: Dust Concentration: 46.52; Average Cone Angle: 41.495

#### **2.3** $\phi$ =**0.85**(Flow Rate: Air 2.5 SLPM; Methane: 0.223 SLPM)

Experiment NO. 1: Dust Concentration: 28.66; Average Cone Angle: 46.909

Experiment NO. 2: Dust Concentration: 6.614; Average Cone Angle: 49.024

Experiment NO. 4: Dust Concentration: 50.71; Average Cone Angle: 47.622

Experiment NO. 5: Dust Concentration: 180.80; Average Cone Angle: 45.595

Experiment NO. 6: Dust Concentration: 24.25; Average Cone Angle: 48.530

Experiment NO. 7: Dust Concentration: 81.58; Average Cone Angle: 47.418

Experiment NO. 8: Dust Concentration: 302.06; Average Cone Angle: 43.973

Experiment NO. 9: Dust Concentration: 94.816; Average Cone Angle: 46.714

Experiment NO. 10: Dust Concentration: 6.61; Average Cone Angle: 48.7619

Experiment NO. 11: Dust Concentration: 63.94; Average Cone Angle: 47.001

## 4. Coal Particles at 53-63 μm

## **4.1** $\phi$ =**0.80**(Flow Rate: Air 2.5 SLPM; Methane: 0.21 SLPM)

Experiment NO. 3: Dust Concentration: 57.60; Average Cone Angle: 40.953

Experiment NO. 5: Dust Concentration: 181.66: Average Cone Angle: 39.308

Experiment NO. 6: Dust Concentration: 203.82; Average Cone Angle: 39.161

Experiment NO. 8: Dust Concentration: 73.11; Average Cone Angle: 40.641

## **4.2** $\phi$ =**0.85** (Flow Rate: Air 2.5 SLPM; Methane: 0.223 SLPM)

Experiment NO. 3: Dust Concentration: 61.73; Average Cone Angle 49.312

Experiment NO. 5: Dust Concentration: 59.53; Average Cone Angle: 48.0801

Experiment NO. 10: Dust Concentration: 79.37; Average Cone Angle: 47.543

Experiment NO. 11: Dust Concentration: 83.78; Average Cone Angle: 47.973

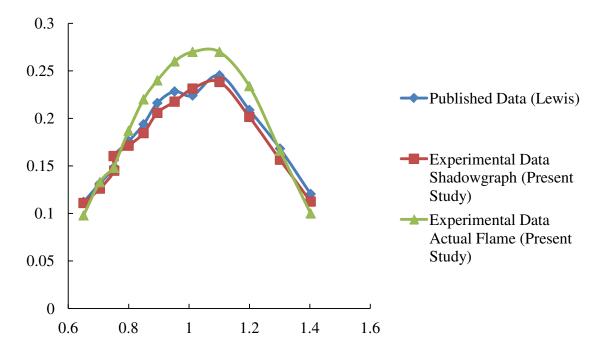
Experiment NO. 17: Dust Concentration: 249.15; Average Cone Angle: 44.883

#### 5. Coal Particles at 0-25 µm

### 5.1 $\phi$ =0.75(Flow Rate: Air 2.0 SLPM; Methane: 0.158 SLPM)

Experiment NO. 3: Dust Concentration: 292.12 g/m<sup>3</sup>; Average Cone Angle: 50.008

## 6. $S_u$ (gas only) with Equivalence Ratio Ranges from 0.7 to 1.4



The laminar burning velocity of premixed methane-air flame without dust is experimentally determined and compared with the published data which is presented in *Combustion, Flames and Explosions of Gases* (1987) by Bernard Lewis and Guenther von Elbe. The plot above concludes the results obtained in this set of tests. As shown in this plot, the burning velocity measured based on shadowgraph and actual flame image is very small. The differences between them are believed to be caused by errors that associated with the test. However, it is noted that the burning velocity where the equivalence ratio is close to 1 is lower than the published data. It is believed that the flow velocity profile is very uneven. It should be noted that the published data are selected from previous study that is mainly due to similar experimental set-up is used. However, experimental data in other studies which uses different experimental set-up may have different value than that presented here. The further action can be done related to this issue is discussed in the Conclusions and Future Work section.

## C. 4 Pictures and Entrainment Data

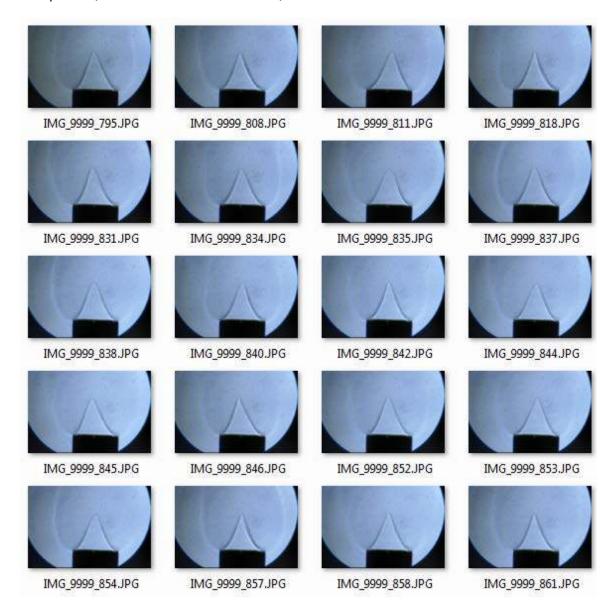
## 1. Dust Free

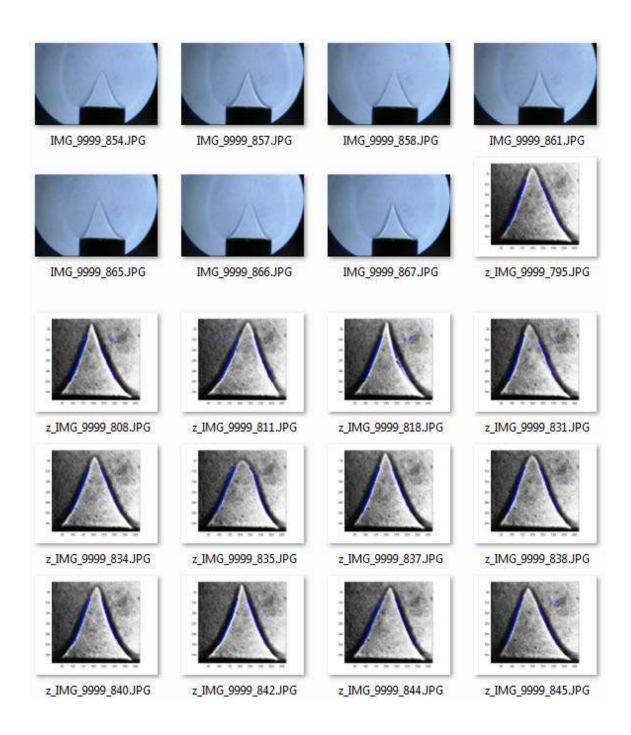
## 1.1 $\phi$ =0.75; Flow Rate: Air 2 SLPM; Methane 0.158 SLPM





# 1.2 $\phi$ =0.80; Flow Rate: Air 2.5 SLPM; Methane 0.21 SLPM







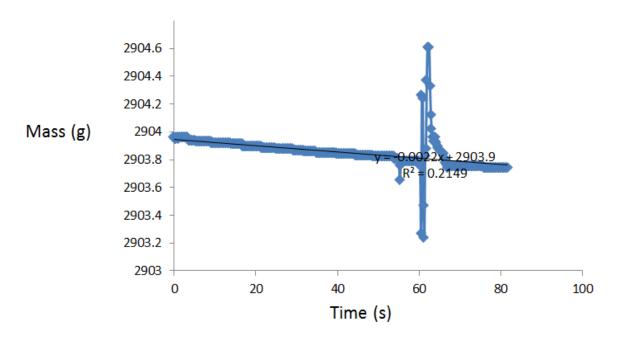
# 1.3 $\phi$ =0.85; Flow Rate: Air 2.5 SLPM; Methane 0.223 SLPM



# 2. Sand Particles (75-90 $\mu$ m)

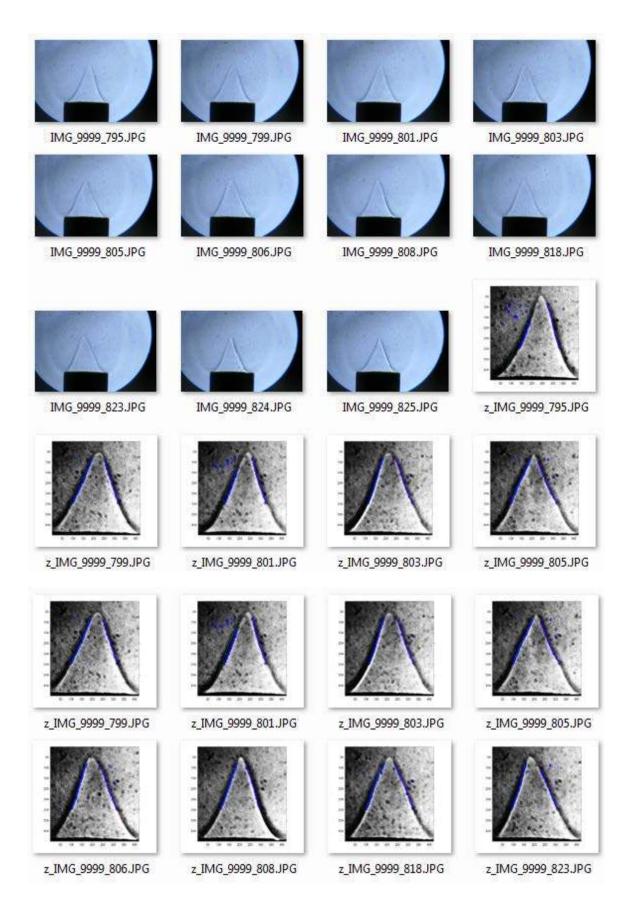
# 2.1 $\phi$ =0.80; Flow Rate: Air 2.5 SLPM; Methane 0.21 SLPM

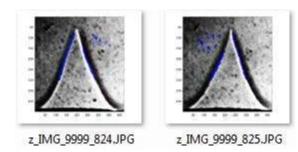
# Experiment NO. 1.



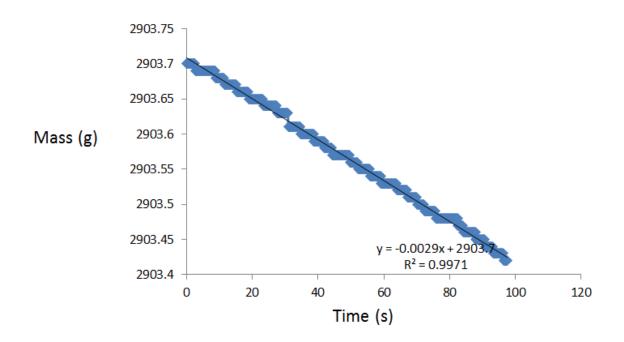
Air F	Flow Rate	Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.21	0.000145	4.51E-05	0.010744	0.497869	0.79968

Entrainment Rate 0.0022 g/sec Concentration 48.74 g/m³



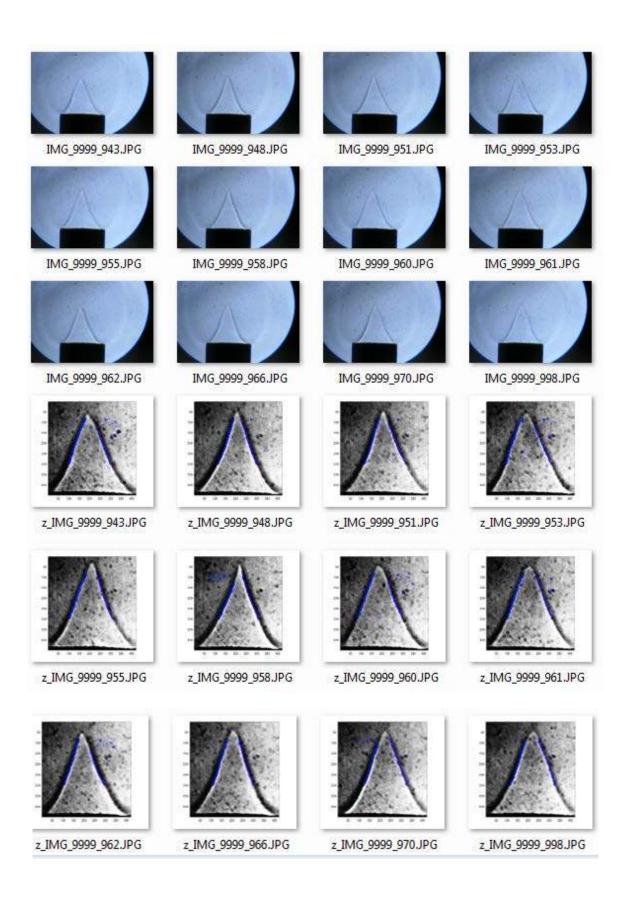


Experiment NO. 2.

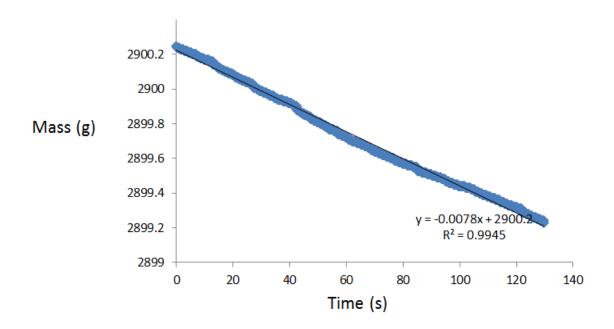


Air Flow Rate Fuel Flow rate		Volume flow rate	nozzle width	U	Phi		
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.21	0.000145	4.51E-05	0.010744	0.497869	0.79968

Entrainment Rate 0.0029 g/sec Concentration 64.24819 g/m³

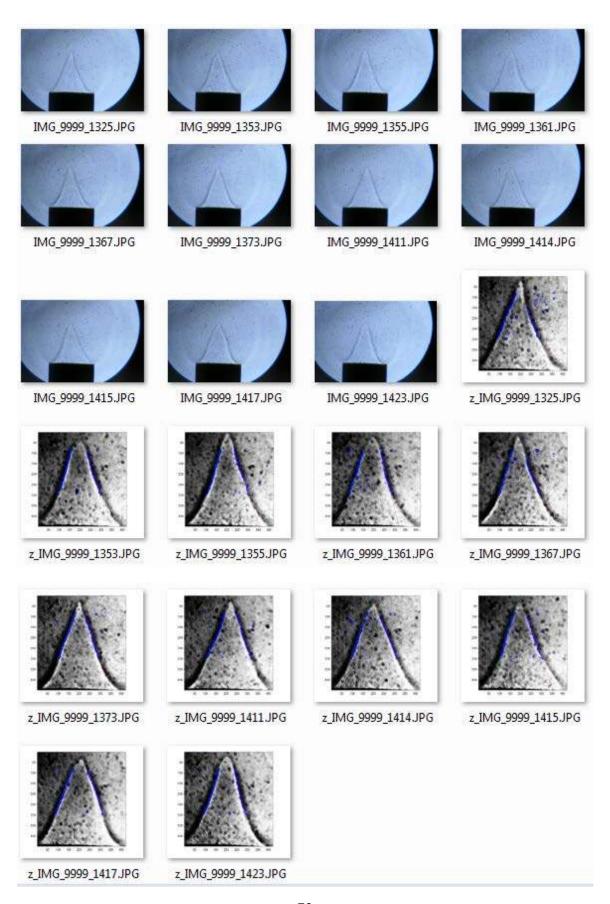


Experiment NO. 3.



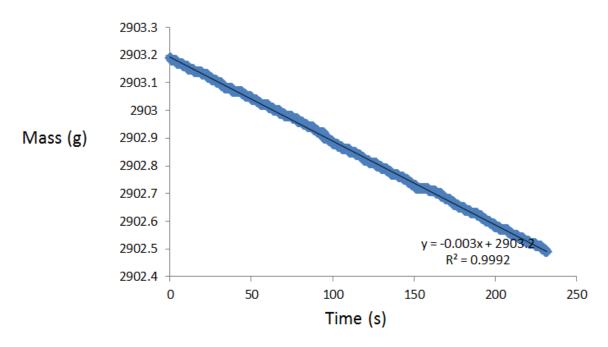
Air I	Flow Rate	ate Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.21	0.000145	4.51E-05	0.010744	0.497869	0.79968

Entrainment Rate 0.0078 g/sec Concentration 172.8055 g/m³



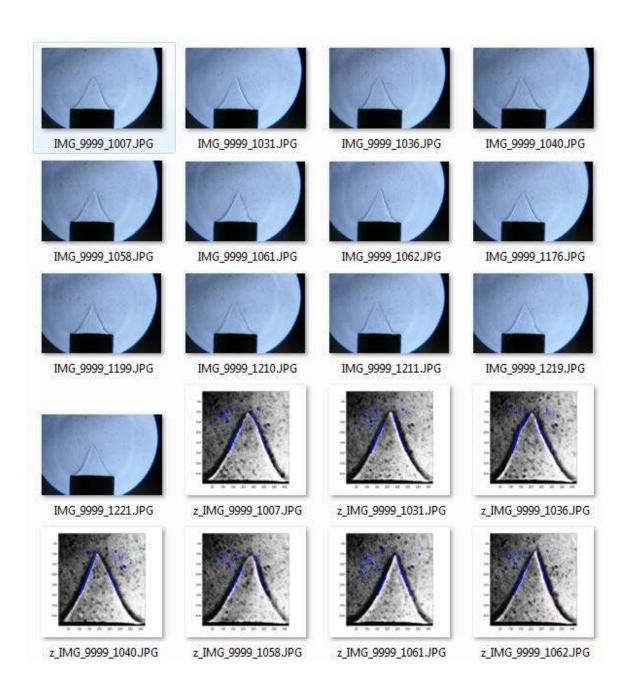
# 2.2 $\phi$ =0.85; Flow Rate: Air 2.5 SLPM; Methane 0.223 SLPM

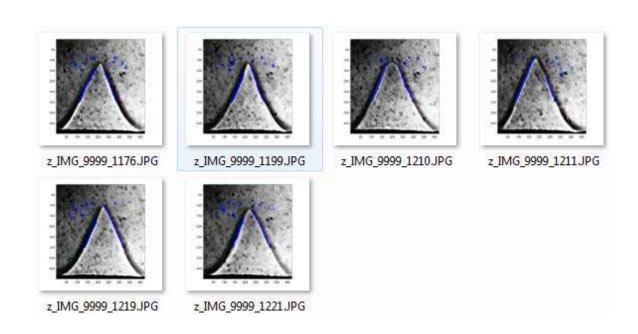
#### **Experiment NO. 1.**



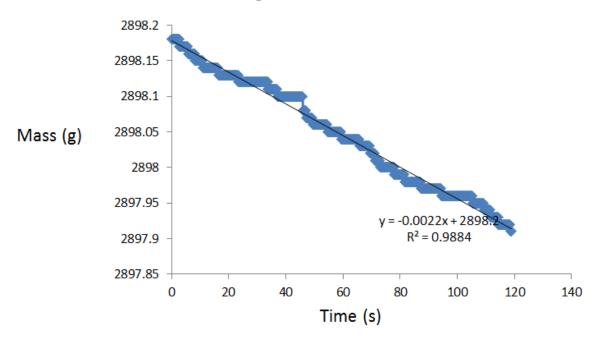
Air Flow Rate		Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.223	0.000155	4.54E-05	0.010744	0.500255	0.849184

Entrainment Rate 0.003 g/sec Concentration 66.14662 g/m<sup>3</sup>



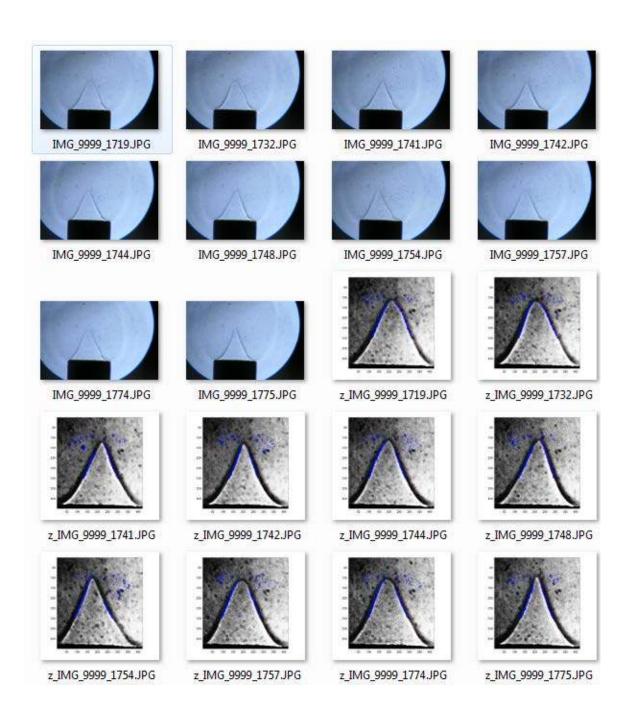


Experiment NO. 3.

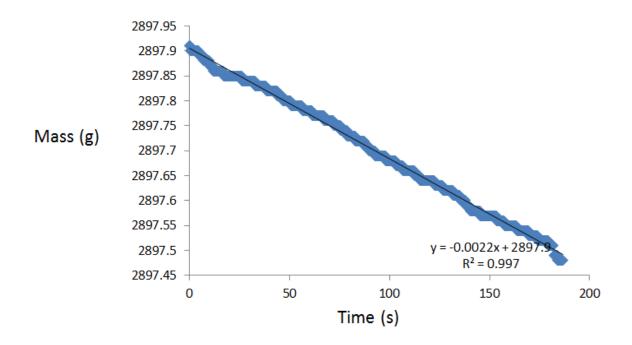


Air F	Air Flow Rate Fuel Flow rate		Volume flow rate	nozzle width	U	Phi	
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.223	0.000155	4.54E-05	0.010744	0.500255	0.849184

Entrainment Rate 0.0022 g/sec Concentration 48.50752 g/m<sup>3</sup>

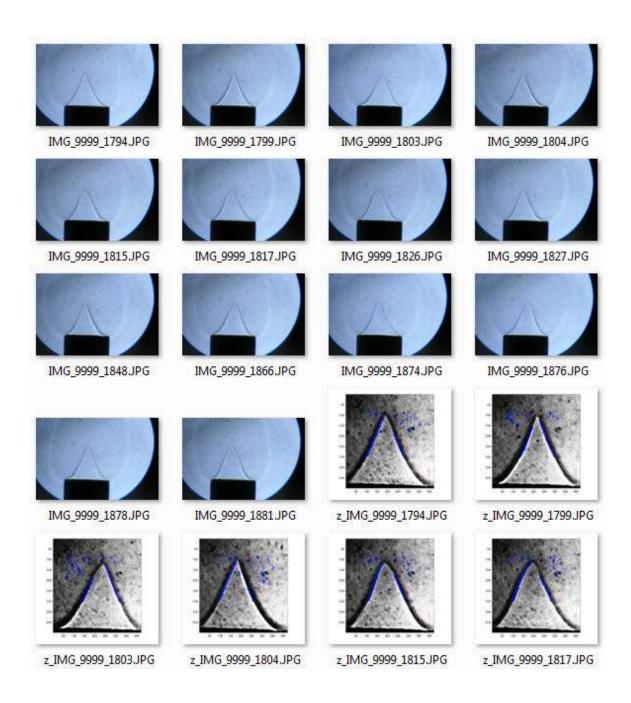


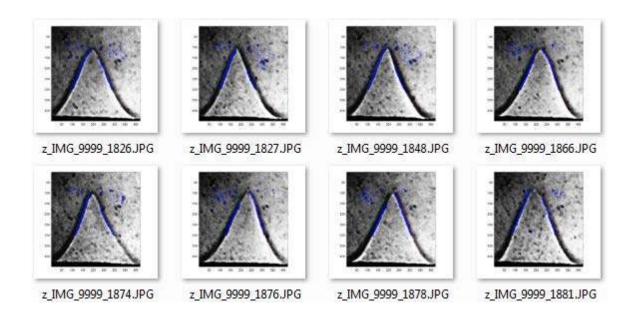
Experiment NO. 4.



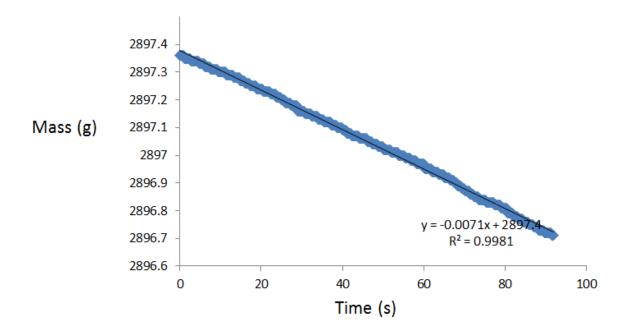
Air I	Air Flow Rate Fuel Flow rate		Volume flow rate	nozzle width	U	Phi	
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.223	0.000155	4.54E-05	0.010744	0.500255	0.849184

Entrainment Rate 0.0022 g/sec Concentration 48.50752 g/m³



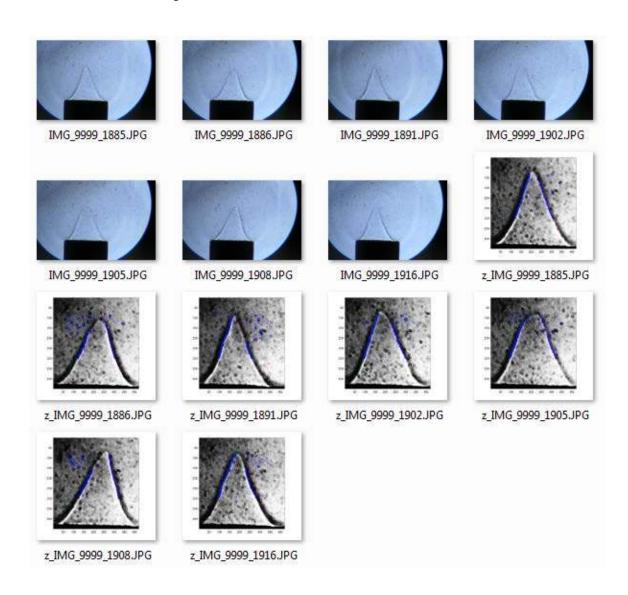


## Experiment NO. 5.



Air I	Air Flow Rate Fuel Flow rate		Volume flow rate	nozzle width	U	Phi	
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.223	0.000155	4.54E-05	0.010744	0.500255	0.849184

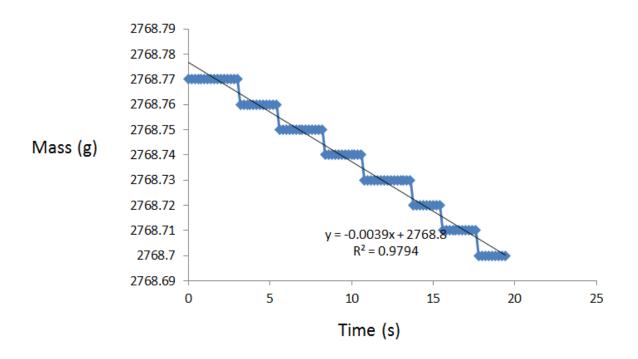
Entrainment Rate 0.0071 g/sec Concentration 156.547 g/m<sup>3</sup>



# 3. Coal Particles (75-90 $\mu$ m)

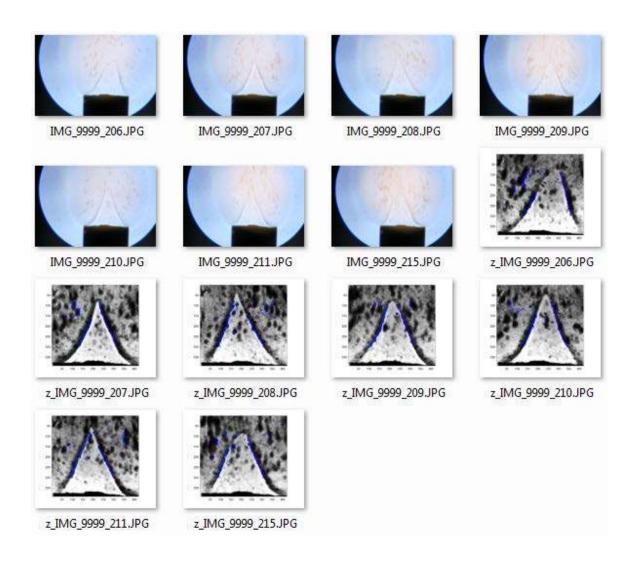
## 3.1 $\phi$ =0.75; Flow Rate: Air 2.0 SLPM; Methane 0.158 SLPM

## Experiment NO. 1.

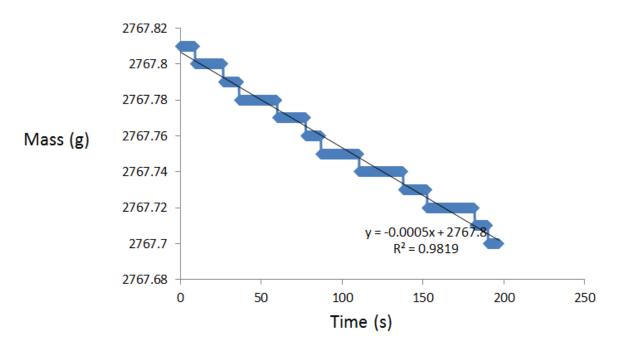


Air I	Flow Rate	Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2	0.001386	0.158	0.000109	3.59E- 05	0.010744	0.39646	0.75208

Entrainment Rate 0.0039 g/sec Concentration 108.5034 g/m<sup>3</sup>

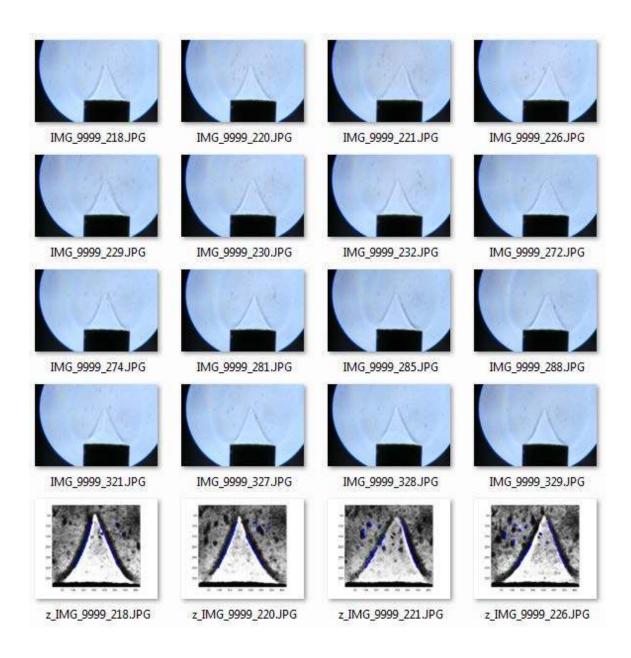


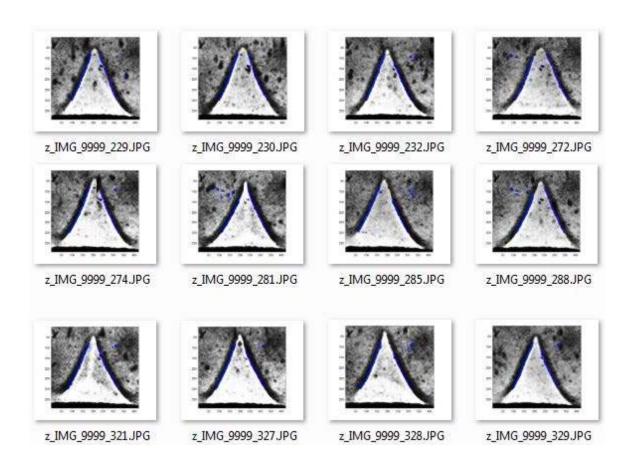
Experiment NO. 2



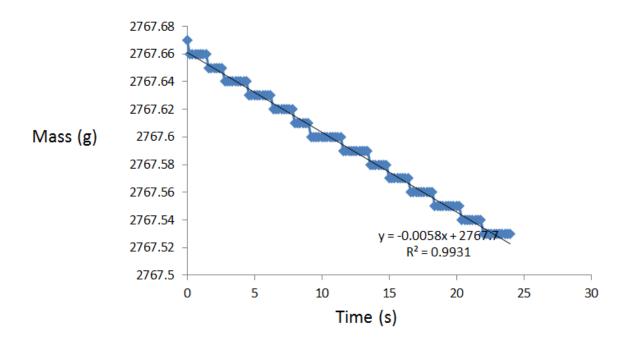
Air I	Air Flow Rate Fuel Flow rate		Volume flow rate	nozzle width	U	Phi	
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.21	0.000145	4.51E- 05	0.010744	0.497869	0.79968

Entrainment Rate 0.0005 g/sec Concentration 11.07727 g/m³



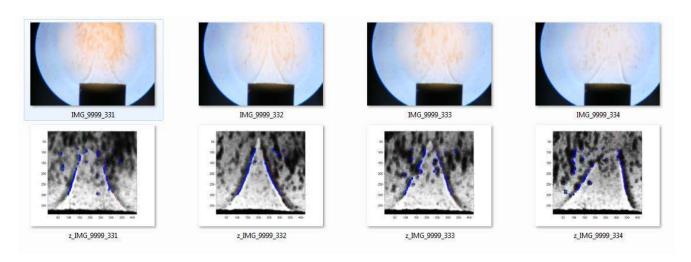


#### **Experiment NO. 3**

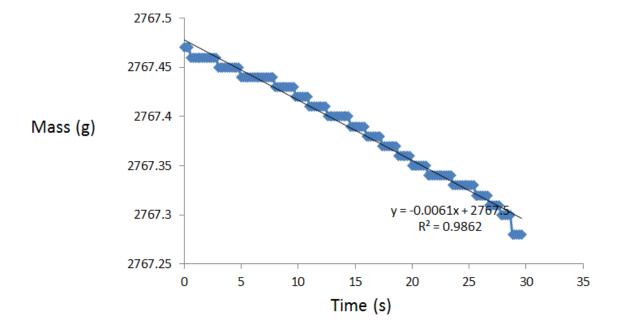


Air Flow Rate		Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2	0.001386	0.158	0.000109	3.59E-05	0.010744	0.39646	0.75208

Entrainment Rate 0.0058 g/sec Concentration 161.3641 g/m<sup>3</sup>

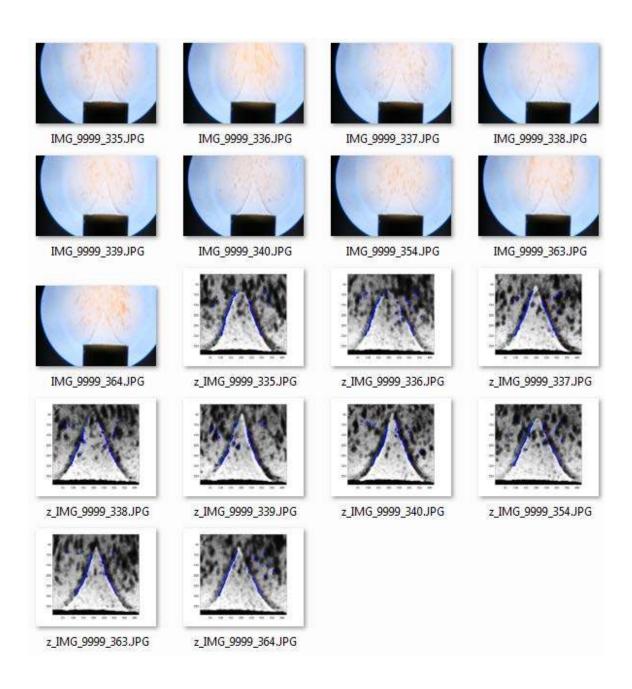


Experiment NO. 4.

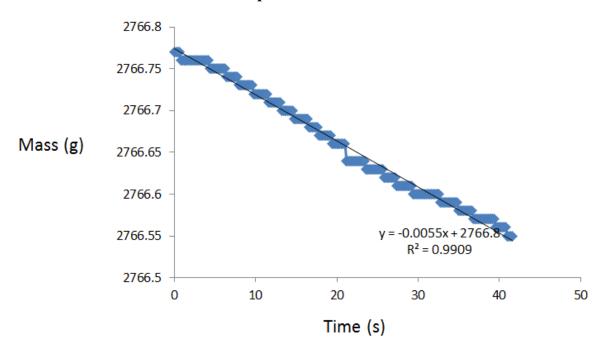


Air I	Air Flow Rate Fuel Flow rate		Volume flow rate	nozzle width	U	Phi	
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2	0.001386	0.158	0.000109	3.59E-05	0.010744	0.39646	0.75208

Entrainment Rate 0.0061 g/sec Concentration 169.7105 g/m³

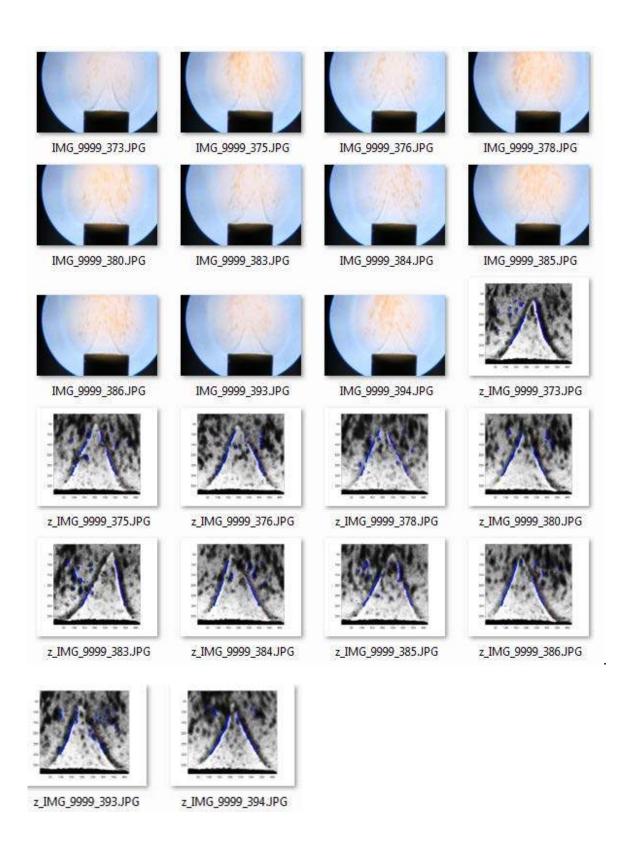


Experiment NO. 5

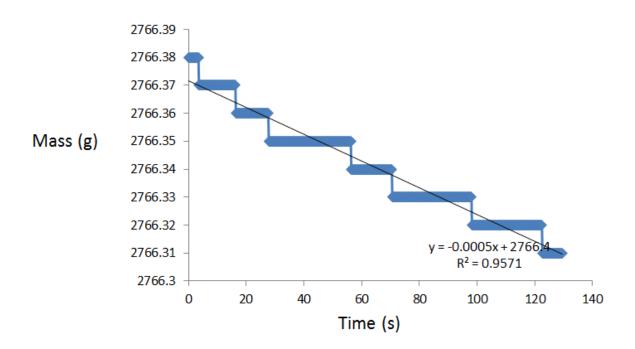


Air F	Air Flow Rate Fuel Flow rate		Volume flow rate	nozzle width	U	Phi	
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2	0.001386	0.158	0.000109	3.59E-05	0.010744	0.39646	0.75208

Entrainment Rate 0.0055 g/sec Concentration 153.0177 g/m³

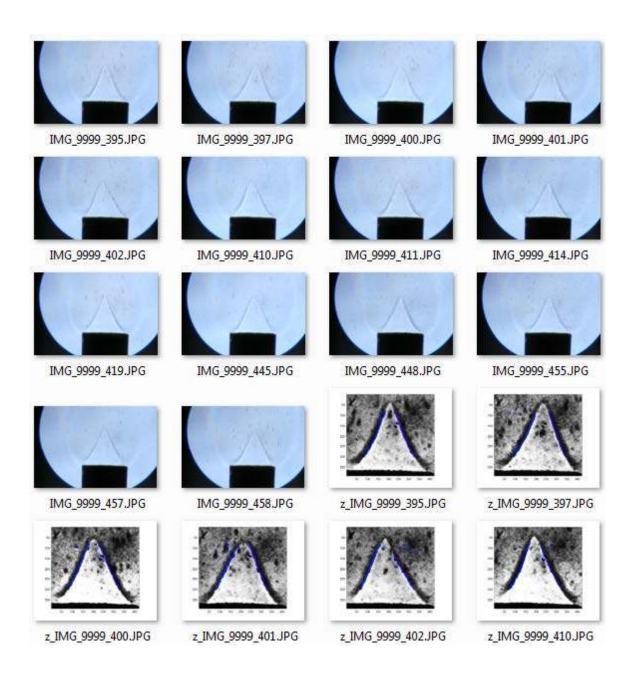


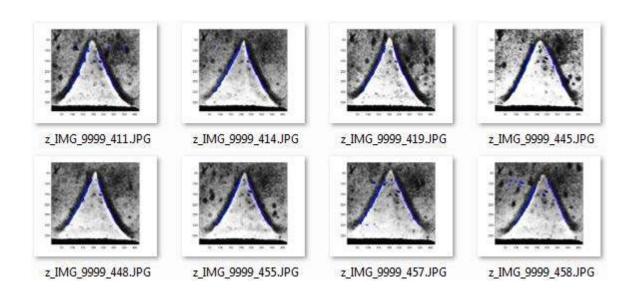
Experiment NO. 6



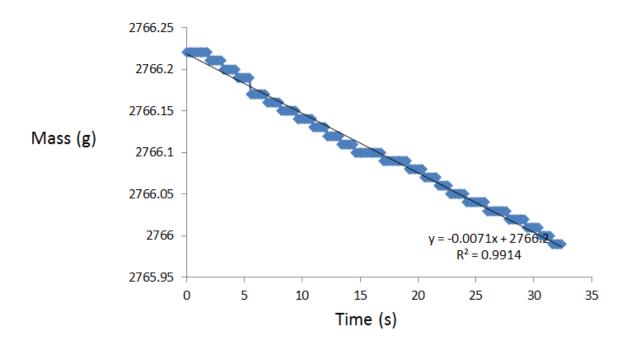
Air I	Air Flow Rate Fuel Flow		Flow rate	Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2	0.001386	0.158	0.000109	3.59E-05	0.010744	0.39646	0.75208

Entrainment Rate 0.0005 g/sec Concentration 13.9107 g/m³



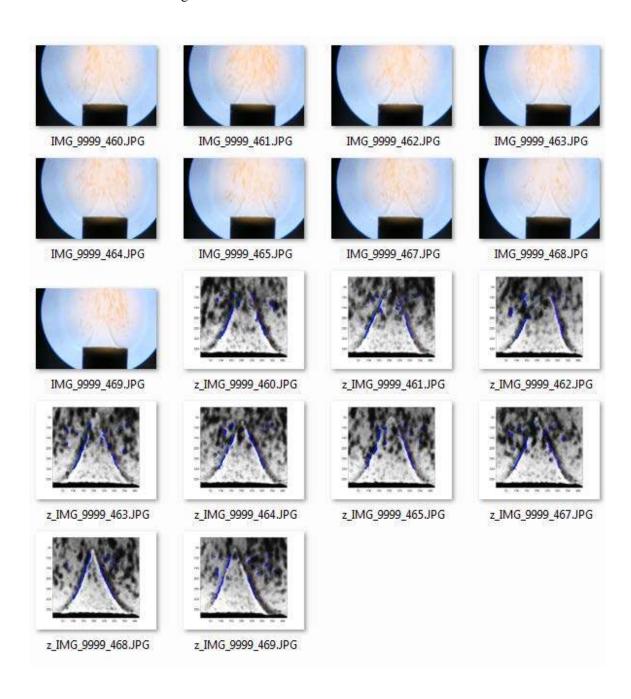


## Experiment NO. 7

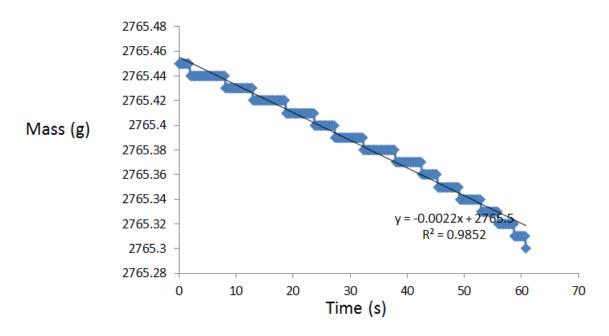


Air Flo	Flow Rate Fuel Flow rate		Volume flow rate	nozzle width	U	Phi	
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2	0.001386	0.158	0.000109	3.59E-05	0.010744	0.39646	0.75208

Entrainment Rate 0.0071 g/sec Concentration 197.5319 g/m³

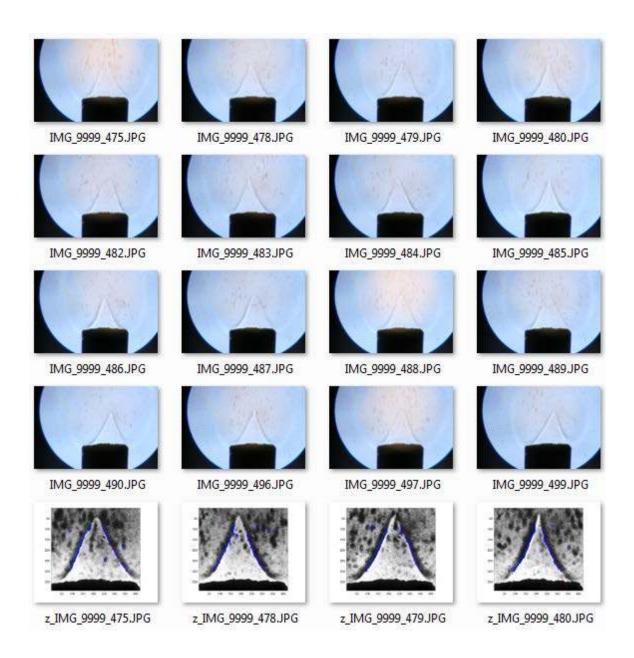


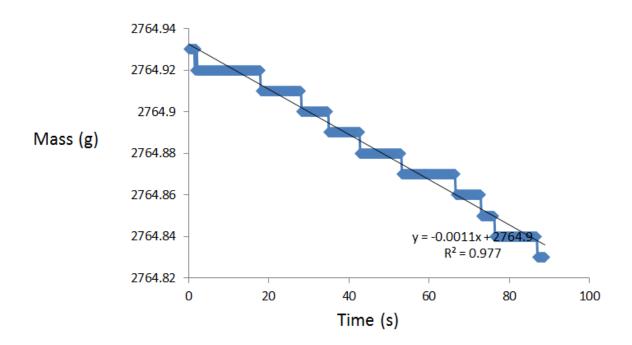
Experiment NO. 8



Air F	Air Flow Rate Fuel Flow rate		Volume flow rate	nozzle width	U	Phi	
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2	0.001386	0.158	0.000109	3.59E-05	0.010744	0.39646	0.75208

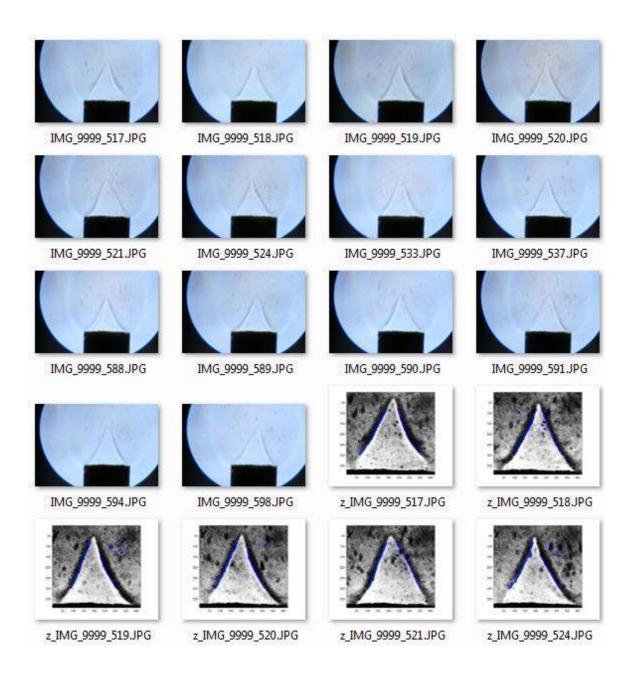
Entrainment Rate 0.0022 g/sec Concentration 61.20707 g/m³

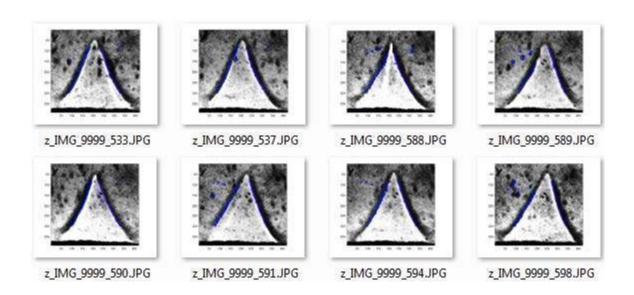




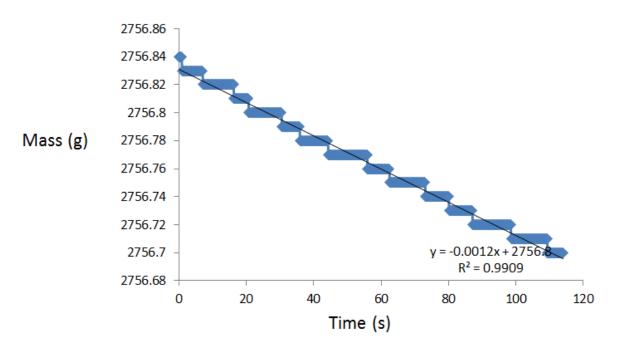
Air Flow Rate		Fuel	Flow rate	Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2	0.001386	0.158	0.000109	3.59E-05	0.010744	0.39646	0.75208

Entrainment Rate 0.0011 g/sec Concentration 30.60353 g/m³



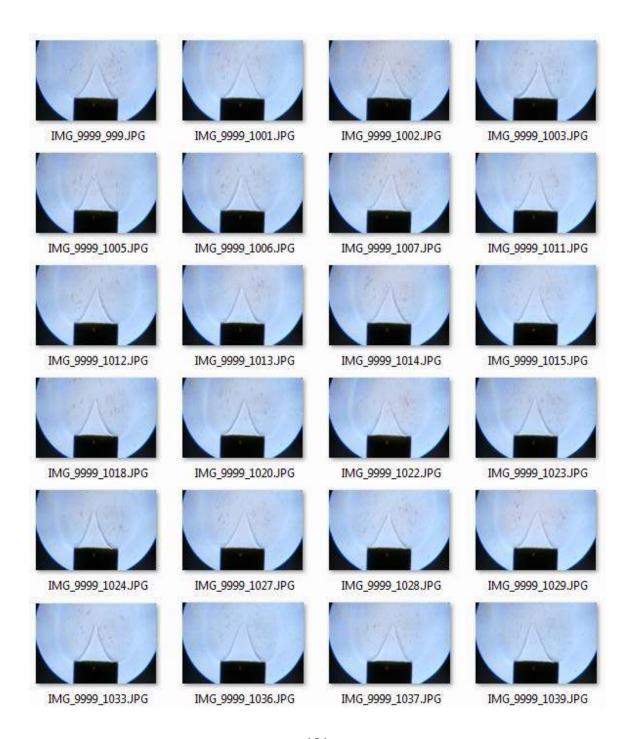


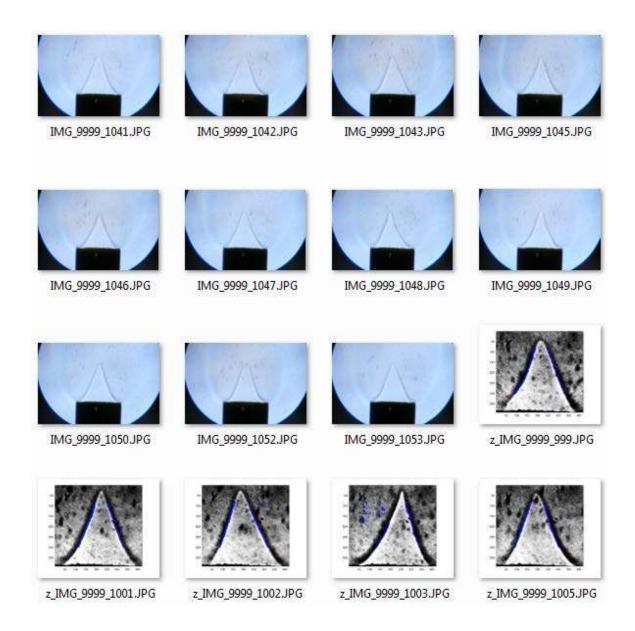
3.2  $\phi$ =0.80; Flow Rate: Air 2.5 SLPM; Methane 0.21 SLPM Experiment NO. 1



Air Flow Rate		Fuel	Flow rate	Volume flow rate	nozzle width	U
SLPM	mole/sec	SLPM mole/sec		m3/s	m	m/sec
2.5	0.001732	0.21	0.000145	4.51E- 05	0.010744	0.497869

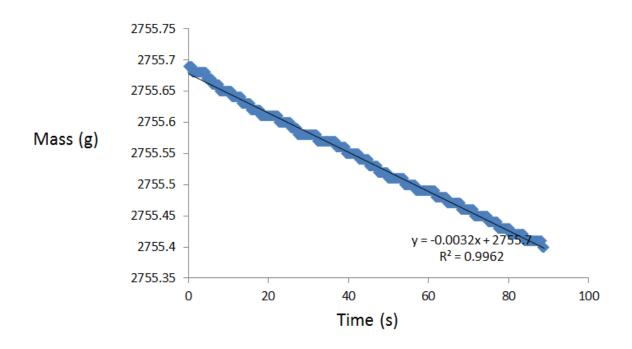
Entrainment Rate 0.0012 g/sec Concentration 26.58546 g/m<sup>3</sup>





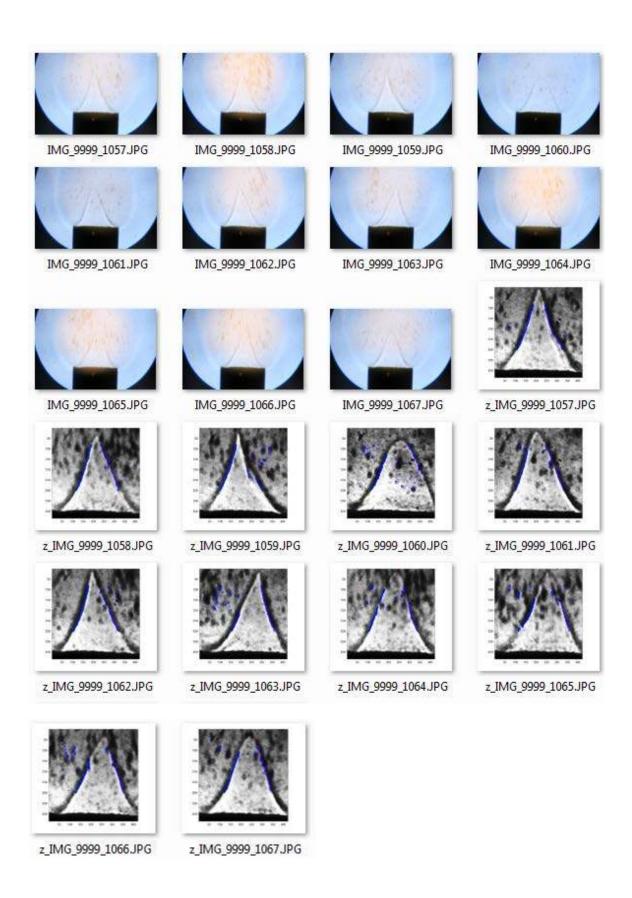


Experiment NO. 2

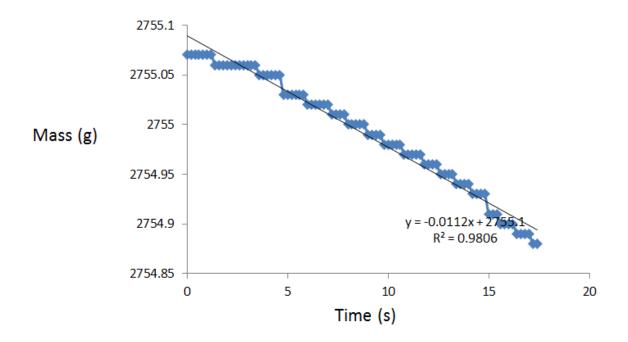


Air Flow Rate		Fuel Flo	w rate	Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.00173	0.21	0.00014 5	4.51E-05	0.010744	0.49786 9	0.7996 8

Entrainment Rate 0.0032 g/sec Concentration 70.89455 g/m<sup>3</sup>

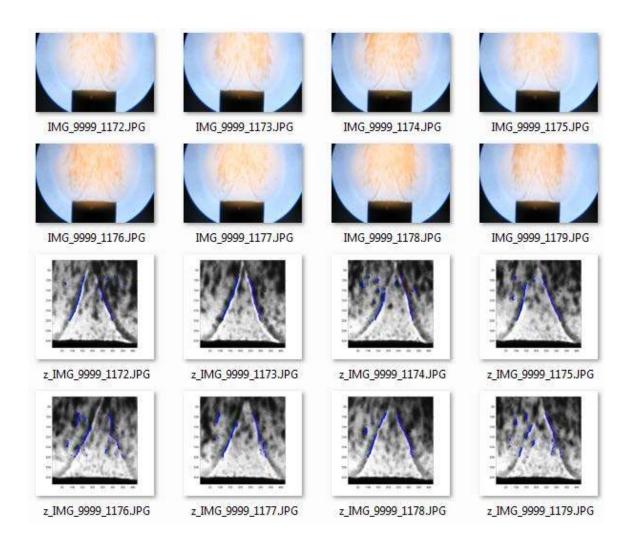


Experiment NO. 4

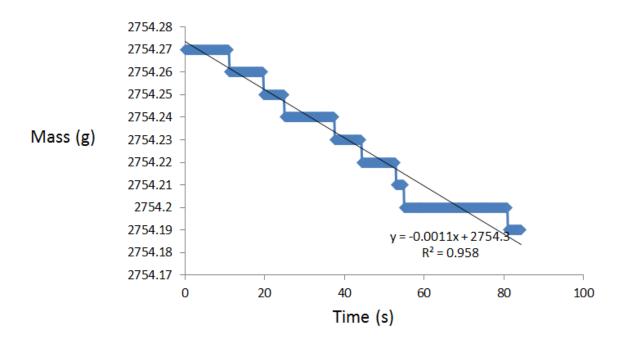


Air Flow Rate		Fuel	Flow rate	Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.21	0.000145	4.51E-05	0.010744	0.497869	0.79968

Entrainment Rate 0.0112 g/sec Concentration 248.1309 g/m³

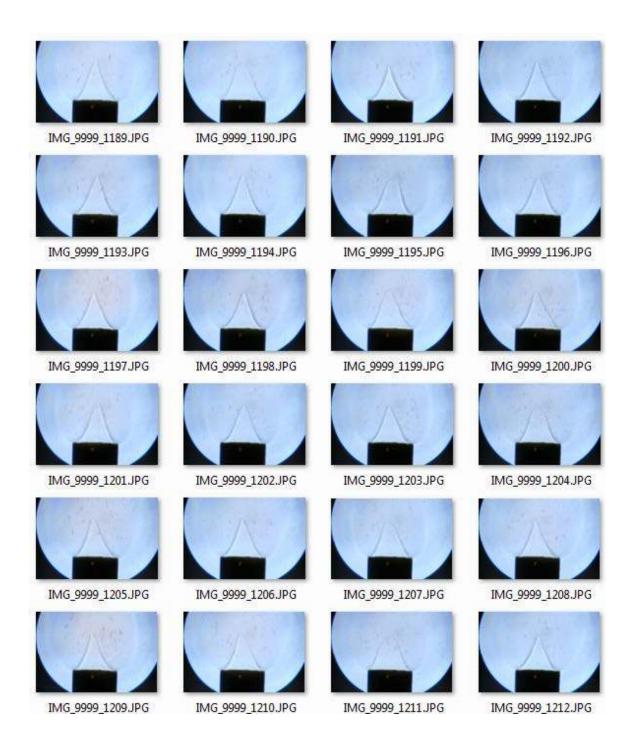


Experiment NO. 5

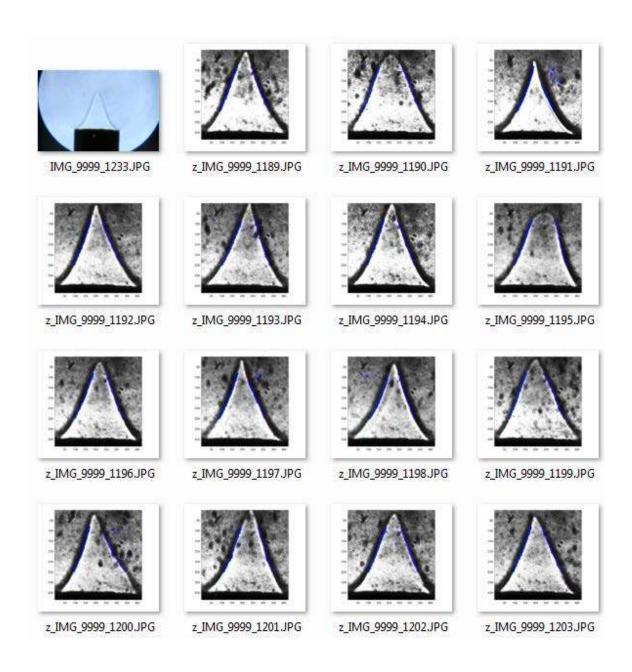


Air Flow Rate		Fuel F	low rate	Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.21	0.000145	4.51E-05	0.010744	0.497869	0.79968

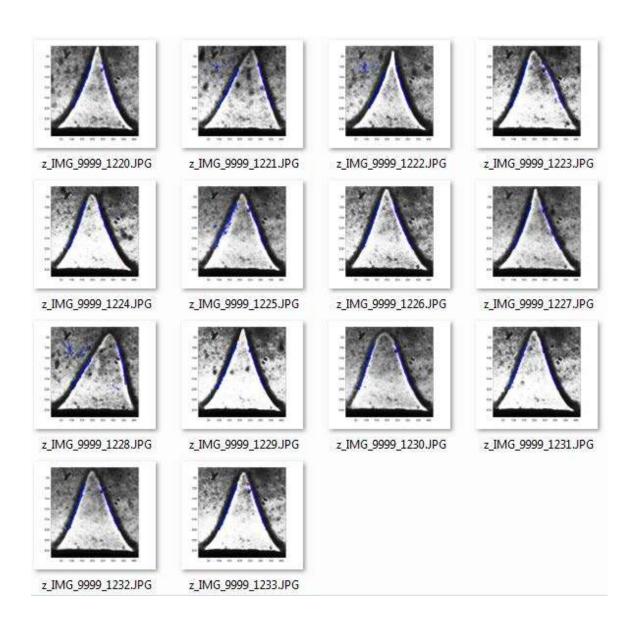
Entrainment Rate 0.0011 g/sec Concentration 24.37 g/m³



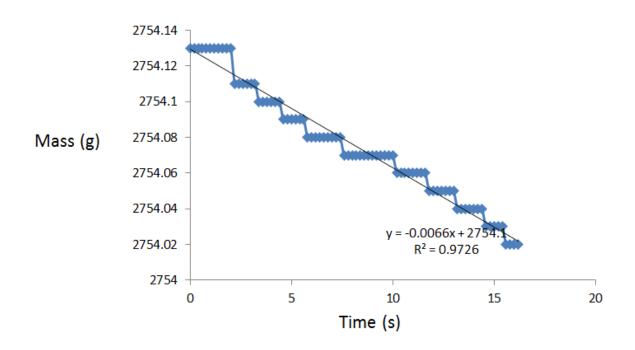






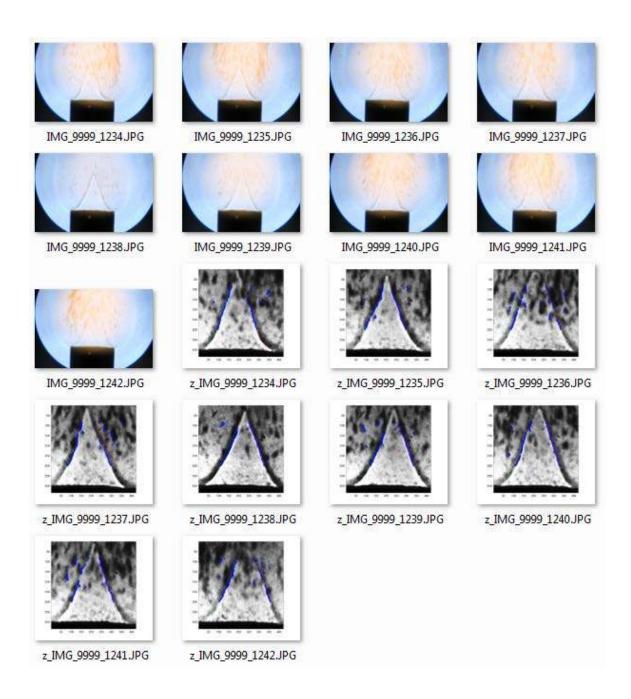


Experiment NO. 6

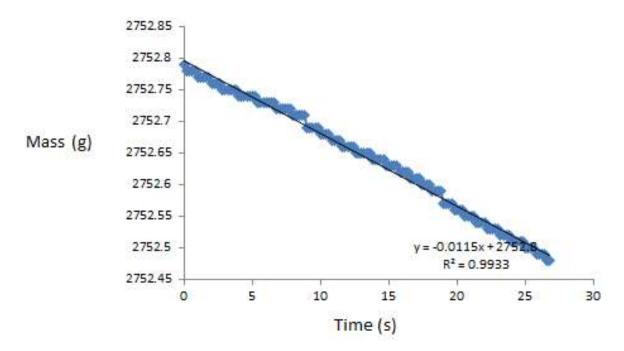


Air Flo	ow Rate	Fuel F	low rate	Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.21	0.000145	4.51E-05	0.010744	0.497869	0.79968

Entrainment Rate 0.0066 g/sec Concentration 146.22 g/m<sup>3</sup>

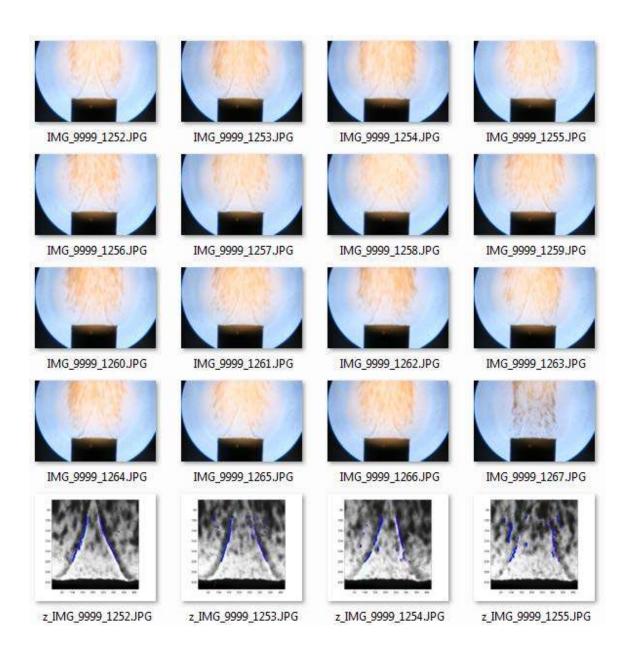


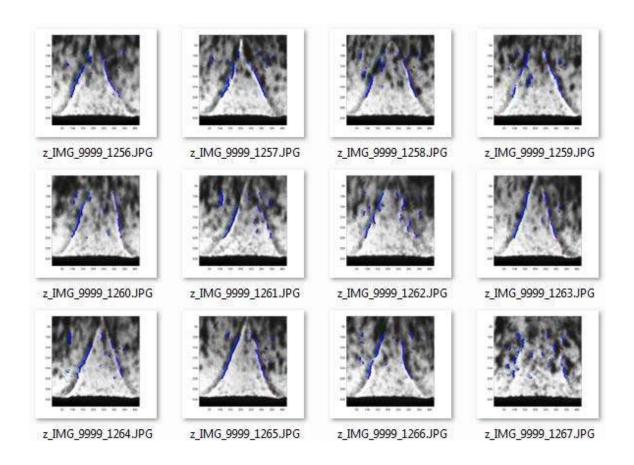
## Experiment NO. 8



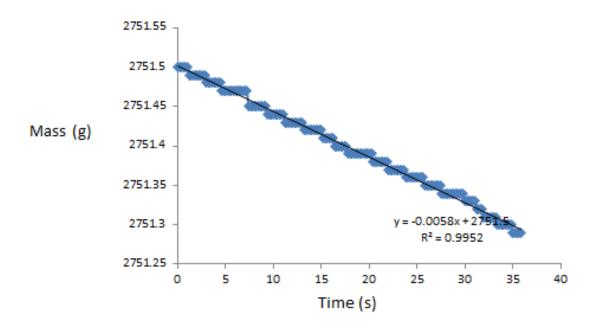
Air F	Flow Rate	Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.21	0.000145	4.51E-05	0.010744	0.497869	0.79968

Entrainment Rate 0.0115 g/sec Concentration 254.7773 g/m³



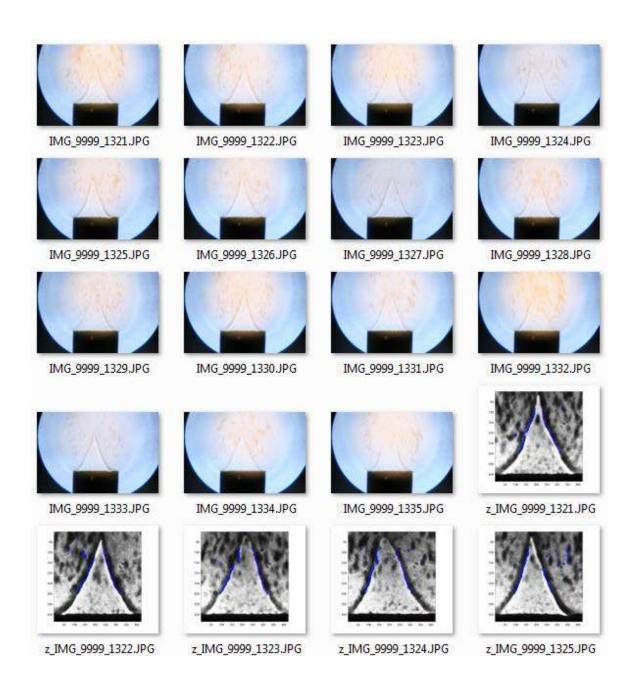


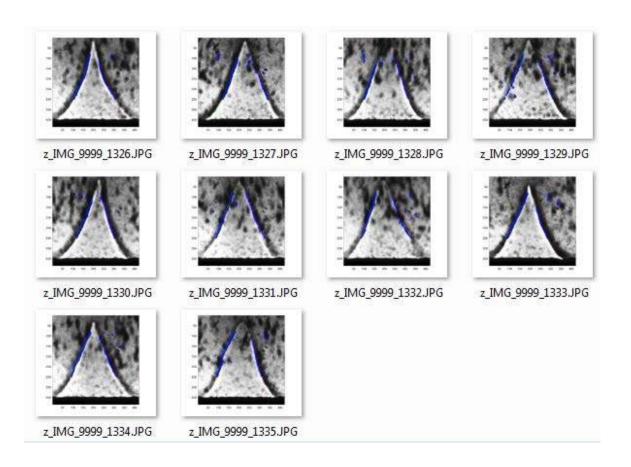
## **Experiment NO. 11**



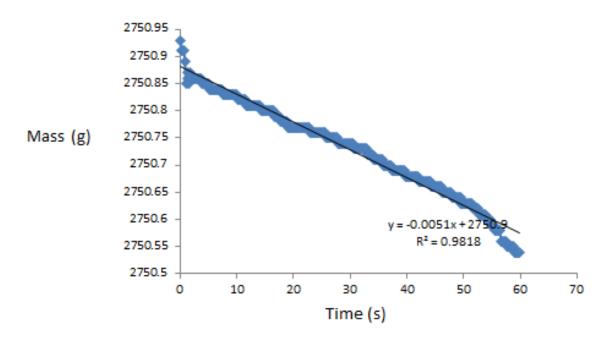
Air Flo	ow Rate	Fuel F	low rate	Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.21	0.000145	4.51E-05	0.010744	0.497869	0.79968

Entrainment Rate 0.0058 g/sec Concentration 128.4964 g/m³



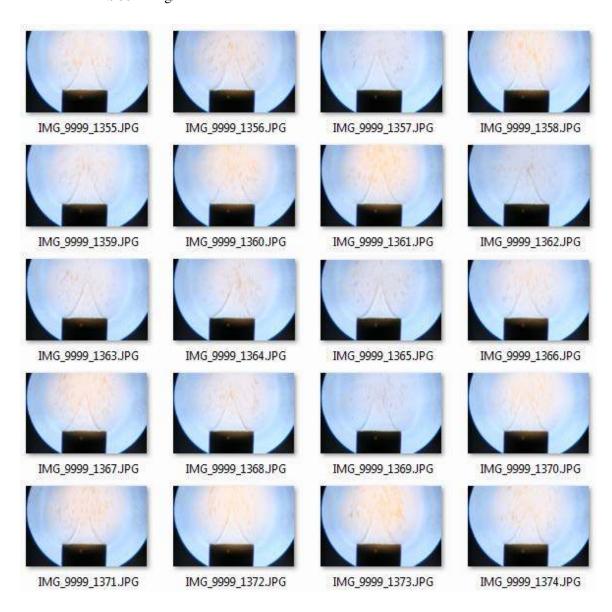


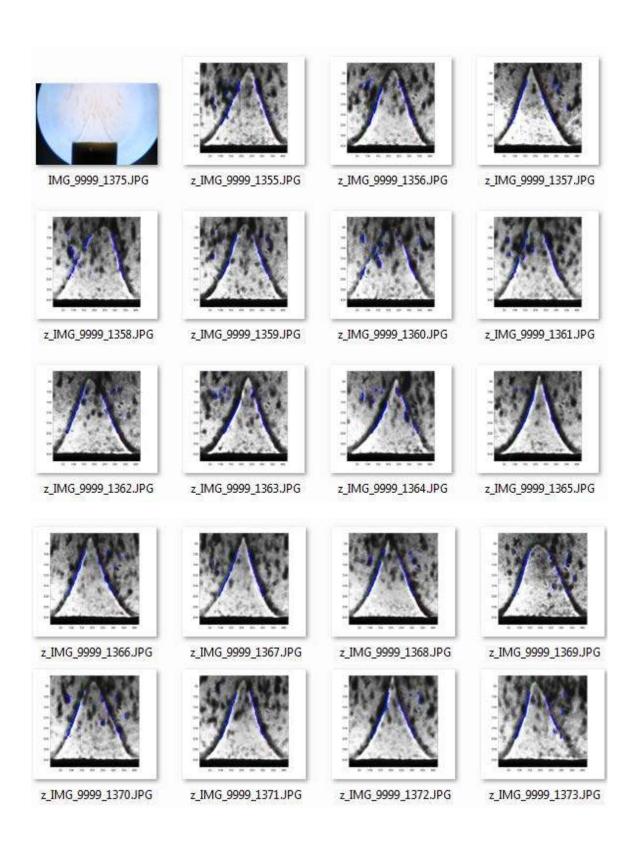
**Experiment NO. 12** 



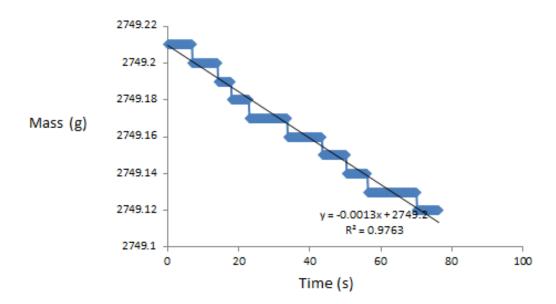
Air F	Flow Rate	Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.21	0.000145	4.51E-05	0.010744	0.497869	0.79968

Entrainment Rate 0.0051 g/sec Concentration 112.9882 g/m³



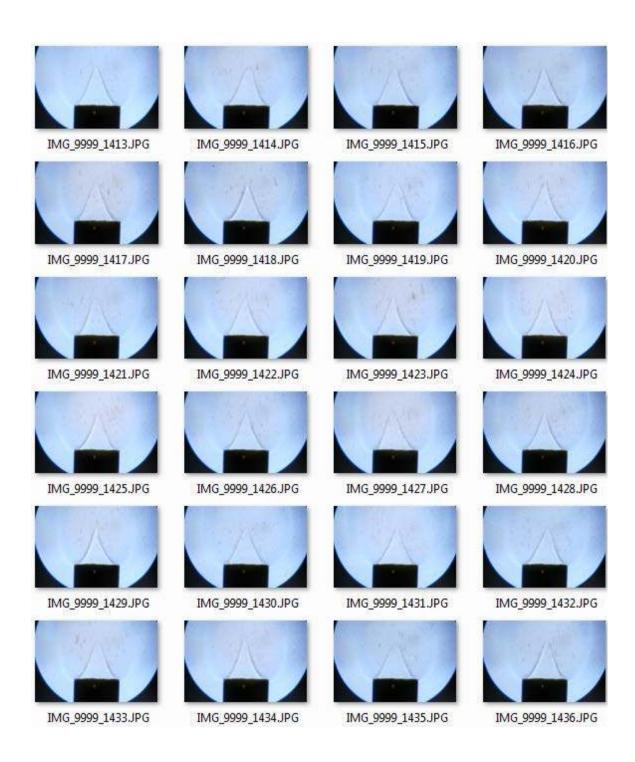


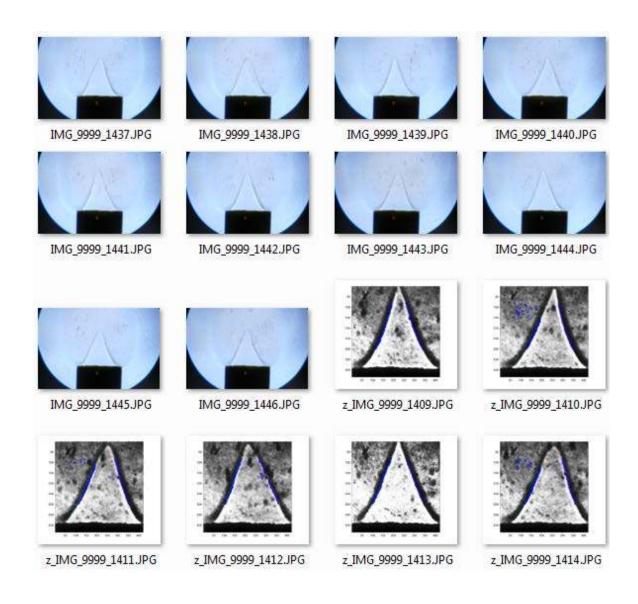
Experiment NO. 13

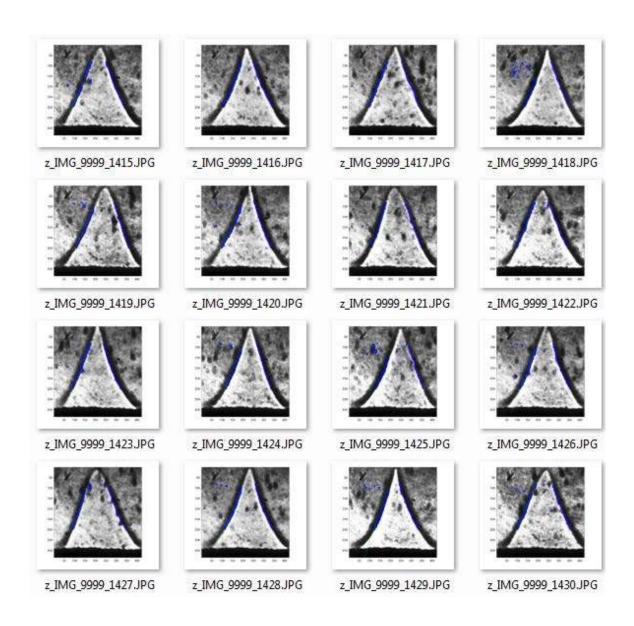


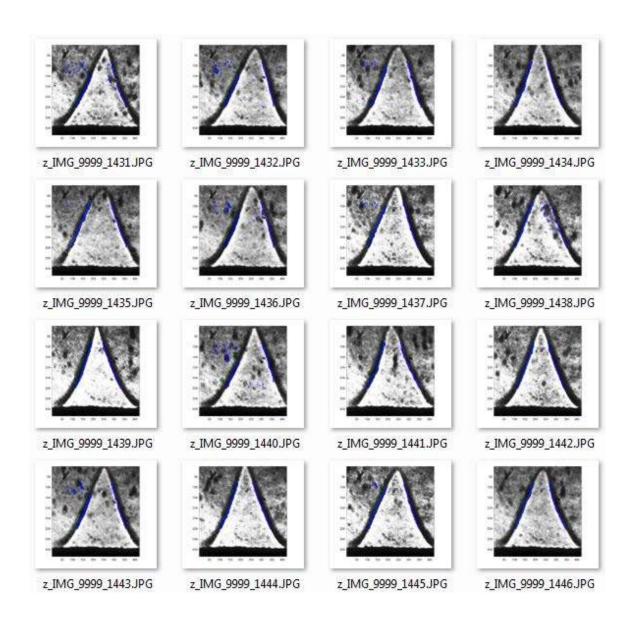
Air F	low Rate	Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLP M	mole/sec	SLP M	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.21	0.000145	4.51E-05	0.010744	0.497869	0.79968

Entrainment Rate 0.0013 g/sec Concentration 28.80091 g/m<sup>3</sup>

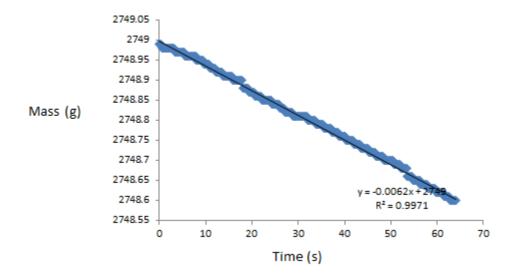






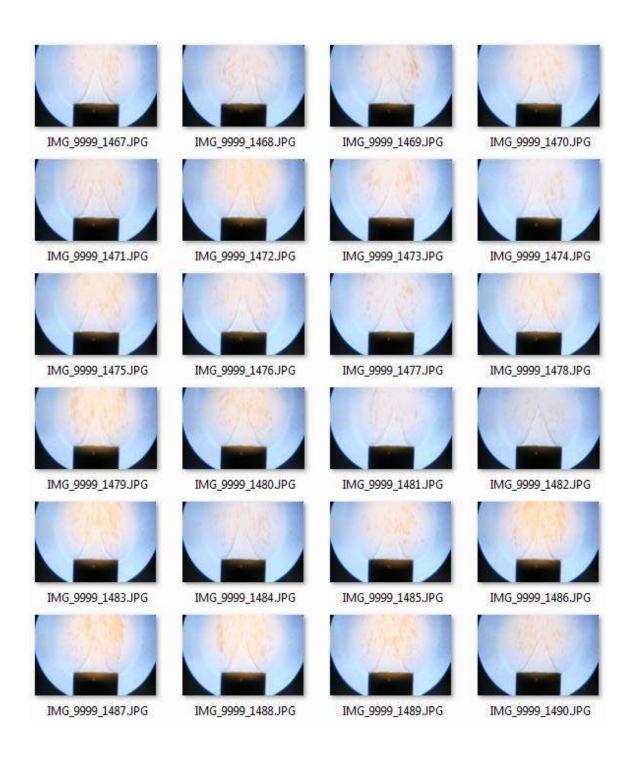


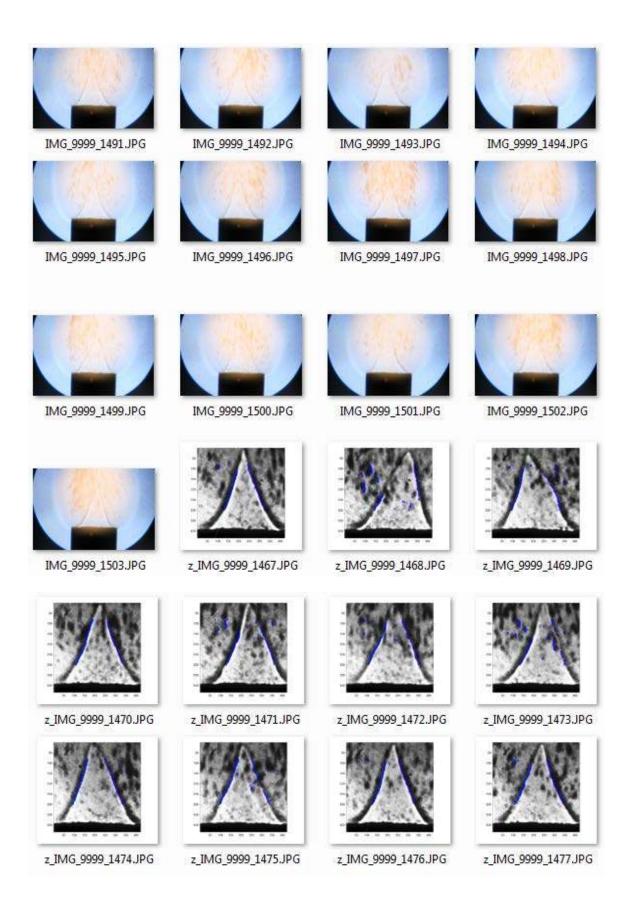
**Experiment NO. 14** 

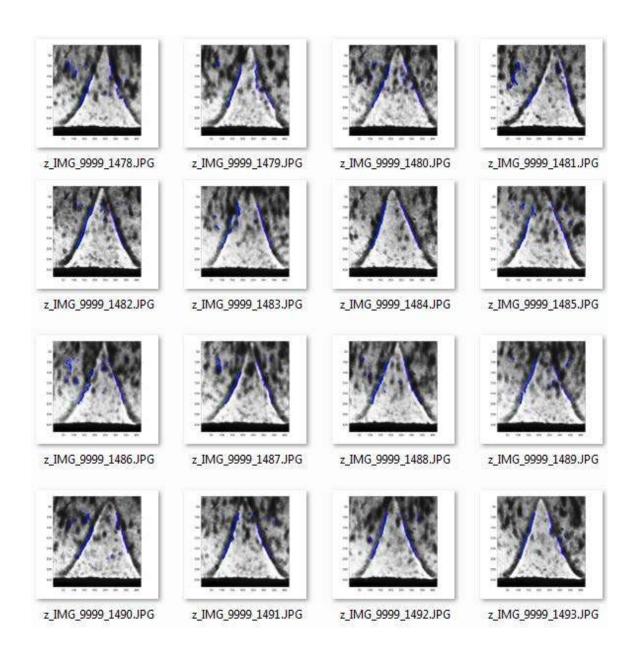


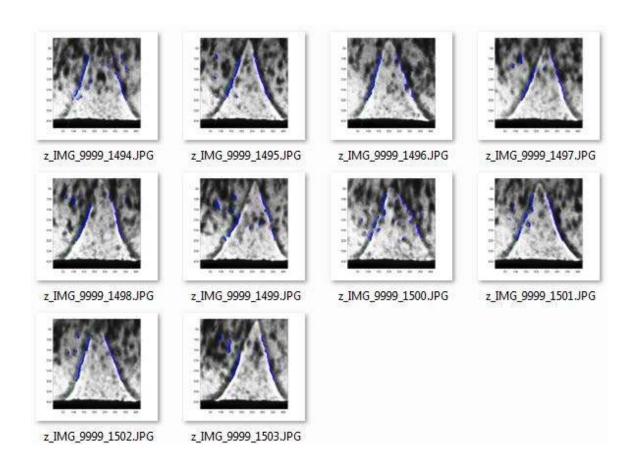
Air F	ir Flow Rate Fuel Flow rate		Volume flow rate	nozzle width	U	Phi	
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.21	0.000145	4.51E-05	0.010744	0.497869	0.79968

Entrainment Rate 0.0062 g/sec Concentration 137.3582 g/m<sup>3</sup>

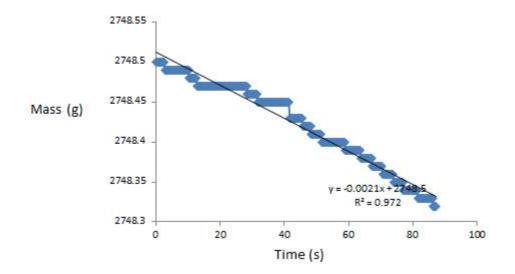






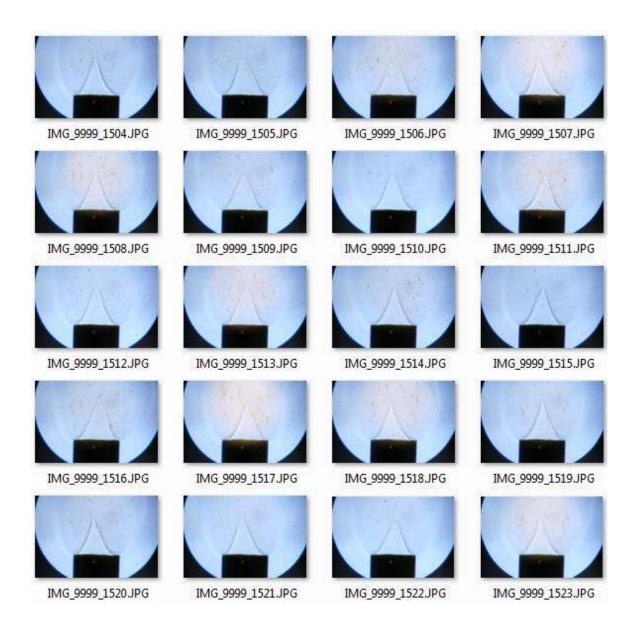


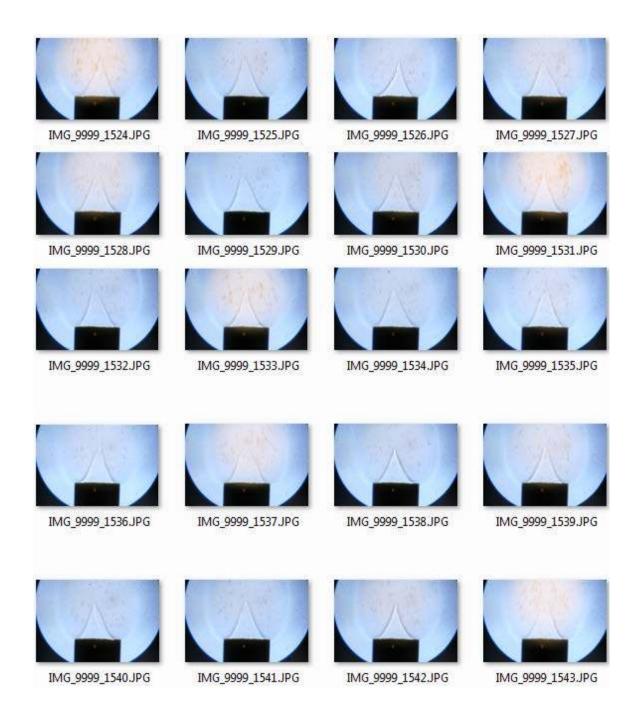
**Experiment NO. 15** 

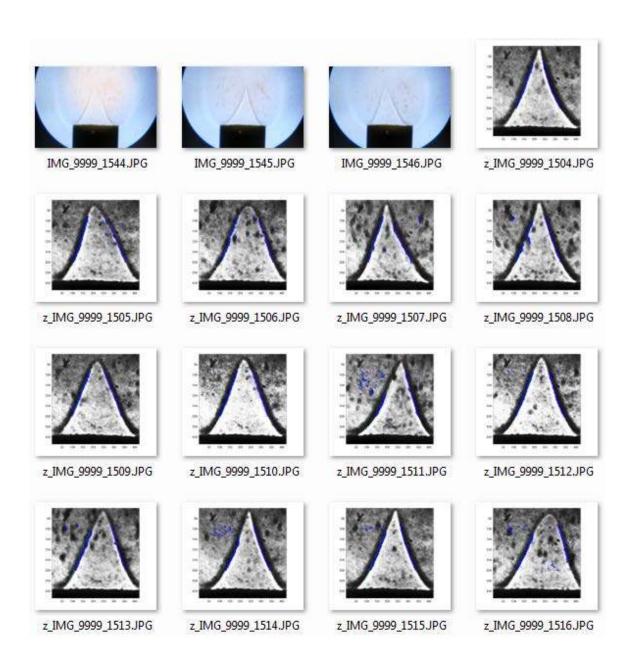


Air I	Flow Rate	Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLP M	mole/sec	SLP M	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.21	0.00014	4.51E-05	0.01074 4	0.497869	0.7996 8

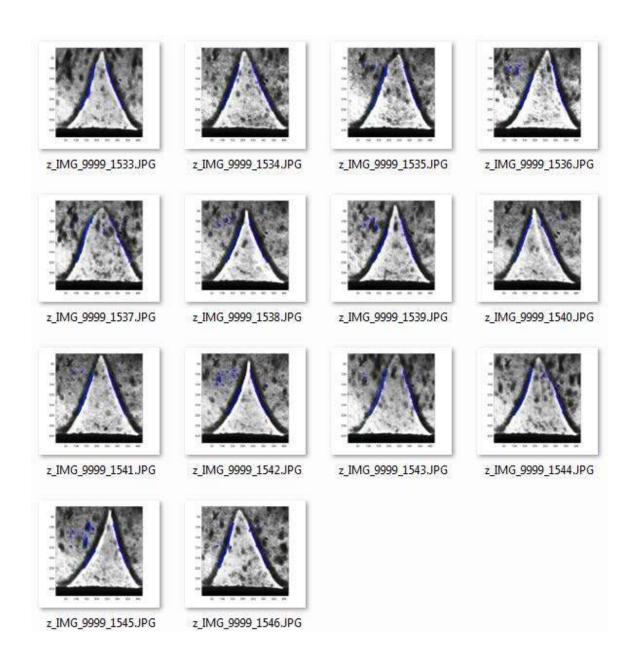
Entrainment Rate 0.0021 g/sec Concentration 46.5245 5 g/m³



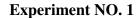


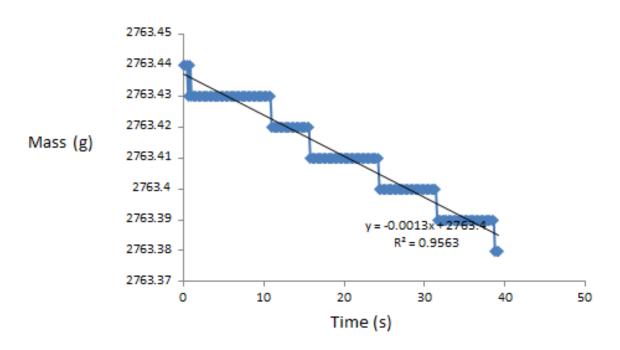






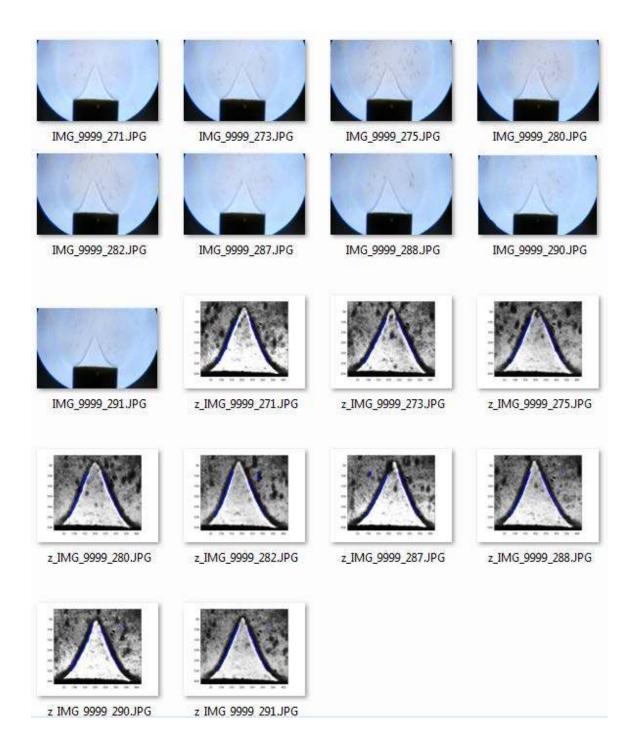
## 3.3 $\phi$ =0.85; Flow Rate: Air 2.5 SLPM; Methane 0.223 SLPM



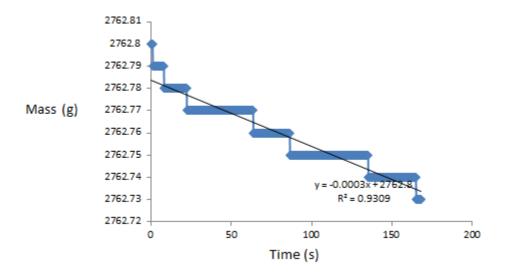


Air F	ir Flow Rate Fuel Flow rate		Volume flow rate	nozzle width	U	Phi	
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.223	0.000155	4.54E-05	0.010744	0.500255	0.849184

Entrainment Rate 0.0013 g/sec Concentration 28.66353 g/m³

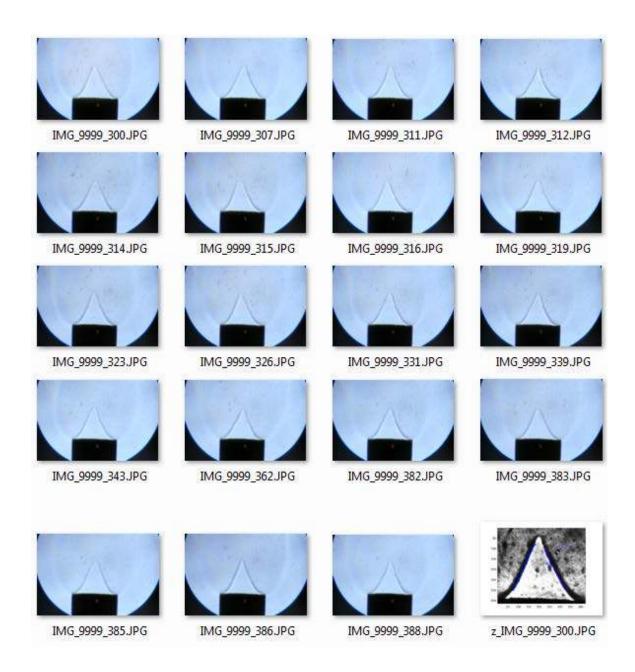


Experiment NO. 2



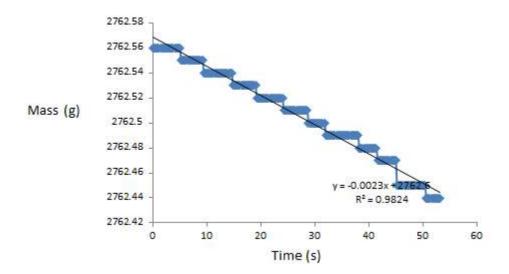
Air F	Air Flow Rate Fuel Flow rate		Volume flow rate	nozzle width	U	Phi	
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.223	0.000155	4.54E-05	0.010744	0.500255	0.849184

Entrainment Rate 0.0003 g/sec Concentration 6.614662 g/m<sup>3</sup>



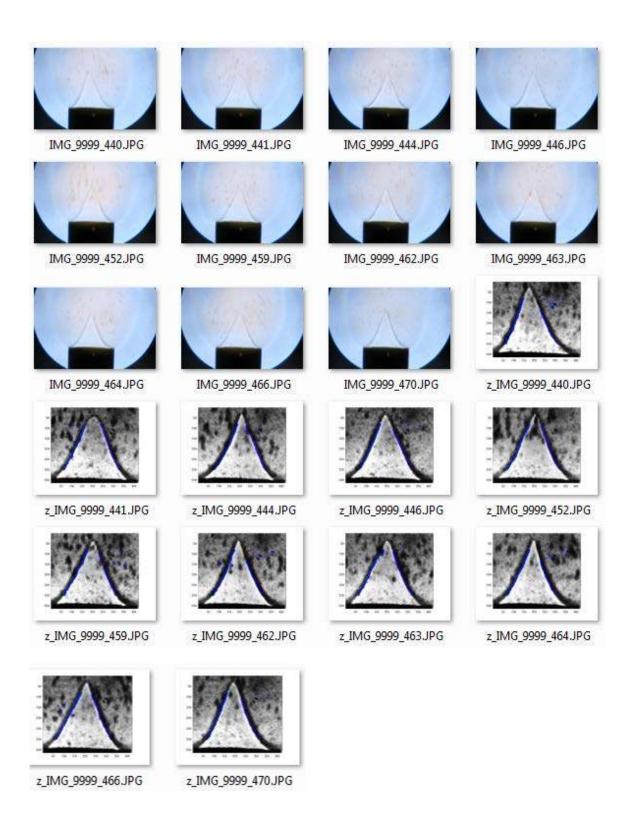


Experiment NO. 4

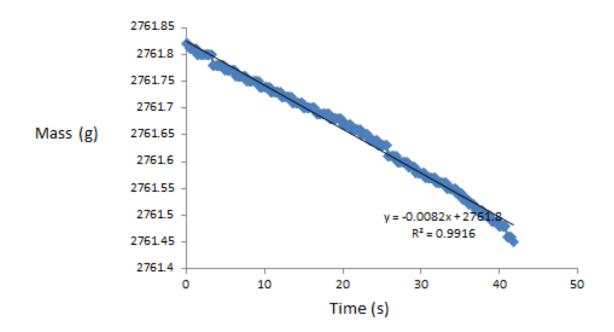


Air F	Flow Rate	Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.223	0.000155	4.54E-05	0.010744	0.500255	0.849184

Entrainment Rate 0.0023 g/sec Concentration 50.71241 g/m³

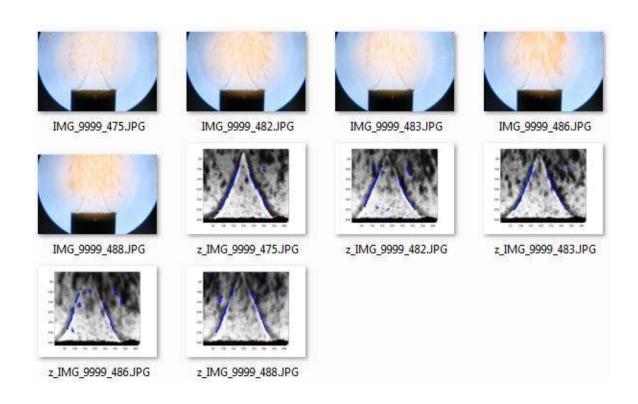


Experiment NO. 5

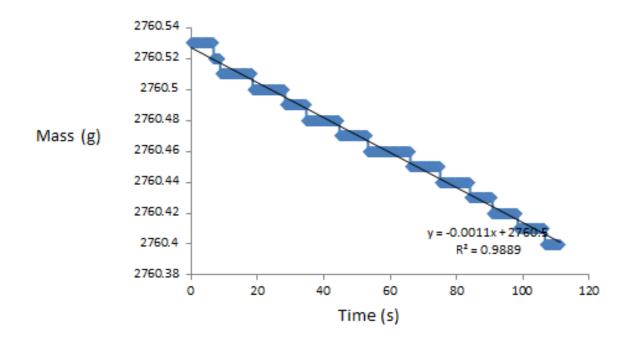


Air F	Flow Rate	Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.223	0.000155	4.54E-05	0.010744	0.500255	0.849184

Entrainment Rate 0.0082 g/sec Concentration 180.8008 g/m³

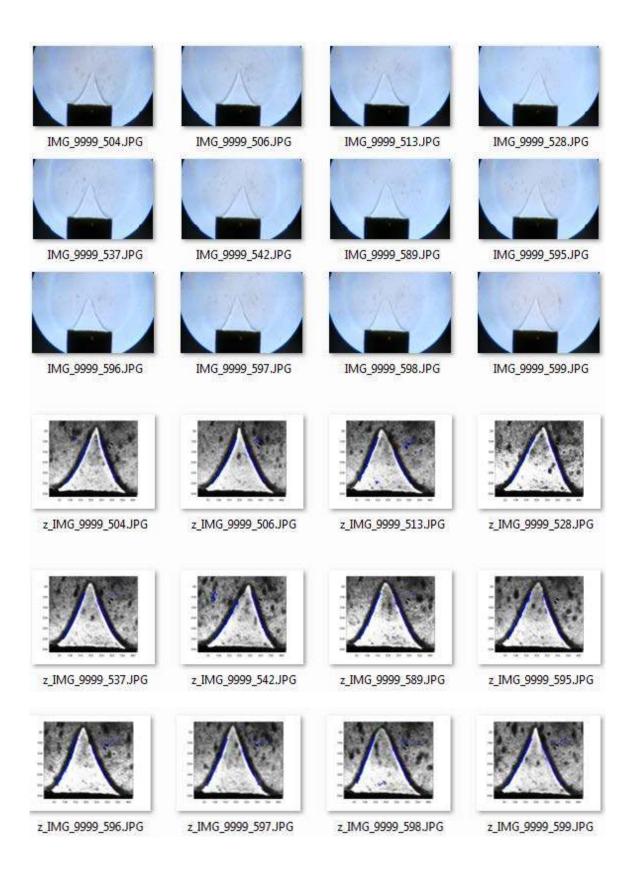


Experiment NO. 6

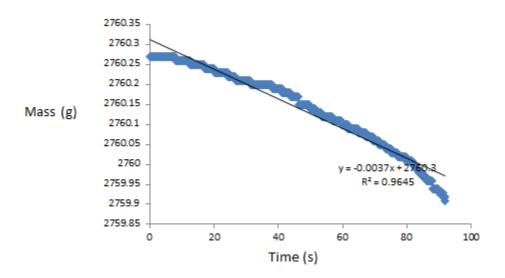


Air F	Air Flow Rate Fuel Flow rate		Volume flow rate	nozzle width	U	Phi	
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.223	0.000155	4.54E-05	0.010744	0.500255	0.849184

Entrainment Rate 0.0011 g/sec Concentration 24.25376 g/m³

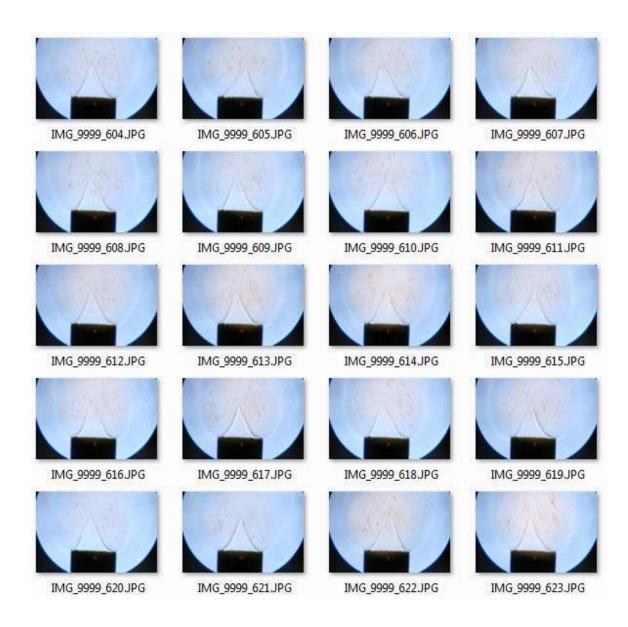


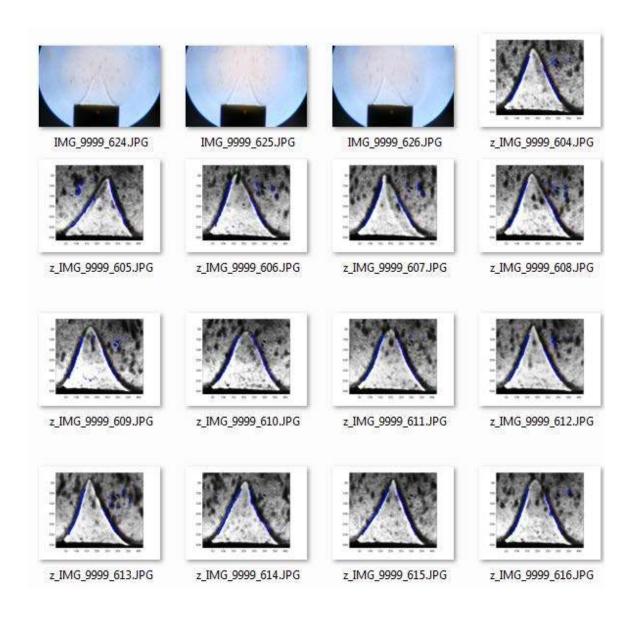
## Experiment NO. 7



Air F	Air Flow Rate Fuel Flow rate		Volume flow rate	nozzle width	U	Phi	
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.223	0.000155	4.54E-05	0.010744	0.500255	0.849184

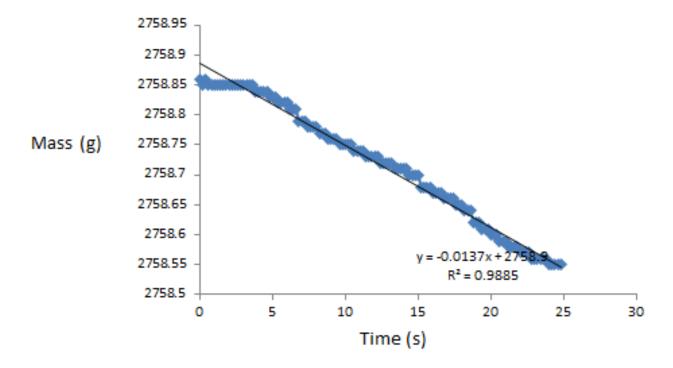
Entrainment Rate 0.0037 g/sec Concentration 81.58083 g/m³





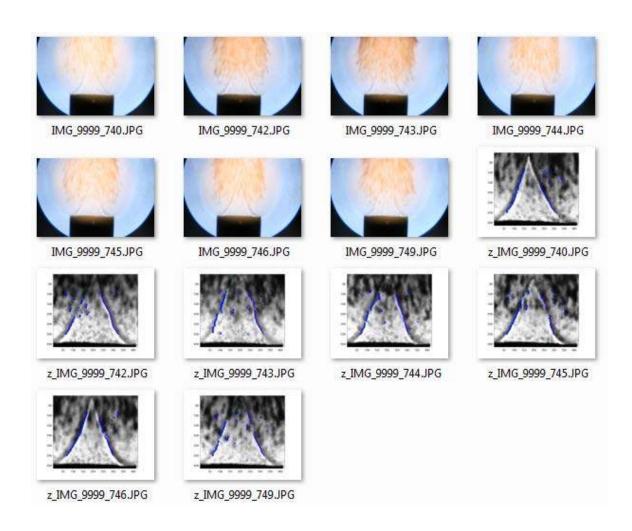


#### **Experiment NO.8**

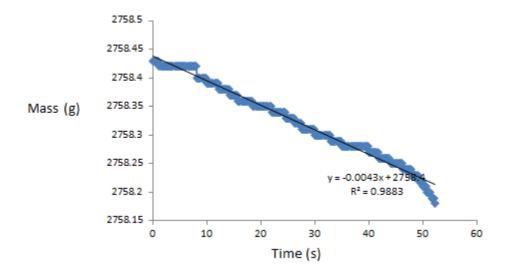


Air F	Air Flow Rate Fuel Flow rate		Volume flow rate	nozzle width	U	Phi	
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.223	0.000155	4.54E-05	0.010744	0.500255	0.849184

Entrainment Rate 0.0137 g/sec Concentration 302.0696 g/m³

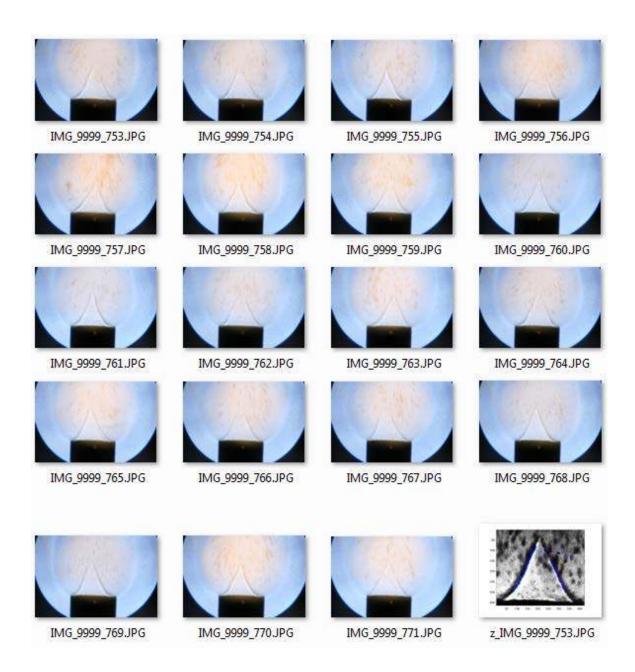


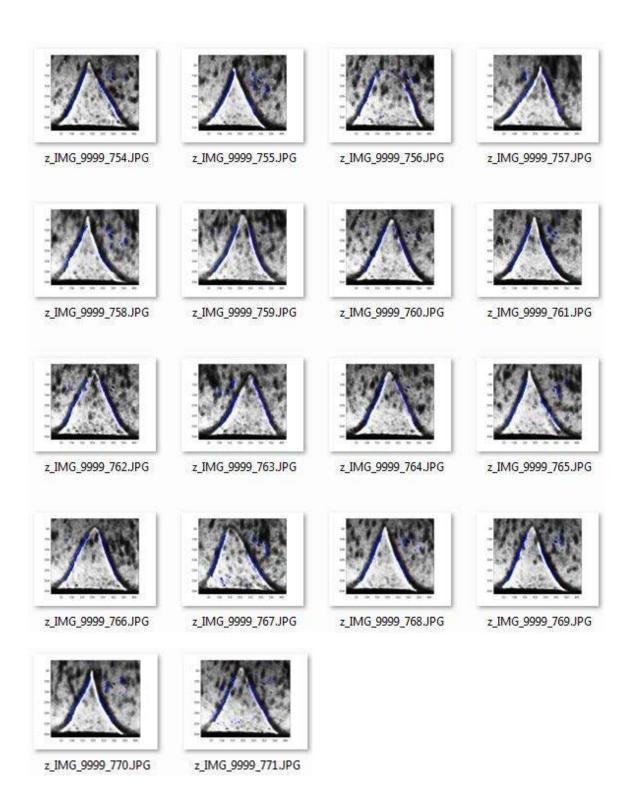
## Experiment NO. 9



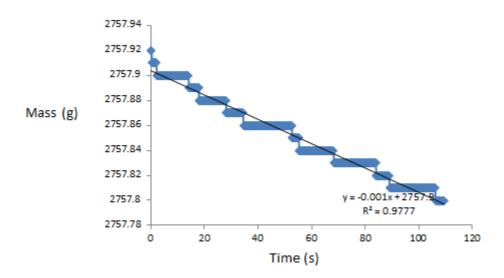
Air I	ir Flow Rate Fuel Flow rate		Volume flow rate	nozzle width	U	Phi	
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.223	0.000155	4.54E- 05	0.010744	0.500255	0.849184

Entrainment Rate 0.0043 g/sec Concentration 94.81015 g/m³



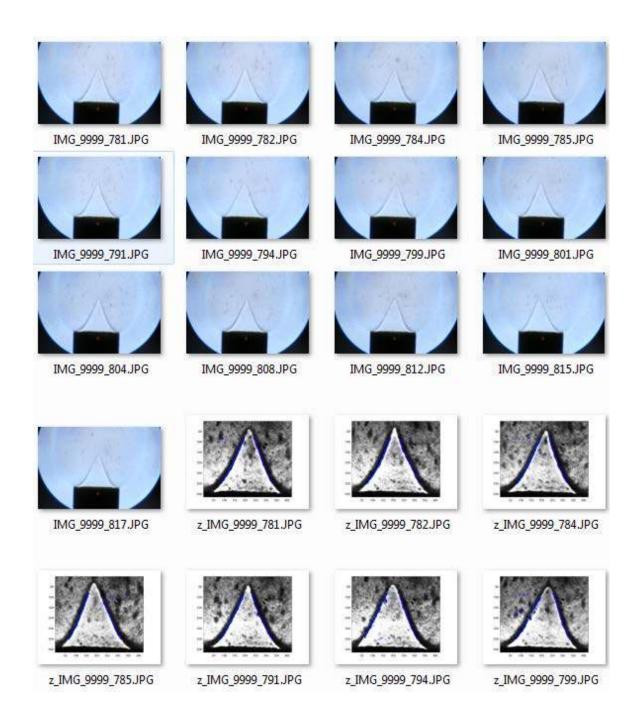


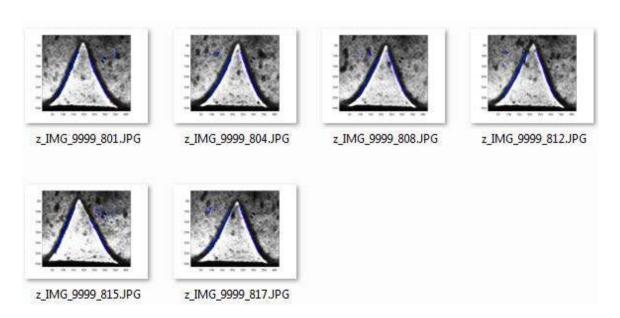
Experiment NO. 10



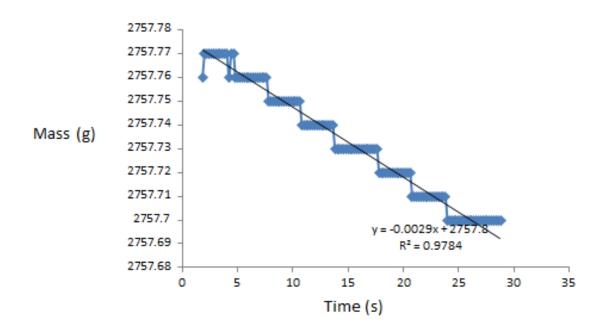
Air F	ow Rate	Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.223	0.000155	4.54E-05	0.010744	0.500255	0.849184

Entrainment Rate 0.0003 g/sec Concentration 6.614662 g/m³



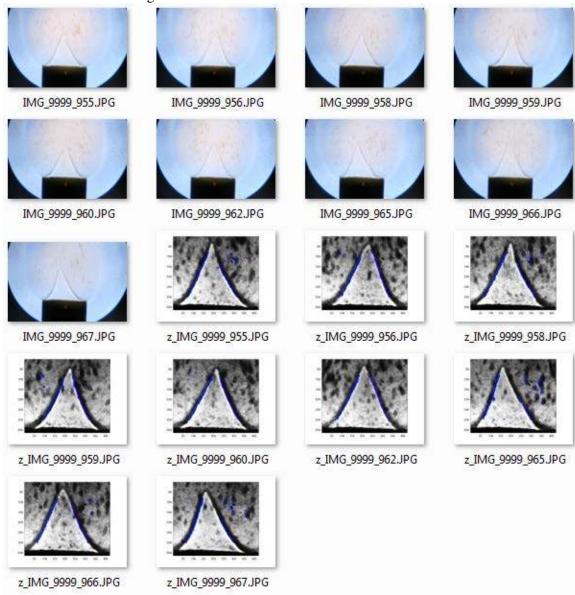


**Experiment NO. 11** 



Air F	Air Flow Rate Fuel Flow rate		Volume flow rate	nozzle width	U	Phi	
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.223	0.000155	4.54E- 05	0.010744	0.500255	0.849184

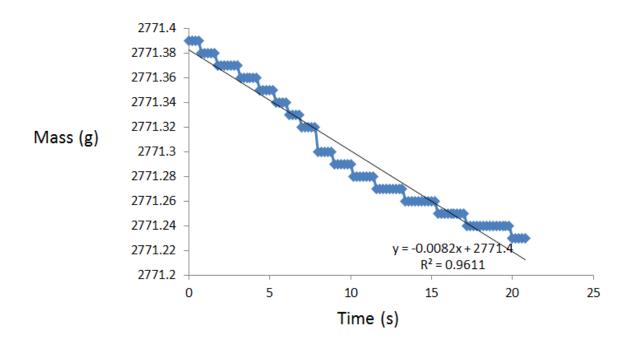
Entrainment Rate 0.0029 g/sec Concentration 63.94173 g/m<sup>3</sup>



## 4. Coal Particles (53-63 μm)

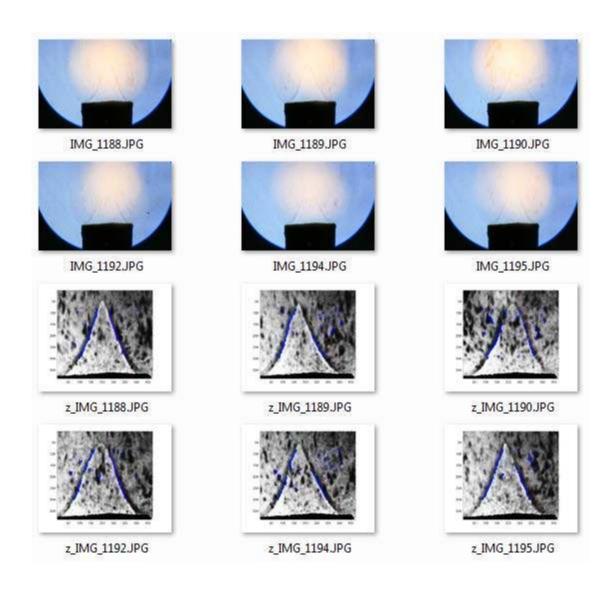
4.1  $\phi$ =0.80; Flow Rate: Air 2.5 SLPM; Methane 0.21 SLPM

**Experiment NO. 5** 

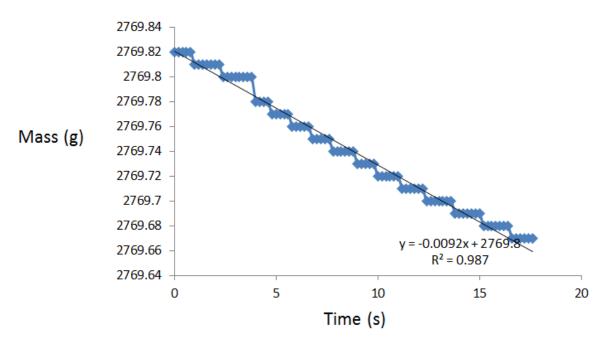


Air I	Flow Rate	Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.21	0.000145	4.51E- 05	0.010744	0.497869	0.79968

Entrainment Rate 8.20E-03 g/sec Concentration 181.6673 g/m³

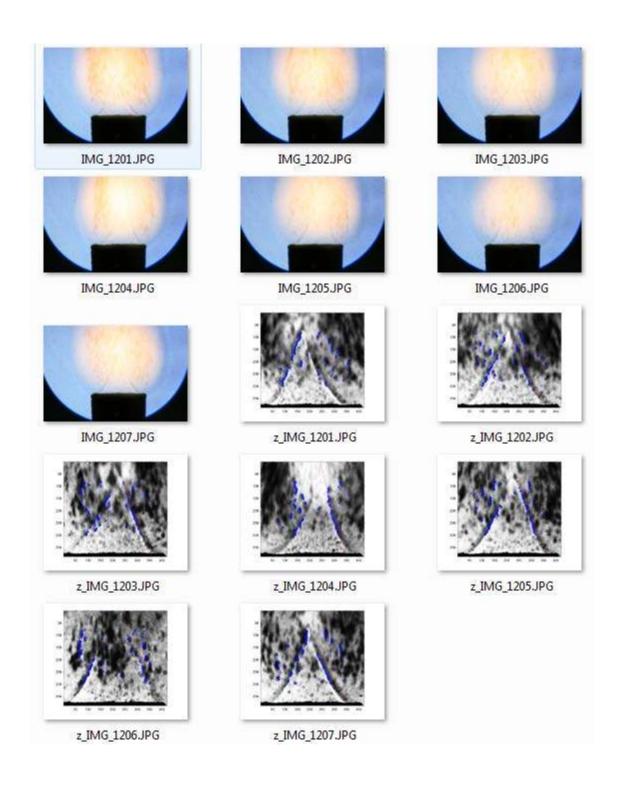


Experiment NO. 6

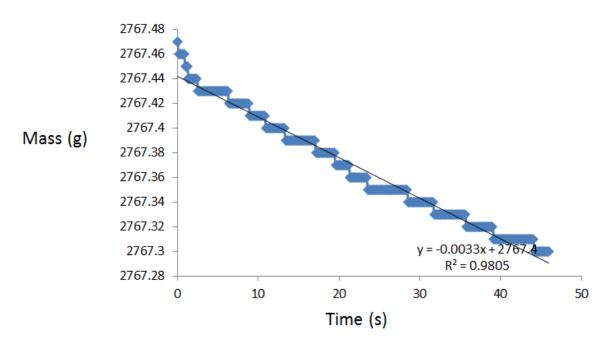


Air I	Flow Rate	Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.21	0.000145	4.51E- 05	0.010744	0.497869	0.79968

Entrainment Rate 9.20E-03 g/sec Concentration 203.8218 g/m³

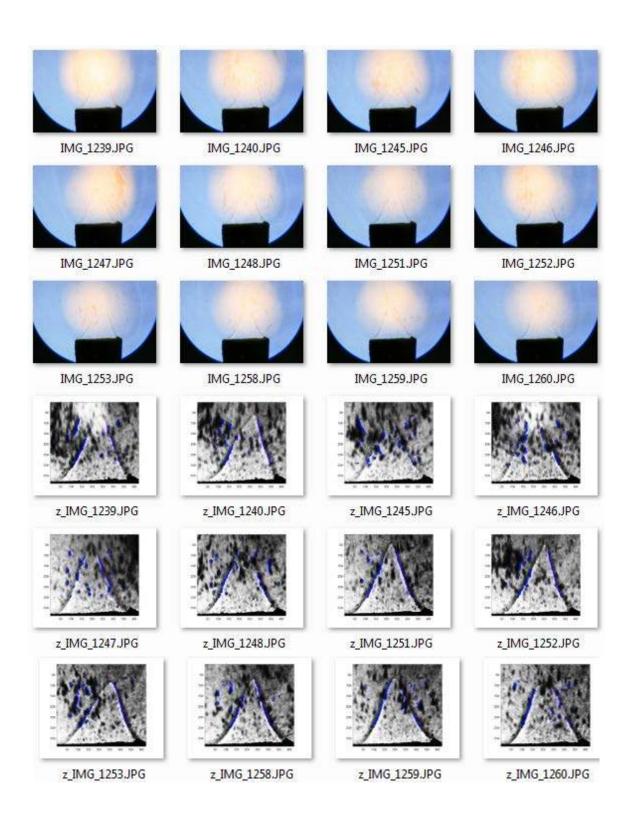


Experiment NO. 8



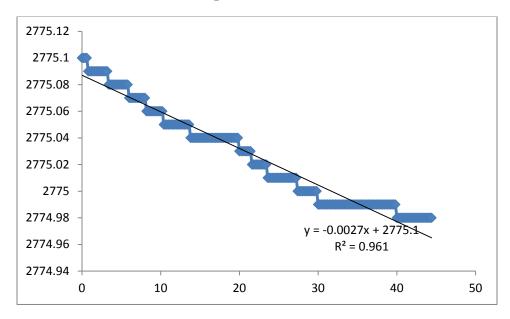
Air I	Flow Rate	Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.21	0.000145	4.51E- 05	0.010744	0.497869	0.79968

Entrainment Rate 3.30E-03 g/sec Concentration 73.11 g/m³



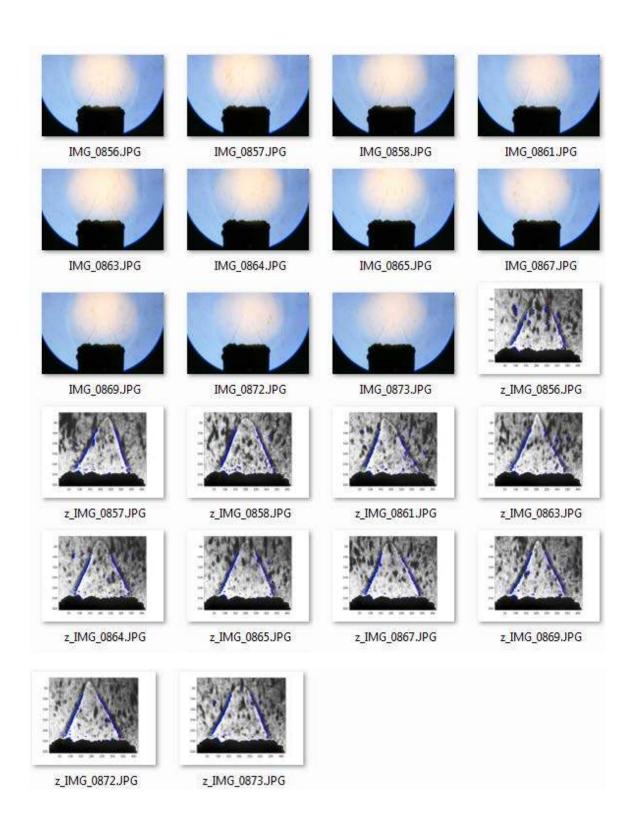
# 4.2 $\phi$ =0.85; Flow Rate: Air 2.5 SLPM; Methane 0.223 SLPM

**Experiment NO. 5** 

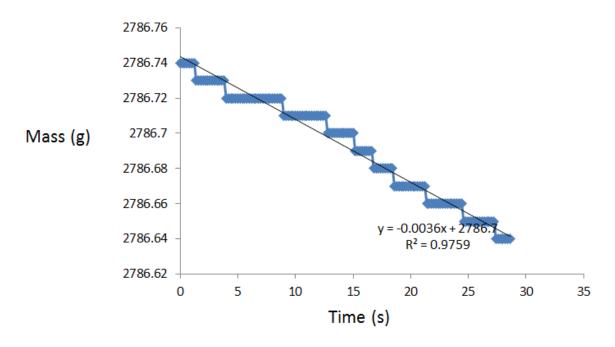


Air l	Flow Rate	Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.223	0.000155	4.54E- 05	0.010744	0.500255	0.849184

Entrainment Rate 2.70E-03 g/sec Concentration 59.53196 g/m³

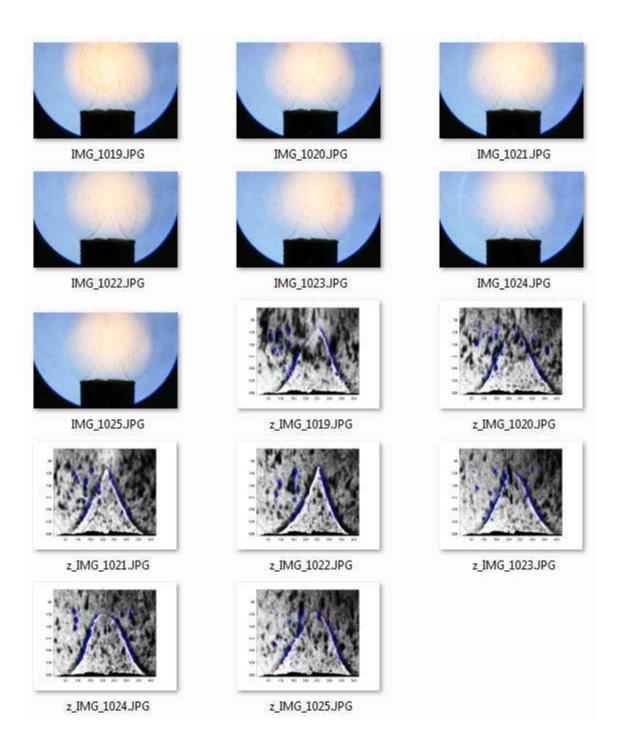


Experiment NO. 10

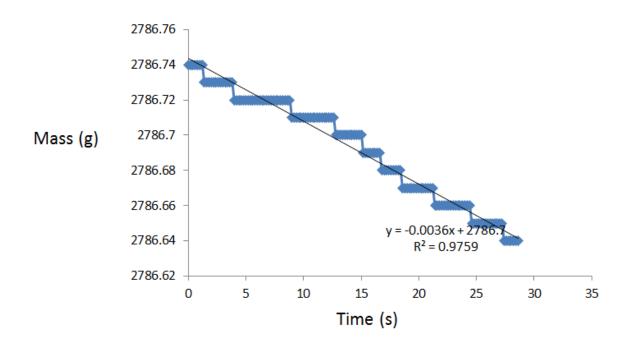


Air I	Flow Rate	Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.223	0.000155	4.54E- 05	0.010744	0.500255	0.849184

Entrainment Rate
3.60E-03 g/sec
Concentration
79.37594 g/m³

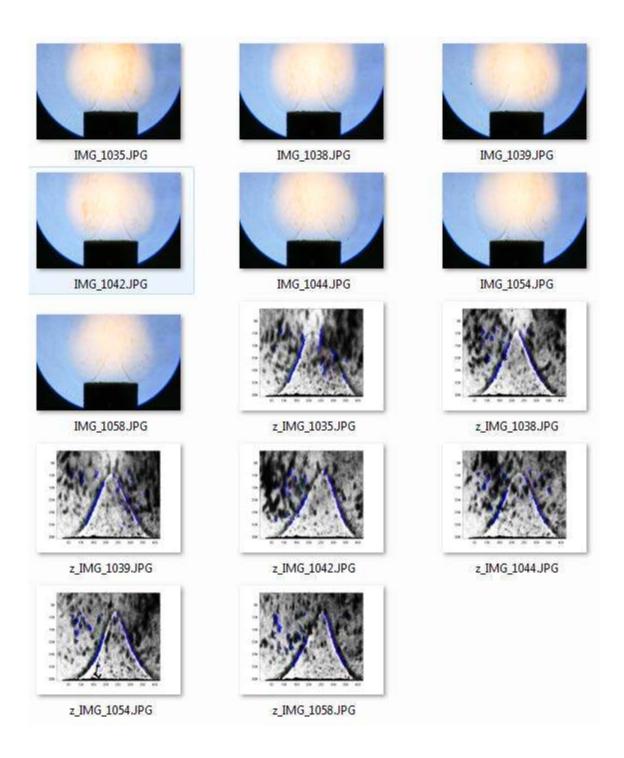


**Experiment NO. 11** 

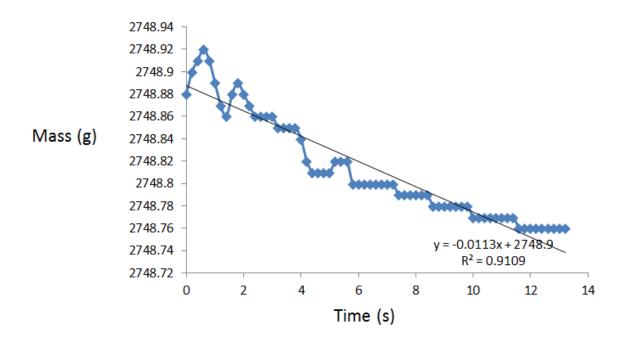


Air Flow Rate		Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.223	0.000155	4.54E- 05	0.010744	0.500255	0.849184

Entrainment Rate
3.80E-03 g/sec
Concentration
83.78572 g/m³

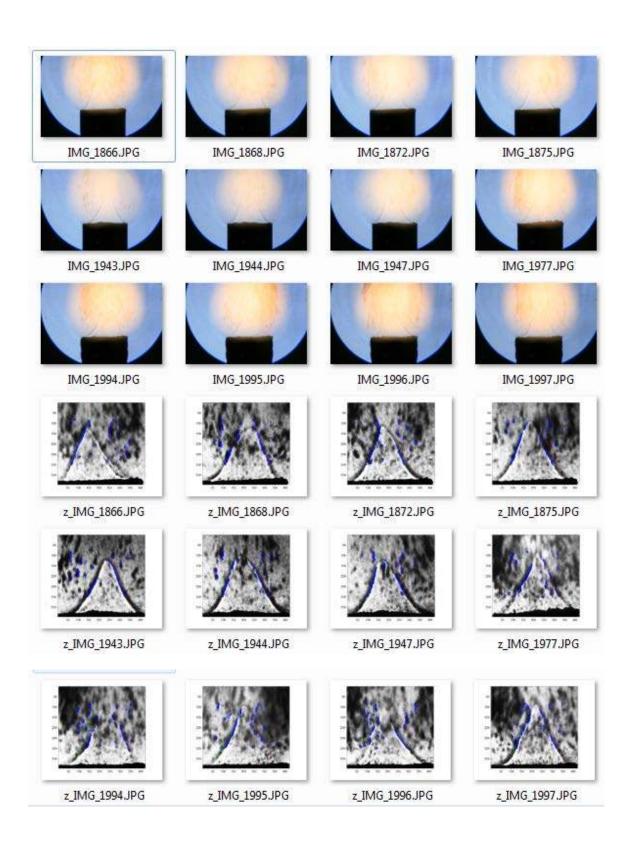


Experiment NO. 17



Air Flow Rate		Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2.5	0.001732	0.223	0.000155	4.54E- 05	0.010744	0.500255	0.849184

Entrainment Rate 1.13E-02 g/sec Concentration 249.1523 g/m<sup>3</sup>



## 5. Coal Particles (0-25 $\mu$ m)

### 5.1 $\phi$ =0.75; Flow Rate: Air 2.0 SLPM; Methane 0.158 SLPM

Experiment NO. 3

Air Flow Rate		Fuel Flow rate		Volume flow rate	nozzle width	U	Phi
SLPM	mole/sec	SLPM	mole/sec	m3/s	m	m/sec	
2	0.001386	0.158	0.000109	3.59E-05	0.010744	0.39646	0.75208

**Entrainment Rate** 

1.05E-02 g/sec

Concentration

292.1247 g/m<sup>3</sup>

